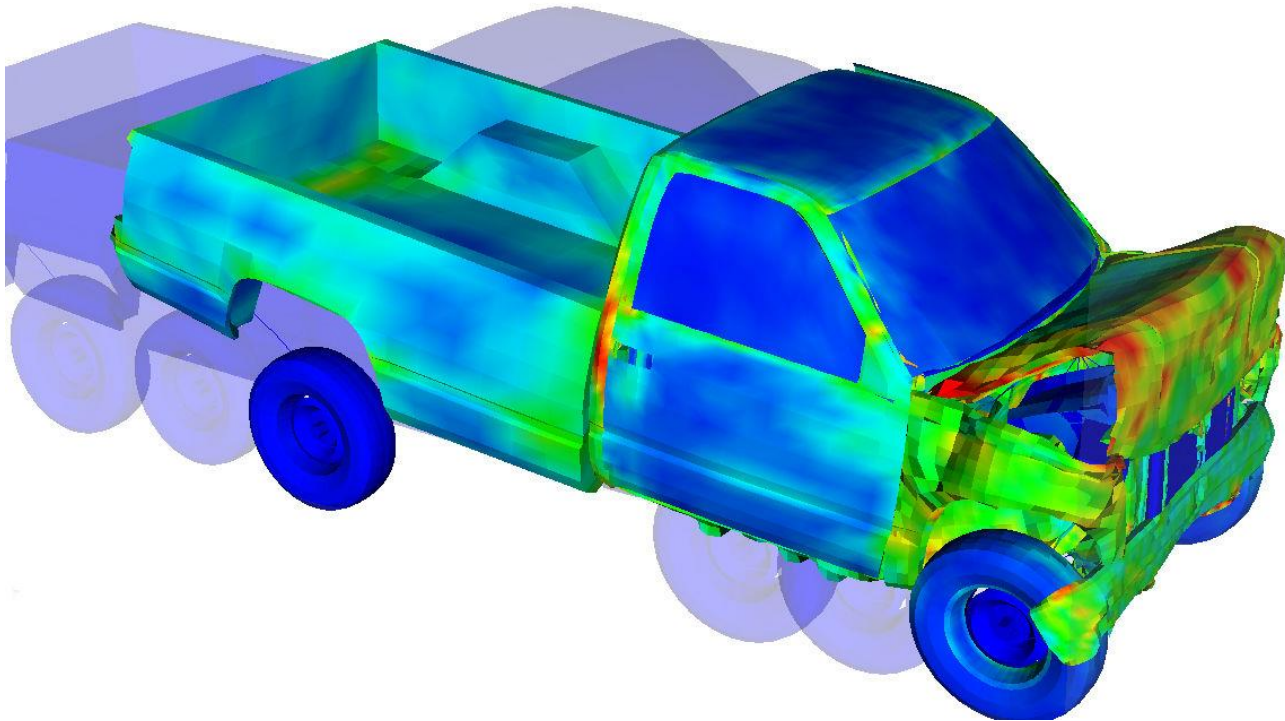
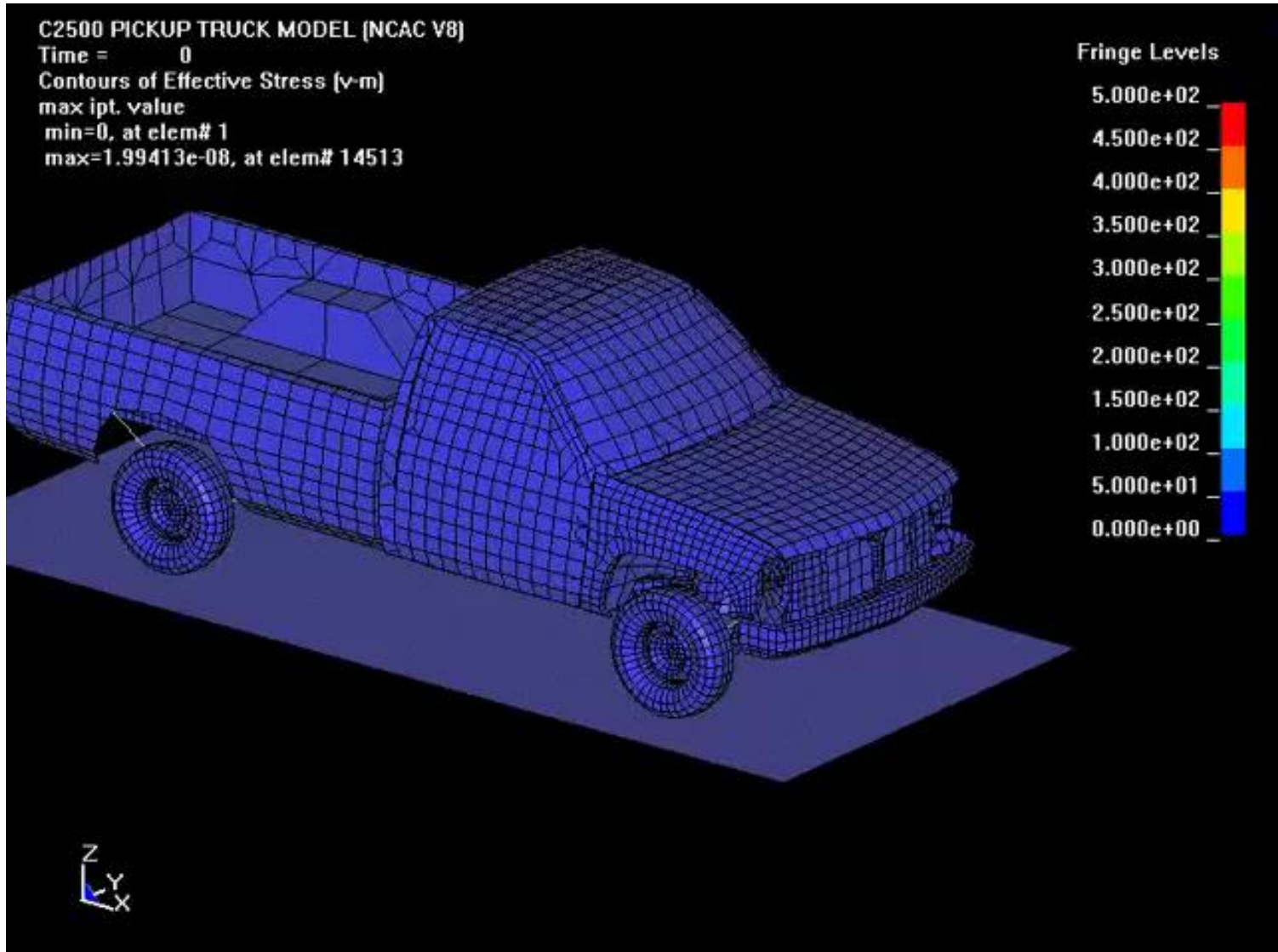

Continuum Mechanics and the Finite Element Method



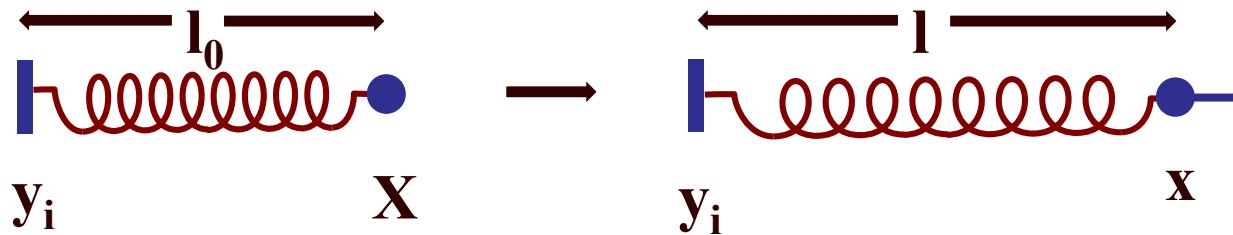
Assignment 2

- ◆ Due on March 2nd @ midnight

Suppose you want to simulate this...



The familiar mass-spring system



Spring length
before/after

$$\mathbf{l} = |\mathbf{x} - \mathbf{y}_i|$$

$$\mathbf{l}_0 = |\mathbf{X} - \mathbf{y}_i|$$

Deformation Measure

$$e = \left(\frac{l}{l_0} - 1 \right)$$

Elastic Energy

$$W = \frac{1}{2} k e^2$$

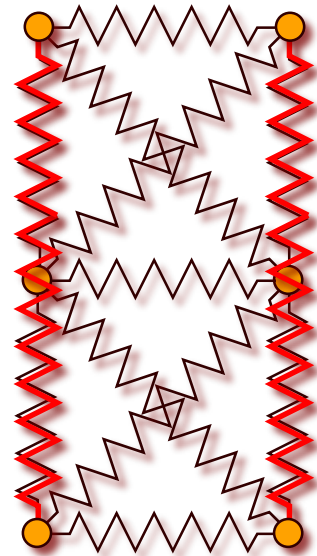
Forces

$$\mathbf{f}_{\text{int}} = - \frac{\partial W}{\partial \mathbf{x}}$$

$$\mathbf{f}_{\text{int}}(\mathbf{x}) = -k \left(\left(\frac{l}{l_0} - 1 \right) \frac{\mathbf{x} - \mathbf{y}_i}{|\mathbf{x} - \mathbf{y}_i|} \right)$$

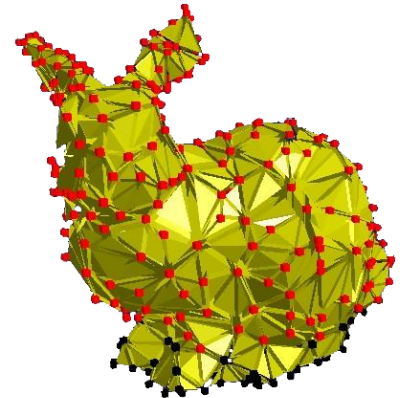
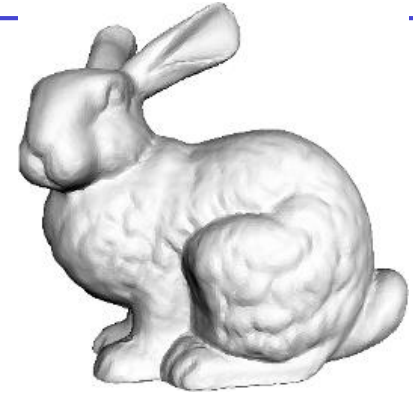
Mass Spring Systems

- ◆ Can be used to model arbitrary elastic/plastic objects, but...
 - Behavior depends on tessellation
 - Find good spring layout
 - Find good spring constants
 - Different types of springs interfere
 - No direct map to measurable material properties



Alternative...

- Start from continuum mechanics
- Discretize with Finite Elements
 - Decompose model into simple elements
 - Setup & solve system of algebraic equations
- Advantages
 - Accurate and controllable material behavior
 - Largely independent of mesh structure



Mass Spring vs Continuum Mechanics

◆ Mass spring systems require:

1. Measure of Deformation $\left(\frac{l}{l_0} - 1\right)$

2. Material Model k

3. Deformation Energy $W = \frac{1}{2}ke^2$

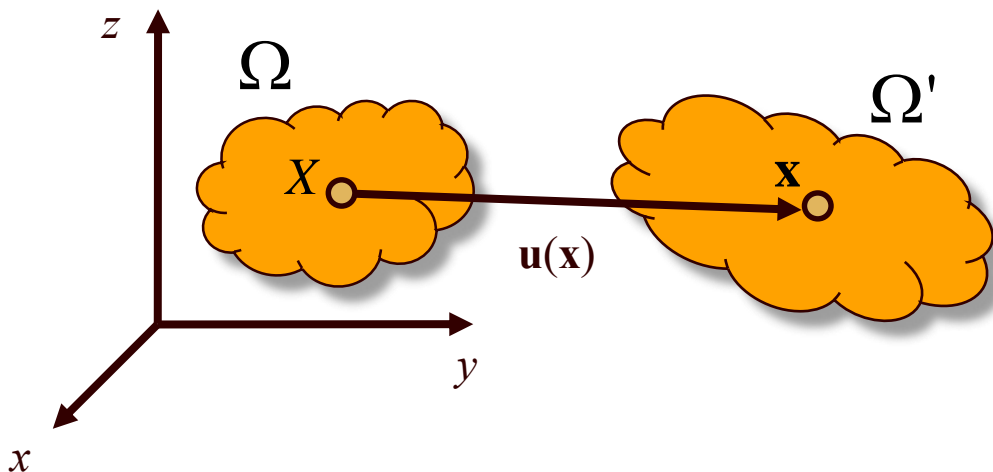
4. Internal Forces $f_{\text{int}} = -\frac{\partial W}{\partial x}$

◆ We need to derive the same types of concepts using continuum mechanics principles

Continuum Mechanics: 3D Deformations

- For a deformable body, identify:
 - undeformed state $\Omega \subset \mathbf{R}^3$ described by positions X
 - deformed state $\Omega' \subset \mathbf{R}^3$ described by positions \mathbf{x}
- Displacement field \mathbf{u} describes Ω' in terms of Ω

$$\mathbf{u} : \Omega \rightarrow \Omega' \quad \mathbf{x} = X + \mathbf{u}(X)$$



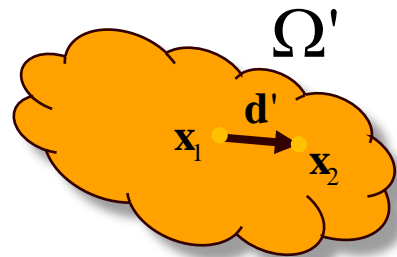
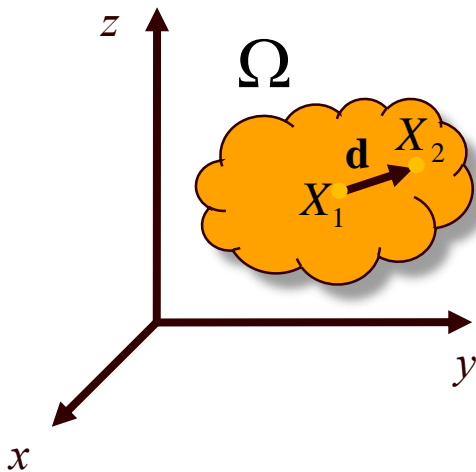
$$\mathbf{u}(X) = \begin{pmatrix} u(x, y, z) \\ v(x, y, z) \\ w(x, y, z) \end{pmatrix}$$

Continuum Mechanics: 3D Deformations

- Consider material points X_1 and X_2 such that $|\mathbf{d}|$ is infinitesimal, where $\mathbf{d} = X_2 - X_1$
- Now consider deformed vector \mathbf{d}'

$$\text{Deformation gradient } \mathbf{F} = \frac{\partial \mathbf{x}}{\partial \mathbf{X}}$$

$$\mathbf{d}' = \mathbf{x}_2 - \mathbf{x}_1 \approx \overbrace{(\mathbf{I} + \nabla \mathbf{u})} \mathbf{d}$$



$$\nabla \mathbf{u} = \begin{pmatrix} \partial_x u & \partial_y u & \partial_z u \\ \partial_x v & \partial_y v & \partial_z v \\ \partial_x w & \partial_y w & \partial_z w \end{pmatrix}$$

So...

- Displacement field transforms points

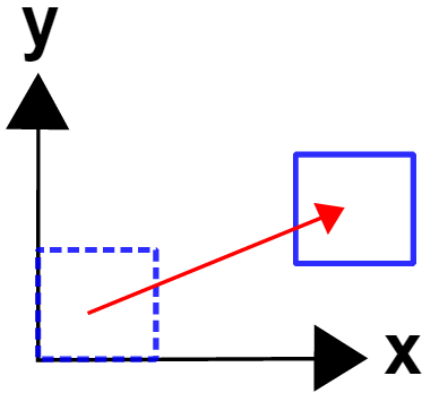
$$\mathbf{x} = \mathbf{X} + \mathbf{u}(\mathbf{X})$$

- Jacobian of displacement field (deformation gradient) transforms differentials (infinitesimal vectors) from undeformed to deformed

$$\mathbf{d}' = (\mathbf{I} + \nabla \mathbf{u}) \mathbf{d} = \mathbf{F} \mathbf{d} \quad \mathbf{F} = \frac{\partial \mathbf{x}}{\partial \mathbf{X}}$$

Displacement Field and Deformation Gradient

- ◆ In general, displacement field is not explicitly described. Nevertheless, toy examples:



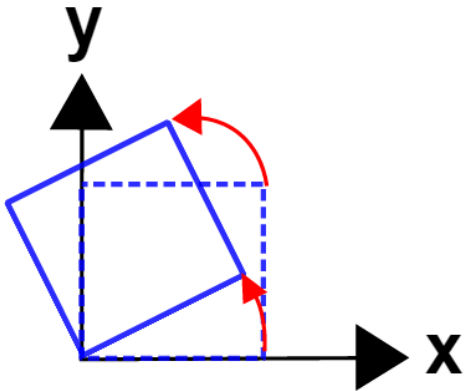
$$x = X + 5$$

$$y = Y + 2$$

$$\mathbf{F} = \mathbf{I}$$

Displacement Field and Deformation Gradient

- ◆ In general, displacement field is not explicitly described. Nevertheless, toy examples:



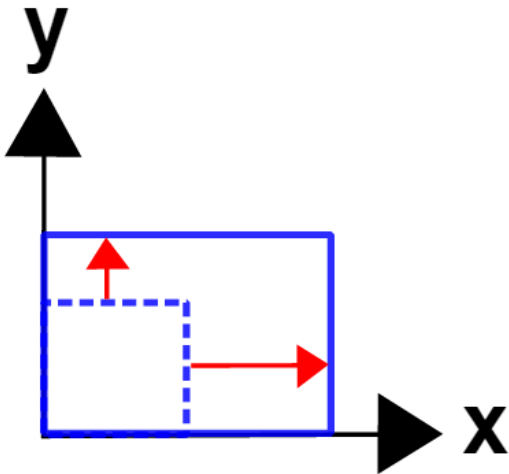
$$x = X \cos \theta - Y \sin \theta$$

$$y = X \sin \theta + Y \cos \theta$$

$$\mathbf{F} = \begin{bmatrix} \cos \theta & -\sin \theta \\ \sin \theta & \cos \theta \end{bmatrix}$$

Displacement Field and Deformation Gradient

- ◆ In general, displacement field is not explicitly described. Nevertheless, toy examples:



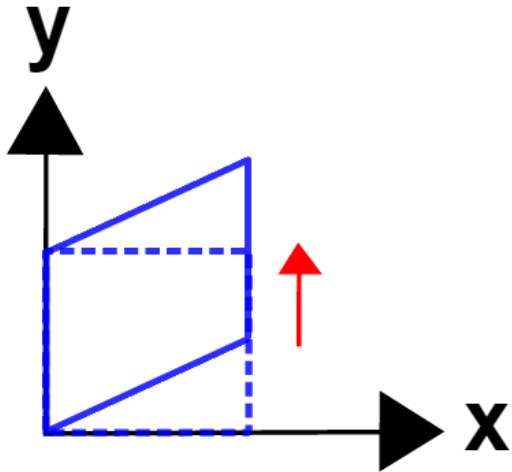
$$x = 2.0X + 0.0Y$$

$$y = 0.0X + 1.5Y$$

$$\mathbf{F} = \begin{bmatrix} 2.0 & 0.0 \\ 0.0 & 1.5 \end{bmatrix}$$

Displacement Field and Deformation Gradient

- ◆ In general, displacement field is not explicitly described. Nevertheless, toy examples:



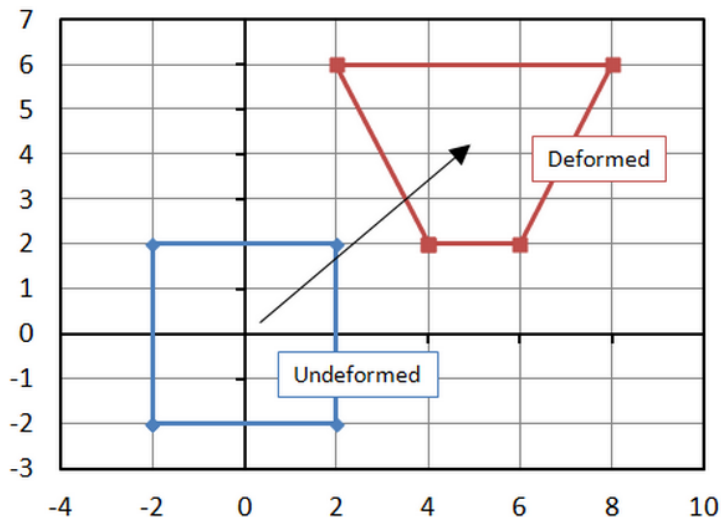
$$x = 1.0X + 0.0Y$$

$$y = 0.5X + 1.0Y$$

$$\mathbf{F} = \begin{bmatrix} 1.0 & 0.0 \\ 0.5 & 1.0 \end{bmatrix}$$

Displacement Field and Deformation Gradient

- ◆ In general, displacement field is not explicitly described. Nevertheless, toy examples:



$$x = X + \frac{1}{4}XY + 5$$

$$y = Y + 4$$

$$\mathbf{F} = \begin{bmatrix} 1 + \frac{1}{4}Y & \frac{1}{4}X \\ 0 & 1 \end{bmatrix}$$

Measure of deformations

- Displacement field transforms points

$$\mathbf{x} = \mathbf{X} + \mathbf{u}(\mathbf{X})$$

- Jacobian of displacement field (deformation gradient) transforms vectors

$$\mathbf{d}' = (\mathbf{I} + \nabla \mathbf{u}) \mathbf{d} = \mathbf{F} \mathbf{d} \quad \mathbf{F} = \frac{\partial \mathbf{x}}{\partial \mathbf{X}}$$

How can we describe deformations?

Back to spring deformation

◆ Deformation measure (strain): $\left(\frac{l}{l_0} - 1 \right)$

Similar to F

◆ Undeformed spring: $\frac{l}{l_0} = 1$

◆ Undeformed (infinitesimal) continuum volume:

$$\mathbf{F} = \mathbf{I} ?$$

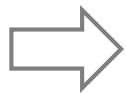
Strain (description of deformation in terms of *relative* displacement)

- ◆ Deformation measure (strain): $\left(\frac{l}{l_0} - 1 \right)$
- ◆ Desirable property: if spring is undeformed, strain is 0 (no change in shape)
- ◆ Can we find a similar measure that would work for infinitesimal volumes?

3D Nonlinear Strain

Idea: to quantify change in shape, measure change in squared length for any arbitrary vector

$$|\mathbf{d}'|^2 - |\mathbf{d}|^2 = \mathbf{d}'^T \mathbf{d}' - \mathbf{d}^T \mathbf{d} = \mathbf{d}^T (\mathbf{F}^T \mathbf{F} - \mathbf{I}) \mathbf{d}$$



Green strain $\mathbf{E} = \frac{1}{2}(\mathbf{F}^T \mathbf{F} - \mathbf{I})$

Strain (description of deformation in terms of *relative* displacement)

- ◆ Deformation measure (strain): $\left(\frac{l}{l_0} - 1 \right)$
- ◆ Desirable property: if spring is undeformed, strain is 0 (no change in shape)
- ◆ Can we find a similar measure that would work for infinitesimal volumes?

$$\text{Green strain } \mathbf{E} = \frac{1}{2}(\mathbf{F}^T \mathbf{F} - \mathbf{I})$$

3D Linear Strain

- Green strain is quadratic in displacements

$$\mathbf{E} = \frac{1}{2}(\mathbf{F}^T \mathbf{F} - \mathbf{I}) = \frac{1}{2}(\nabla \mathbf{u} + \nabla \mathbf{u}^T + \nabla \mathbf{u}^T \nabla \mathbf{u})$$

- Neglecting quadratic term (small deformation assumption) leads to the linear

**Cauchy strain
(small strain)**

$$\boldsymbol{\varepsilon} = \frac{1}{2}(\nabla \mathbf{u} + \nabla \mathbf{u}^t) = \frac{1}{2}(\mathbf{F} + \mathbf{F}^t) - \mathbf{I}$$

- Written out:

$$\boldsymbol{\varepsilon} = \frac{1}{2} \begin{pmatrix} 2\partial_x u & \partial_y u + \partial_x v & \partial_z u + \partial_x w \\ \partial_x v + \partial_y u & 2\partial_y v & \partial_z v + \partial_y w \\ \partial_x w + \partial_z u & \partial_y w + \partial_z v & 2\partial_z w \end{pmatrix}$$

Notation

$$\mathbf{u}(\mathbf{x}) = \begin{pmatrix} u(x, y, z) \\ v(x, y, z) \\ w(x, y, z) \end{pmatrix}$$

3D Linear Strain

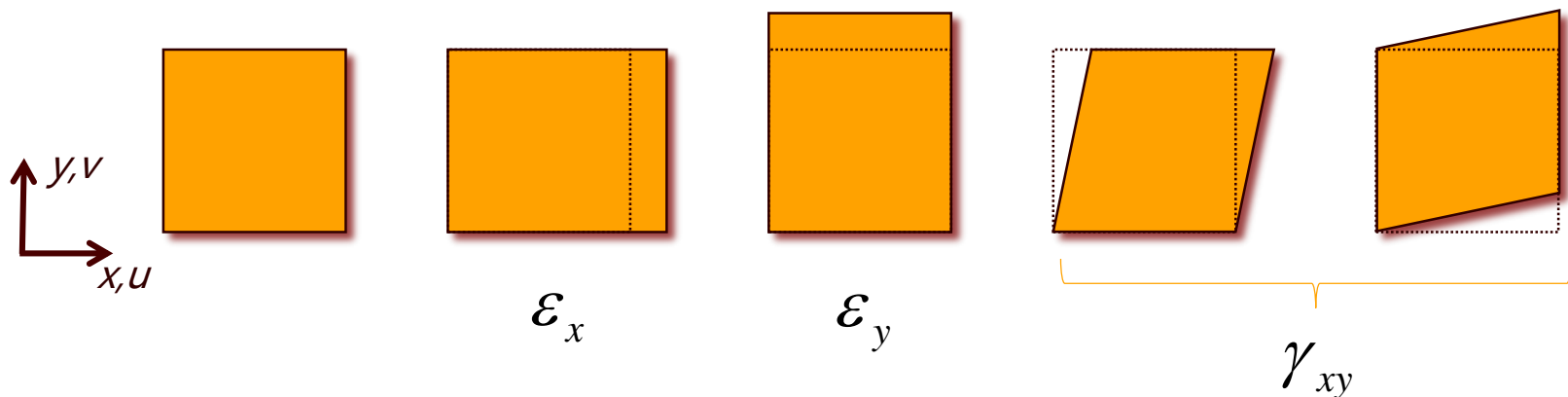
- Linear Cauchy strain

$$\boldsymbol{\varepsilon} = \frac{1}{2} \begin{pmatrix} 2\partial_x u & \partial_y u + \partial_x v & \partial_z u + \partial_x w \\ \partial_x v + \partial_y u & 2\partial_y v & \partial_z v + \partial_y w \\ \partial_x w + \partial_z u & \partial_y w + \partial_z v & 2\partial_z w \end{pmatrix} =: \begin{pmatrix} \varepsilon_x & \gamma_{xy} & \gamma_{xz} \\ \gamma_{xy} & \varepsilon_y & \gamma_{yz} \\ \gamma_{xz} & \gamma_{yz} & \varepsilon_z \end{pmatrix}$$

ε_i : normal strains

γ_i : shear strains

- Geometric interpretation



Cauchy vs. Green strain

- ◆ Nonlinear Green strain is rotation-invariant
 - Apply incremental rotation \mathbf{R} to given deformation \mathbf{F} to obtain $\mathbf{F}' = \mathbf{R}\mathbf{F}$
 - Then $\mathbf{E}' = \frac{1}{2}(\mathbf{F}'^T \mathbf{F}' - \mathbf{I}) = \mathbf{E}$
- ◆ Linear Cauchy strain is not rotation-invariant
$$\boldsymbol{\varepsilon}' = \frac{1}{2}(\mathbf{F}' + \mathbf{F}'^t) \neq \boldsymbol{\varepsilon}$$

→ artifacts for larger rotations



Mass Spring vs Continuum Mechanics

◆ Mass spring systems:

1. Measure of Deformation $\left(\frac{l}{l_0} - 1\right)$
2. Material Model $\cdot k$
3. Deformation Energy $W = \frac{1}{2}ke^2$
4. Internal Forces $f_{\text{int}} = -\frac{\partial W}{\partial x}$

◆ Continuum Mechanics:

1. Measure of Deformation: Green or Cauchy strain
2. Material Model

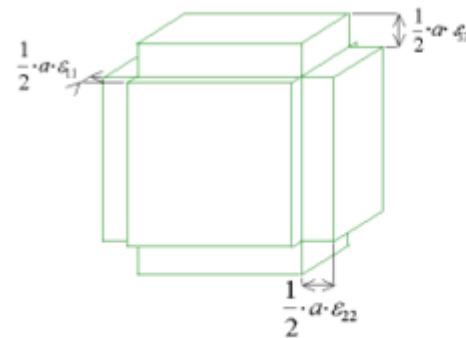
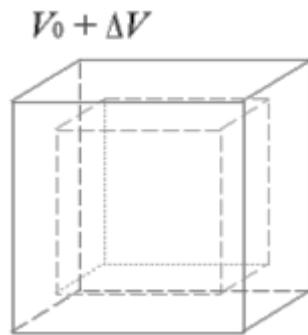
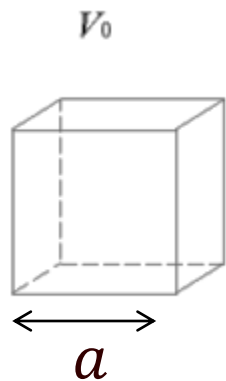
Material Model: linear isotropic material

- Material model links strain to energy (and stress)
- Linear isotropic material (*generalized Hooke's law*)
 - Energy density $\Psi = \frac{1}{2}\lambda\text{tr}(\boldsymbol{\varepsilon})^2 + \mu\text{tr}(\boldsymbol{\varepsilon}^2)$
 - Lamé parameters λ and μ are material constants related to Poisson Ratio and Young's modulus
- Interpretation
 - $\text{tr}(\boldsymbol{\varepsilon}^2) = \|\boldsymbol{\varepsilon}\|_F^2$ penalizes all strain components equally
 - $\text{tr}(\boldsymbol{\varepsilon})^2$ penalizes dilatations, i.e., volume changes

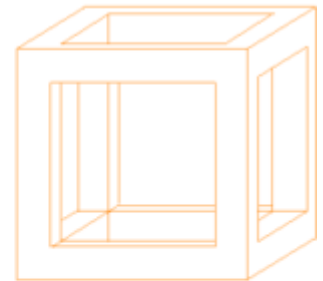
Volumetric Strain (*dilatation, hydrostatic strain*)

- ◆ Consider a cube with side length a
- ◆ For a given deformation $\boldsymbol{\varepsilon}$, the volumetric strain is

$$\begin{aligned} \Delta V/V_0 &= (a(1 + \varepsilon_{11}) \cdot a(1 + \varepsilon_{22}) \cdot a(1 + \varepsilon_{33}) - a^3)/a^3 \\ &= (\varepsilon_{11} + \varepsilon_{22} + \varepsilon_{33}) + O(\boldsymbol{\varepsilon}^2) \approx \text{tr}(\boldsymbol{\varepsilon}) \end{aligned}$$



$$V_0 + (\varepsilon_{11} + \varepsilon_{22} + \varepsilon_{33}) \cdot V_0$$

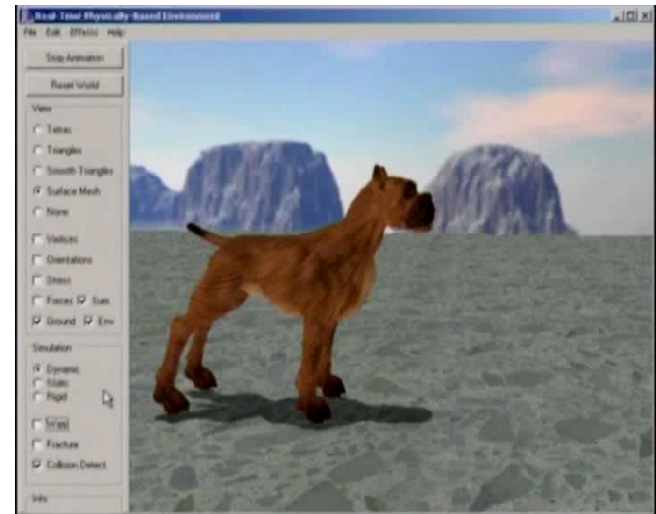


$$\left(\frac{1}{2} \cdot \sum_{i \neq j} \varepsilon_{ii} \cdot \varepsilon_{jj} + \varepsilon_{11} \cdot \varepsilon_{22} \cdot \varepsilon_{33} \right) \cdot V_0$$

Linear isotropic material

$$\text{Energy density: } \Psi = \frac{1}{2} \lambda \text{tr}(\boldsymbol{\varepsilon})^2 + \mu \text{tr}(\boldsymbol{\varepsilon}^2)$$

- ◆ Problem: Cauchy strain is not invariant under rotations
→ artifacts for rotations
- ◆ Solutions:
 - Corotational elasticity
 - Nonlinear elasticity



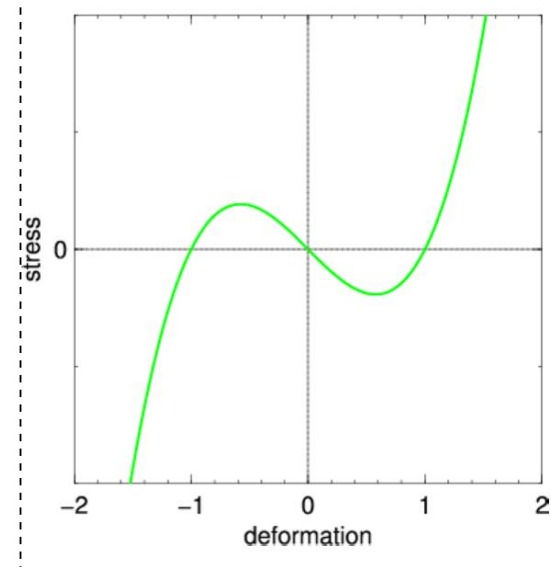
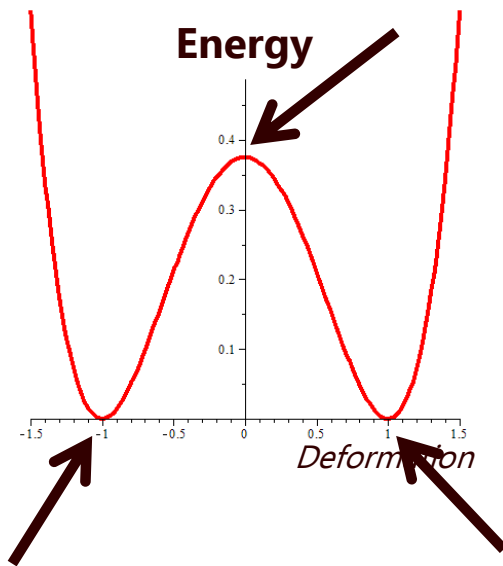
Material Model: non-linear isotropic model

- ◆ Replace Cauchy strain with Green strain \rightarrow *St. Venant-Kirchhoff material* (StVK)
- ◆ Energy density: $\Psi_{StVK} = \frac{1}{2} \lambda \text{tr}(\mathbf{E})^2 + \mu \text{tr}(\mathbf{E}^2)$
- ◆ Rotation invariant!

Problems with StVK

- ◆ StVK softens under compression

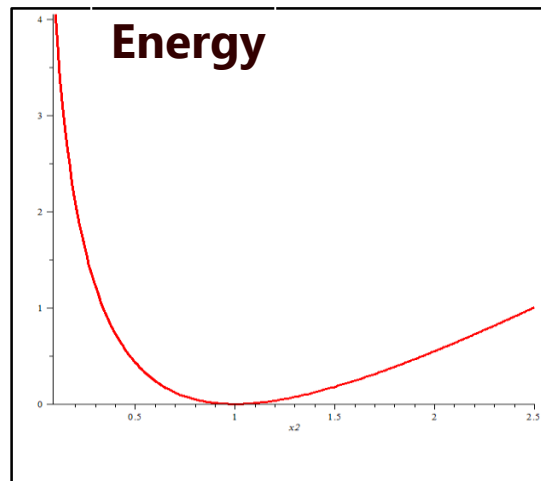
$$\Psi_{StVK} = \frac{1}{2} \lambda \text{tr}(\mathbf{E})^2 + \mu \text{tr}(\mathbf{E}^2)$$



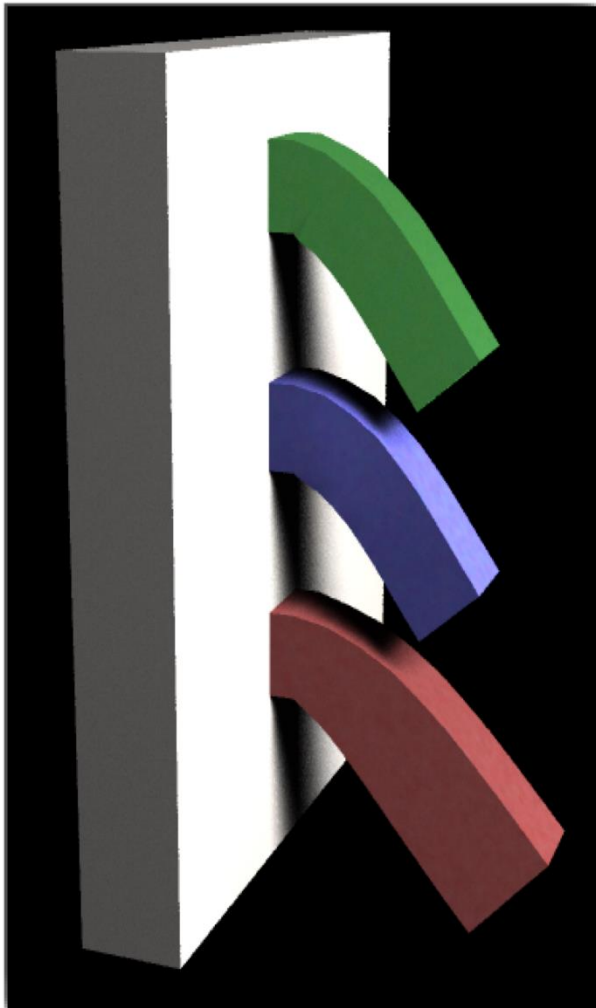
Advanced nonlinear materials

- ◆ Green Strain $\mathbf{E} = \frac{1}{2}(\mathbf{F}^t\mathbf{F} - \mathbf{I}) = \frac{1}{2}(\mathbf{C} - \mathbf{I})$
- ◆ Split into *deviatoric* (i.e. shape changing/distortion) and *volumetric* (dilation, volume changing) deformations
 - Volumetric: $J = \det(\mathbf{F})$ Deviatoric: $\hat{\mathbf{C}} = \det(\mathbf{F})^{-2/3} \mathbf{C}$
- ◆ Neo-Hookean material:

$$\Psi_{NH} = \frac{\mu}{2} \text{tr}(\hat{\mathbf{C}} - \mathbf{I}) - \mu \ln(J) + \frac{\lambda}{2} \ln(J)^2$$



Different Models



St. Venant-Kirchhoff

Neo-Hookean

Linear

Mass Spring vs Continuum Mechanics

◆ Mass spring systems:

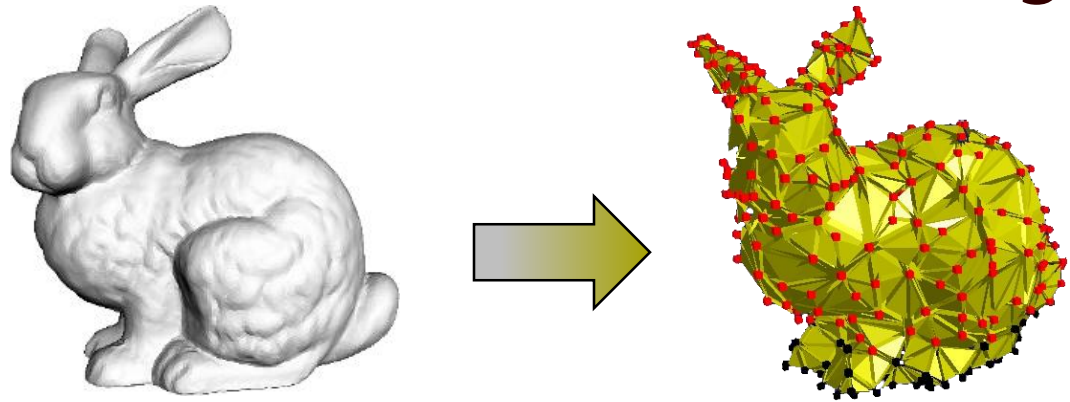
1. Measure of Deformation $\left(\frac{l}{l_0} - 1\right)$
2. Material Model $\cdot k$
3. Deformation Energy $W = \frac{1}{2}ke^2$
4. Internal Forces $f_{\text{int}} = -\frac{\partial W}{\partial x}$

◆ Continuum Mechanics:

1. Measure of Deformation: Green or Cauchy strain
2. Material Model: linear, StVK, Neo-Hookean, etc
3. From Energy Density to Deformation Energy:
Finite Element Discretization

Finite Element Discretization

- ◆ Divide domain into discrete elements, e.g., *tetrahedra*



- Explicitly store displacement values at nodes (\mathbf{x}_i).
- Displacement field everywhere else obtained through interpolation: $\mathbf{x}(X) = \sum N_i(X) \mathbf{x}_i$
- Deformation Gradient: $\mathbf{F} = \frac{\partial \mathbf{x}(X)}{\partial X} = \sum_i \mathbf{x}_i \left(\frac{\partial N_i}{\partial X} \right)^t$

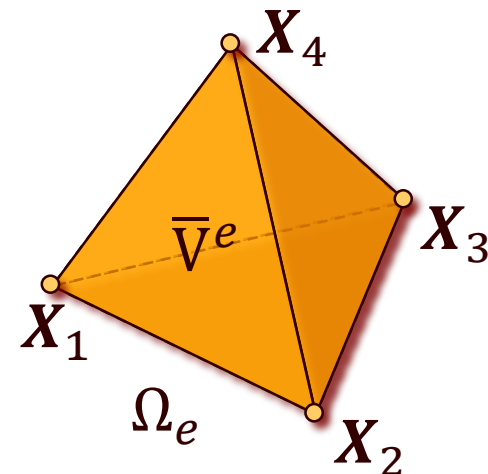
Basis Functions

- Basis functions (aka shape functions) $N_i(X_j): \mathbf{R}^3 \rightarrow \mathbf{R}$
- Satisfy delta-property: $N_i(X_j) = \delta_{ij}$
- Simplest choice: linear basis functions

$$N_i(\bar{x}, \bar{y}, \bar{z}) = a_i \bar{x} + b_i \bar{y} + c_i \bar{z} + d_i$$

- Compute N_i (and $\frac{\partial N_i}{\partial X}$) through

$$\begin{pmatrix} x_1 & y_1 & z_1 & 1 \\ x_2 & y_2 & z_2 & 1 \\ x_3 & y_3 & z_3 & 1 \\ x_4 & y_4 & z_4 & 1 \end{pmatrix} \begin{pmatrix} a_i \\ b_i \\ c_i \\ d_i \end{pmatrix} = \begin{pmatrix} \delta_{1i} \\ \delta_{2i} \\ \delta_{3i} \\ \delta_{4i} \end{pmatrix}$$



Constant Strain Elements (Linear Basis Functions)

- Displacement field is continuous in space
- Deformation Gradient, strain, stress are not
 - Constant Strain per element
- Deformation Gradient can be computed as
 - $\mathbf{F} = \mathbf{eE}^{-1}$ where \mathbf{e} and \mathbf{E} are matrices whose columns are edge vectors in undeformed and deformed configurations

Constant Strain Elements: From energy density to deformation energy

- Integrate energy density over the entire element:

$$W^e = \int_{\Omega_e} \Psi(\mathbf{F})$$

- If basis functions are linear:
 - \mathbf{F} is linear in \mathbf{x}_i
 - \mathbf{F} is constant throughout element:

$$W^e = \int_{\Omega_e} \Psi(\mathbf{F}) = \Psi(\mathbf{F}) \cdot \bar{V}^e$$

Mass Spring vs Finite Element Method

◆ Mass spring systems:

1. Measure of Deformation $\left(\frac{l}{l_0} - 1\right)$
2. Material Model k
3. Deformation Energy $W = \frac{1}{2}ke^2$
4. Internal Forces $f_{\text{int}} = -\frac{\partial W}{\partial x}$

◆ Continuum Mechanics:

1. Measure of Deformation: Green or Cauchy strain
2. Material Model: linear, StVK, Neo-Hookean, etc
3. Deformation Energy: integrate over elements
4. Internal Forces: $f_{\text{int}} = -\frac{\partial W}{\partial x}$

FEM recipe

- ◆ Discretize into elements (triangles/tetrahedrons, etc)
- ◆ For each element
 - Compute deformation gradient $\mathbf{F} = \mathbf{e}\mathbf{E}^{-1}$
 - Use material model to define energy density $\Psi(\mathbf{F})$
 - Integrate over elements to compute energy: W
 - Compute nodal forces as: $f_{\text{int}} = -\frac{\partial W}{\partial x}$

FEM recipe

St. Venant-Kirchhoff material

$$\mathbf{E} = \frac{1}{2}(\mathbf{F}^T \mathbf{F} - \mathbf{I})$$

$$\Psi = \mu \|\mathbf{E}\|_F + \frac{\lambda}{2} \text{tr}^2(\mathbf{E})$$

Neohookean elasticity

$$I_1 = \|\mathbf{F}\|_F^2, \quad J = \det \mathbf{F}$$

$$\Psi = \frac{\mu}{2}(I_1 - 3) - \mu \log(J) + \frac{\lambda}{2} \log^2(J)$$

Area/volume of element

$$\mathbf{f} = -\frac{\partial W}{\partial \mathbf{x}} = -V \underbrace{\frac{\partial \Psi}{\partial \mathbf{F}}}_{\mathbf{P}} \frac{\partial \mathbf{F}}{\partial \mathbf{x}}$$

First Piola-Kirchhoff stress tensor \mathbf{P}

FEM recipe

St. Venant-Kirchhoff material

$$\mathbf{E} = \frac{1}{2}(\mathbf{F}^T \mathbf{F} - \mathbf{I})$$

$$\Psi = \mu \|\mathbf{E}\|_F + \frac{\lambda}{2} \text{tr}^2(\mathbf{E})$$

$$\mathbf{P} = \mathbf{F} [2\mu \mathbf{E} + \lambda \text{tr}(\mathbf{E}) \mathbf{I}]$$

Neohookean elasticity

$$I_1 = \|\mathbf{F}\|_F^2, \quad J = \det \mathbf{F}$$

$$\Psi = \frac{\mu}{2} (I_1 - 3) - \mu \log(J) + \frac{\lambda}{2} \log^2(J)$$

$$\mathbf{P} = \mu(\mathbf{F} - \mathbf{F}^{-T}) + \lambda \log(J) \mathbf{F}^{-T}$$

Area/volume of element

$$\mathbf{f} = -\frac{\partial W}{\partial \mathbf{x}} = -V \underbrace{\frac{\partial \Psi}{\partial \mathbf{F}}}_{\text{Area/volume of element}} \frac{\partial \mathbf{F}}{\partial \mathbf{x}}$$

First Piola-Kirchhoff stress tensor \mathbf{P}

FEM recipe

St. Venant-Kirchhoff material

$$\mathbf{E} = \frac{1}{2}(\mathbf{F}^T \mathbf{F} - \mathbf{I})$$

$$\Psi = \mu \|\mathbf{E}\|_F + \frac{\lambda}{2} \text{tr}^2(\mathbf{E})$$

$$\mathbf{P} = \mathbf{F} [2\mu \mathbf{E} + \lambda \text{tr}(\mathbf{E}) \mathbf{I}]$$

Neohookean elasticity

$$I_1 = \|\mathbf{F}\|_F^2, \quad J = \det \mathbf{F}$$

$$\Psi = \frac{\mu}{2} (I_1 - 3) - \mu \log(J) + \frac{\lambda}{2} \log^2(J)$$

$$\mathbf{P} = \mu(\mathbf{F} - \mathbf{F}^{-T}) + \lambda \log(J) \mathbf{F}^{-T}$$

For a tetrahedron, this works out to:

$$[\mathbf{f}_1 \ \mathbf{f}_2 \ \mathbf{f}_3] = -V \mathbf{P} \mathbf{E}^{-T}; \quad \mathbf{f}_4 = -\mathbf{f}_1 - \mathbf{f}_2 - \mathbf{f}_3$$

Additional reading: <http://www.femdefo.org/>

Material Parameters

St. Venant-Kirchhoff material

$$\mathbf{E} = \frac{1}{2}(\mathbf{F}^T \mathbf{F} - \mathbf{I})$$

$$\Psi = \mu \|\mathbf{E}\|_F + \frac{\lambda}{2} \text{tr}^2(\mathbf{E})$$

Neohookean elasticity

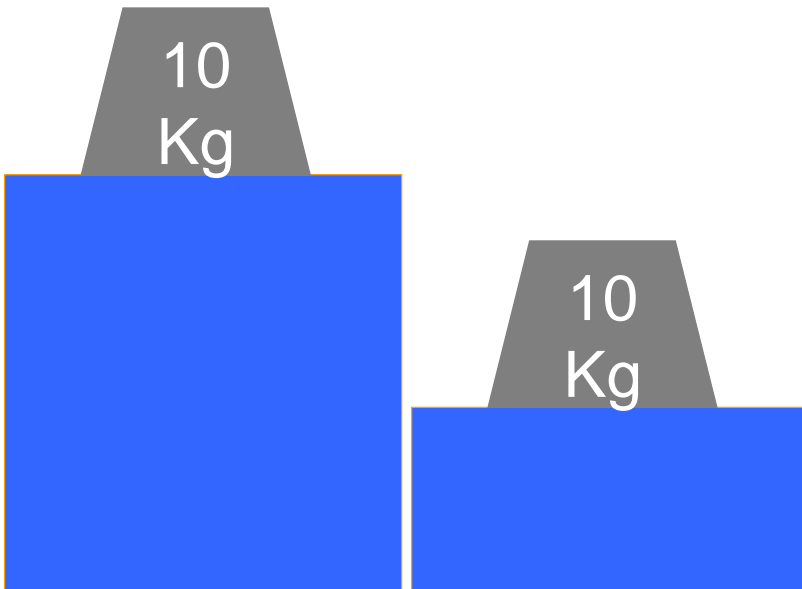
$$I_1 = \|\mathbf{F}\|_F^2, \quad J = \det \mathbf{F}$$

$$\Psi = \frac{\mu}{2}(I_1 - 3) - \mu \log(J) + \frac{\lambda}{2} \log^2(J)$$

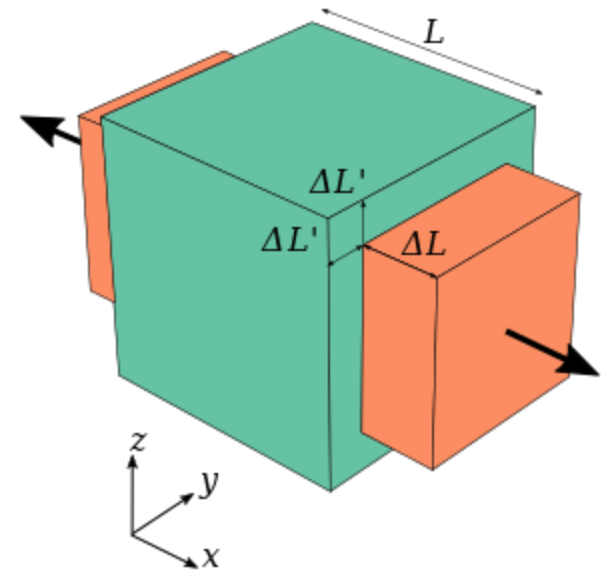
Lame parameters λ and μ are material constants related to the fundamental physical parameters: Poisson's Ratio and Young's modulus (http://en.wikipedia.org/wiki/Lamé_parameters)

Young's Modulus and Poisson Ratio

Lame parameters λ and μ are material constants related to the fundamental physical parameters: Poisson's Ratio and Young's modulus (http://en.wikipedia.org/wiki/Lamé_parameters)



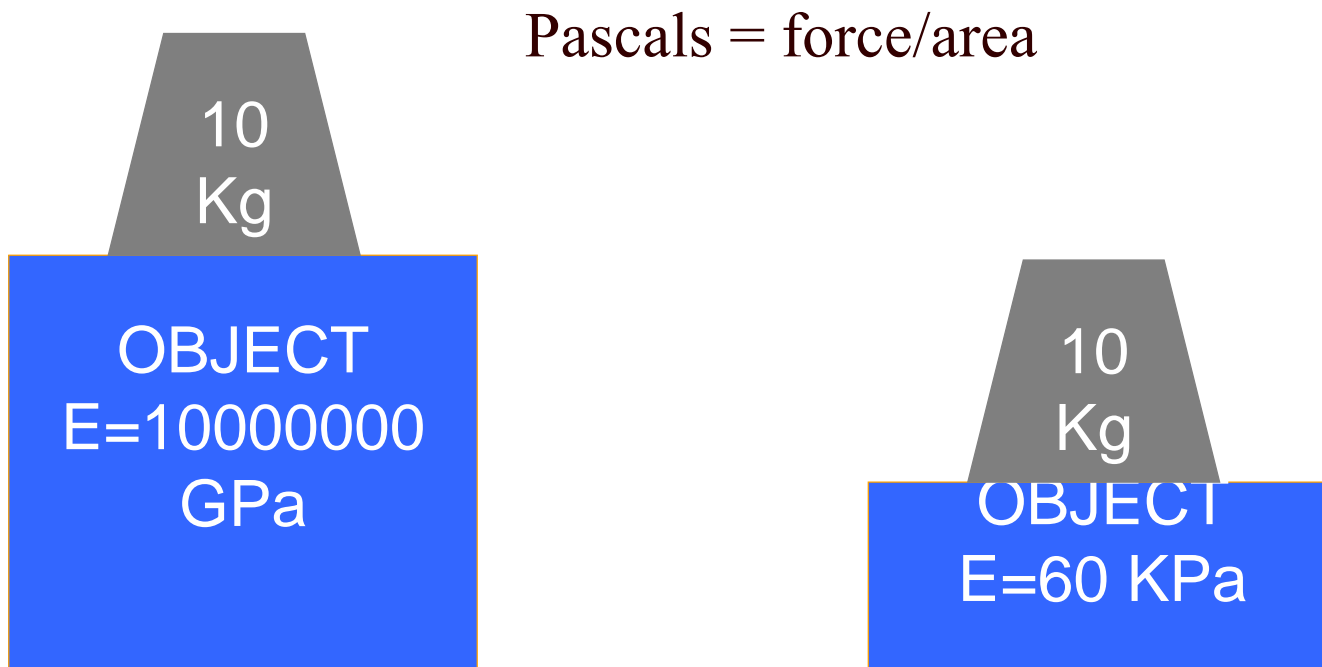
Young's modulus (E),
measure of stiffness



Poisson's ratio (ν), relative
transverse to axial deformation

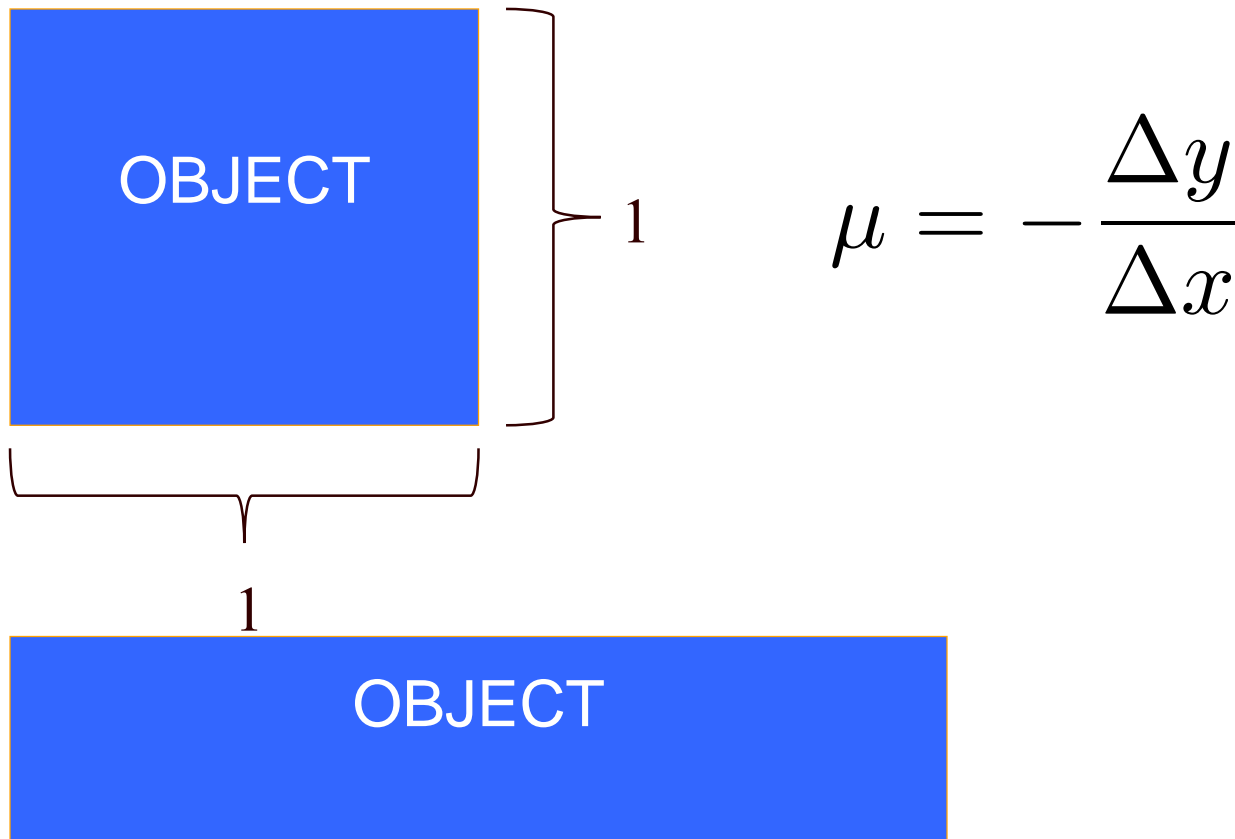
What Do These Parameters Mean

- ◆ Stiffness is pretty intuitive



What Do These Parameters Mean

Poisson's Ratio controls volume preservation

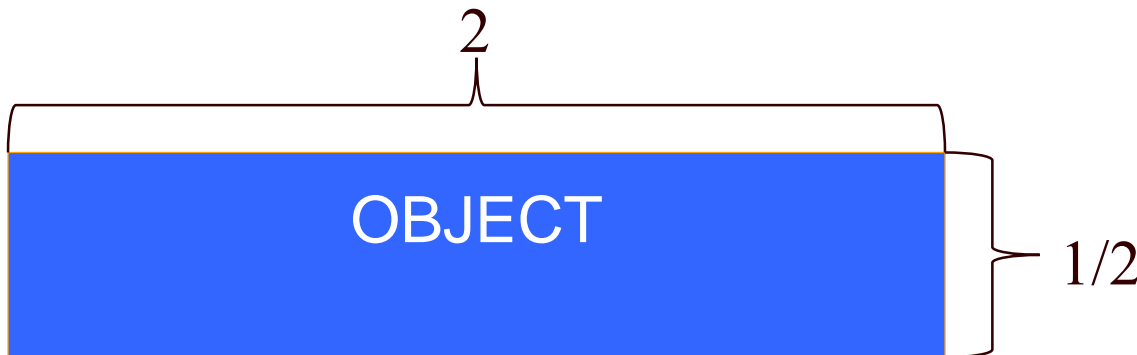


What Do These Parameters Mean

Poisson's Ratio controls volume preservation



$$\mu = -\frac{\Delta y}{\Delta x}$$



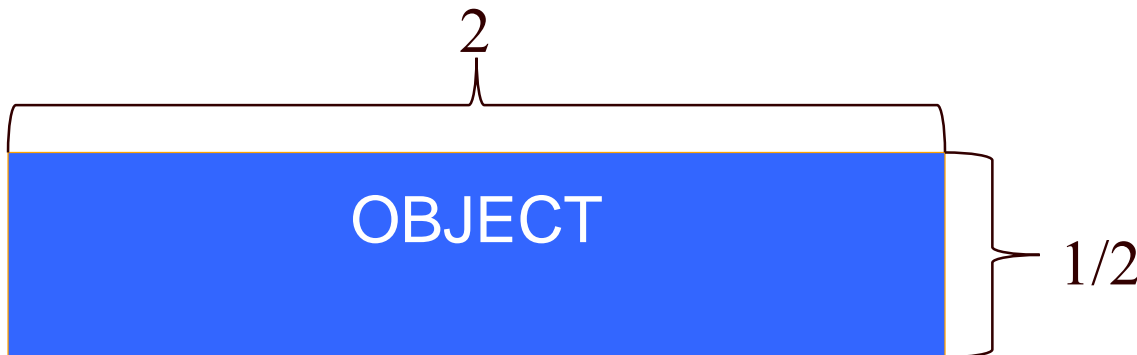
What Do These Parameters Mean

Poisson's Ratio controls volume preservation



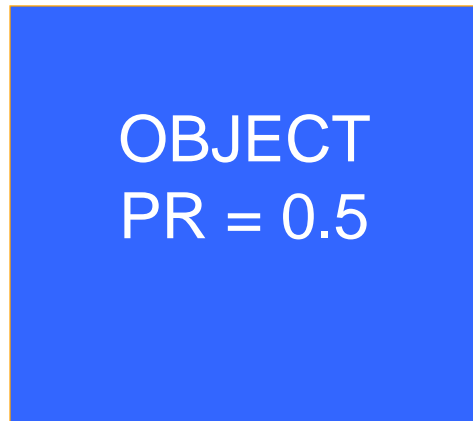
$$\mu = -\frac{\Delta y}{\Delta x}$$

Poisson's Ratio (ν) = ?

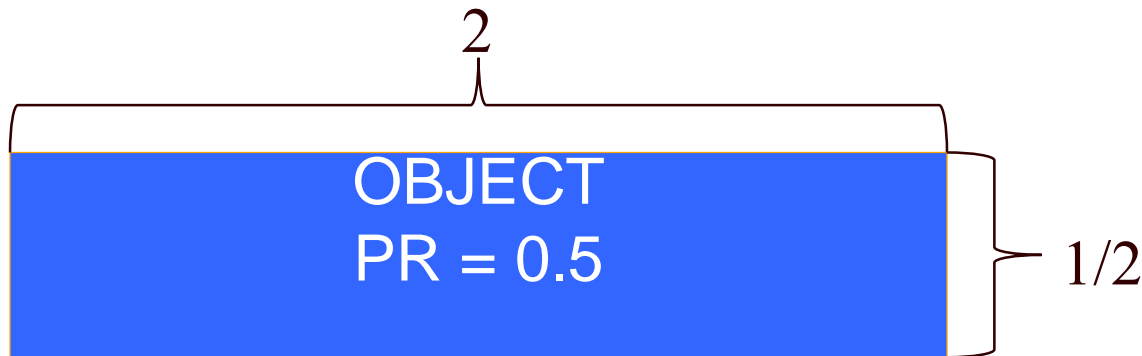


What Do These Parameters Mean

Poisson's Ratio controls volume preservation

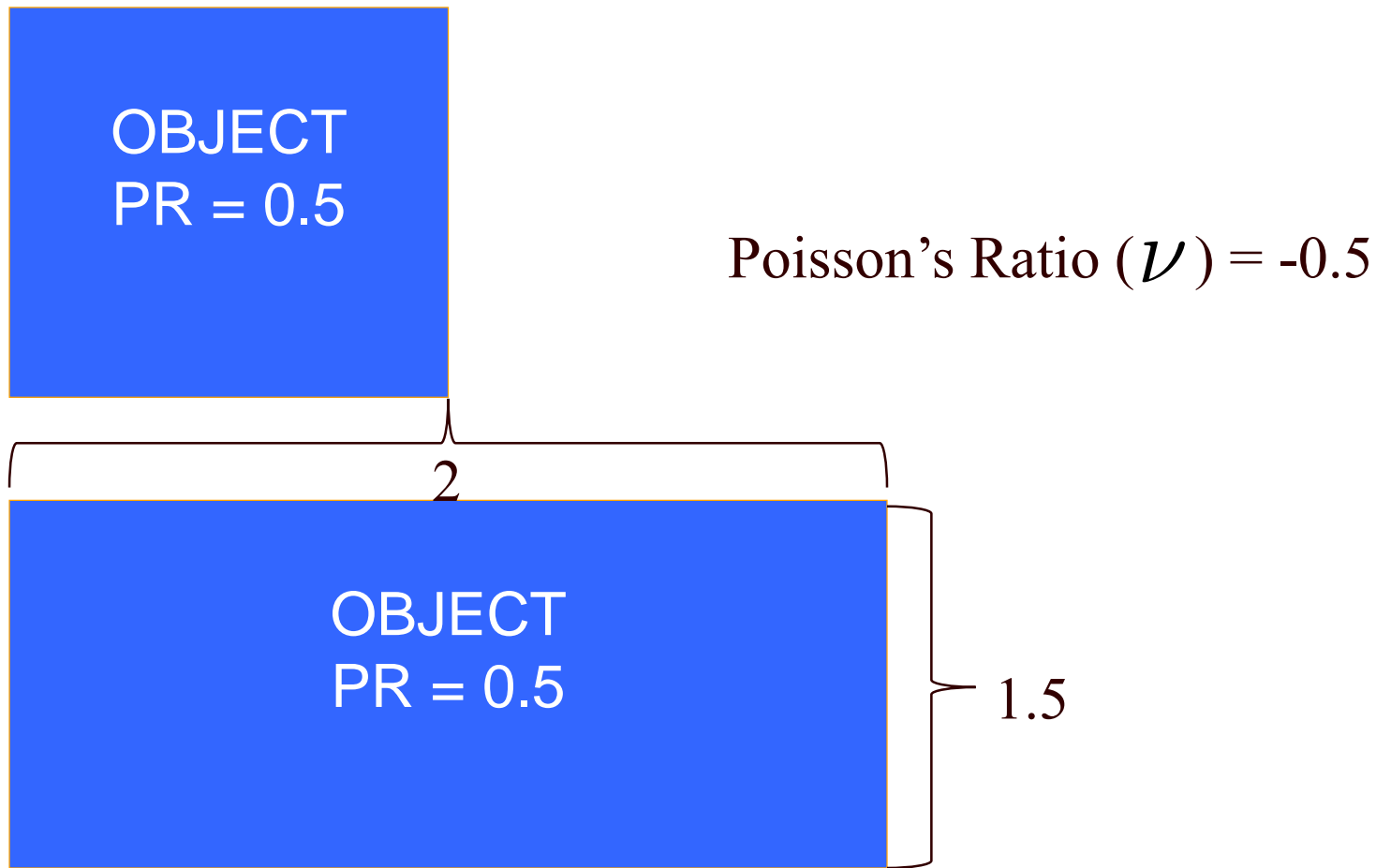


Poisson's Ratio (ν) = 0.5



What Do These Parameters Mean

Poisson's ratio is between -1 and 0.5



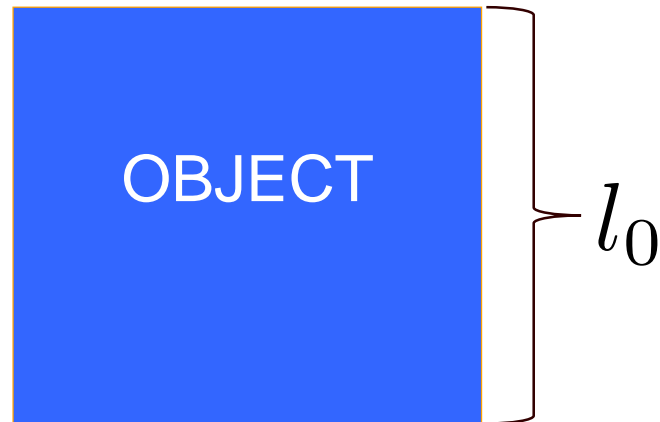
Negative Poisson's Ratio



Measurement

- ◆ Where do material parameters come from?

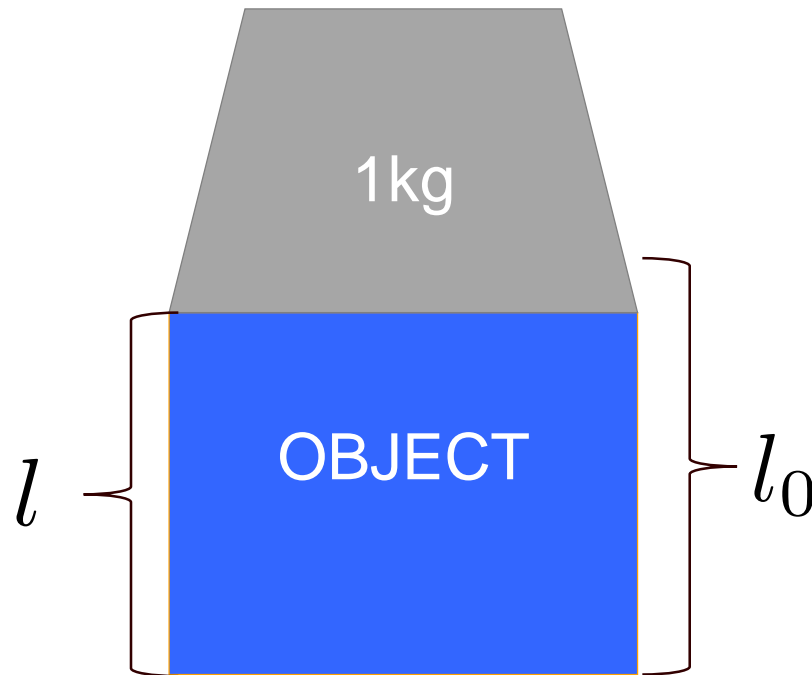
Simple Measurement: Stiffness



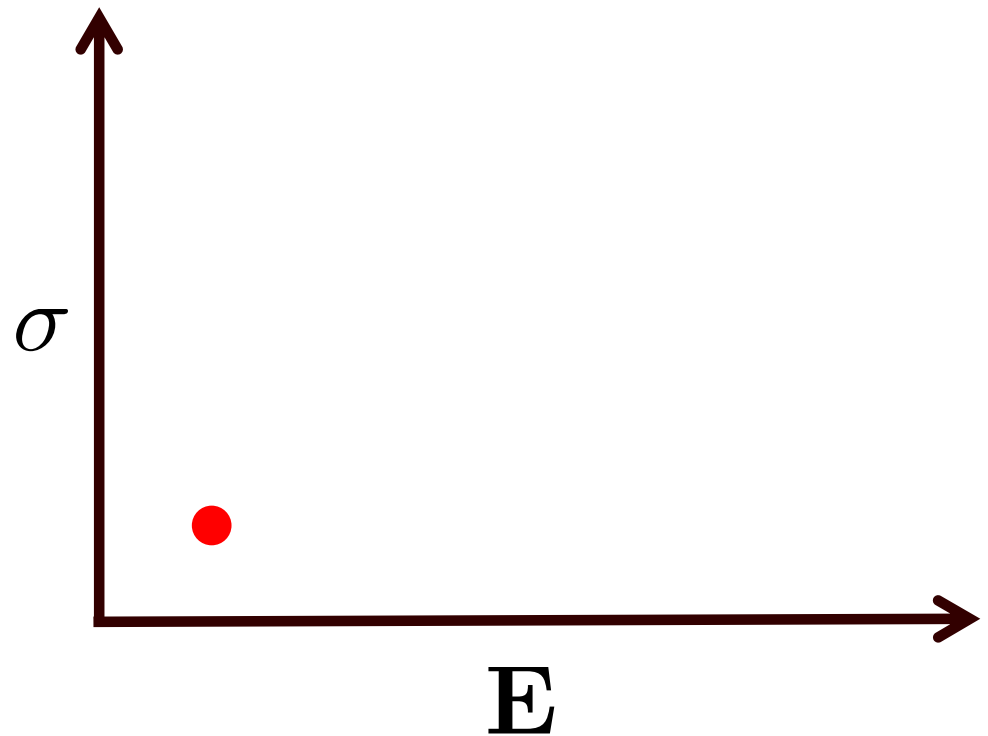
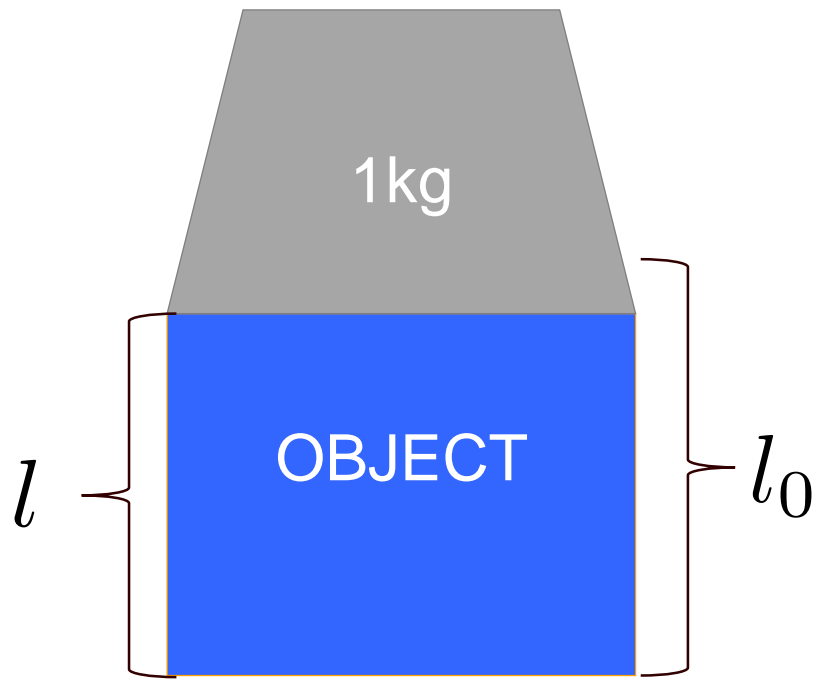
Simple Measurement

What's the Force (Stress)?

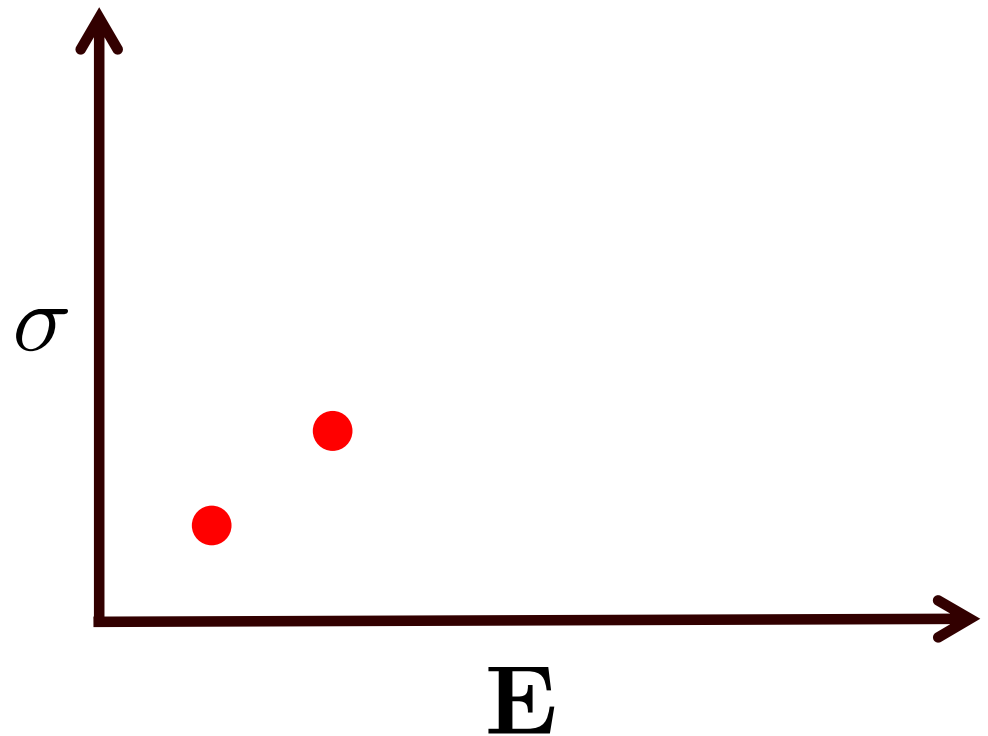
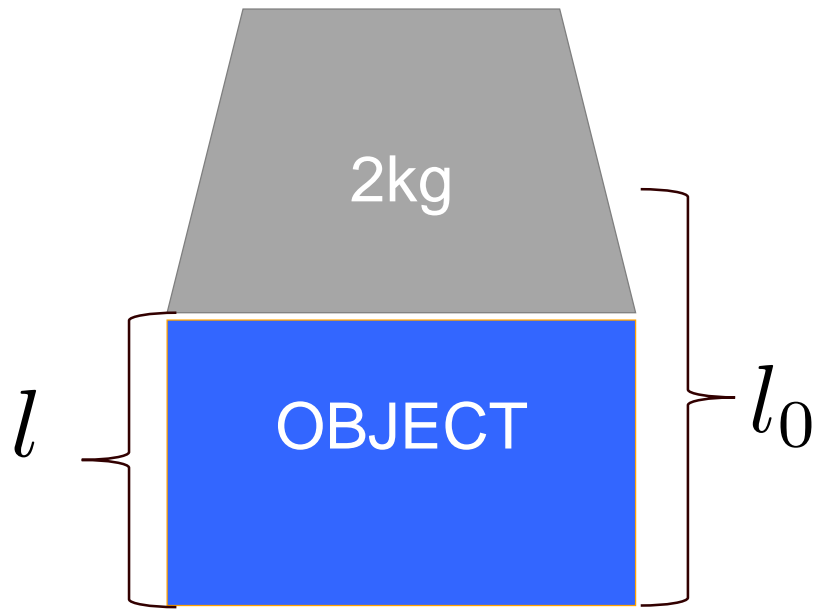
What's the Deformation (Strain)?



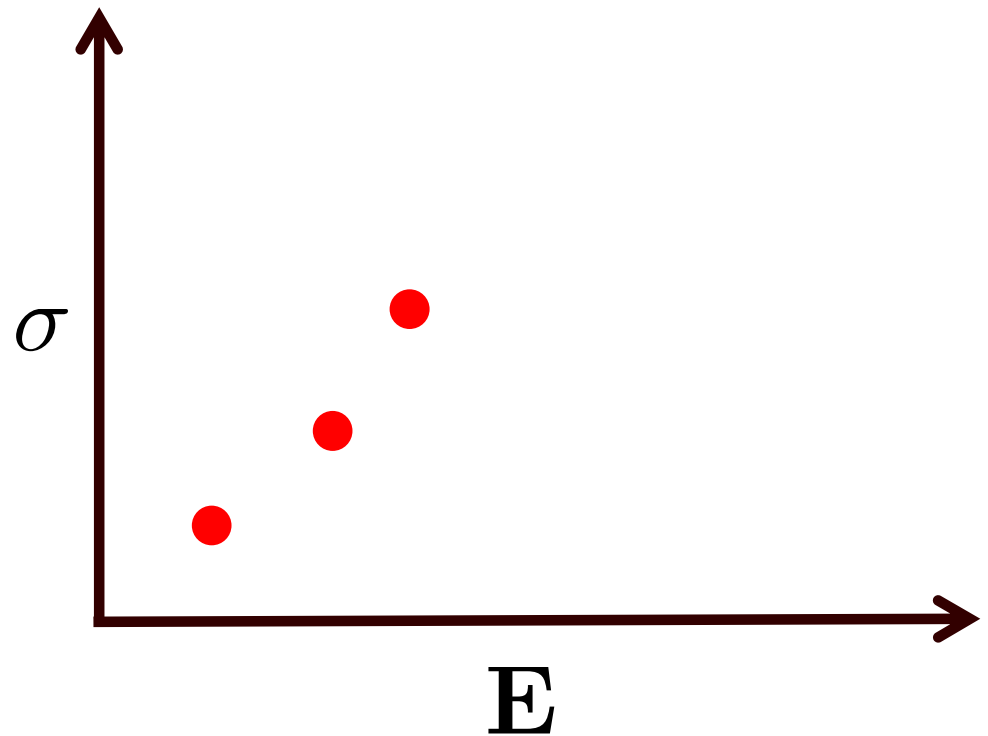
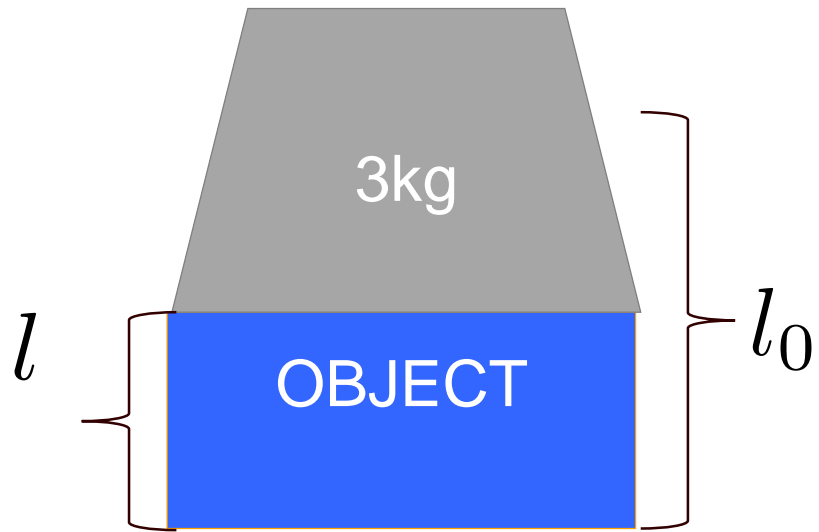
Simple Measurement



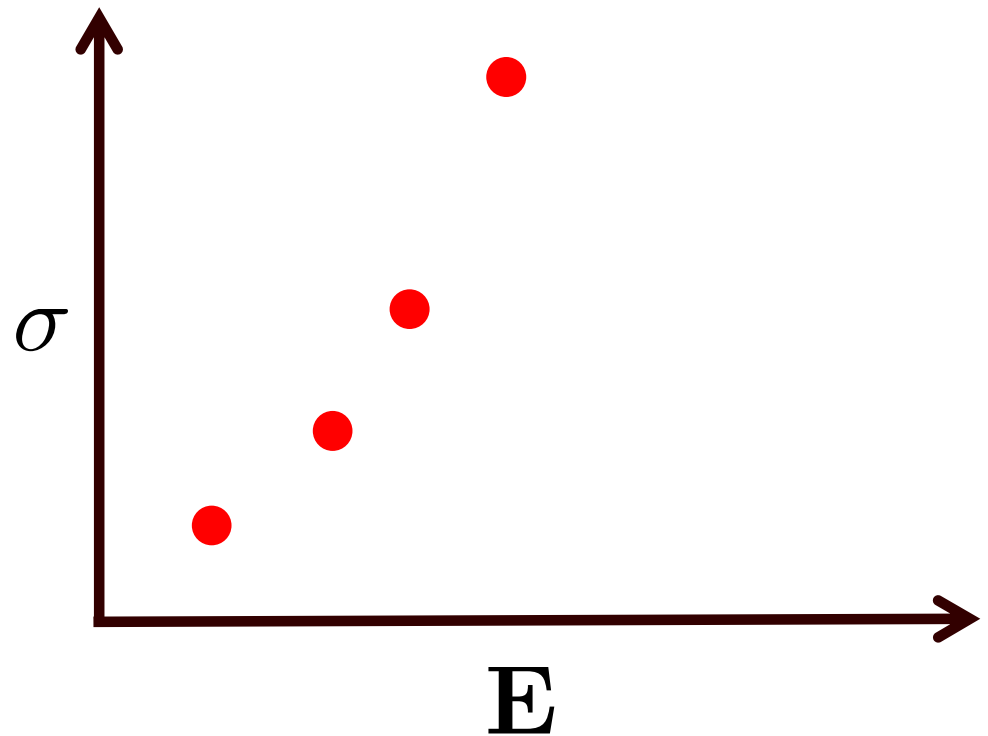
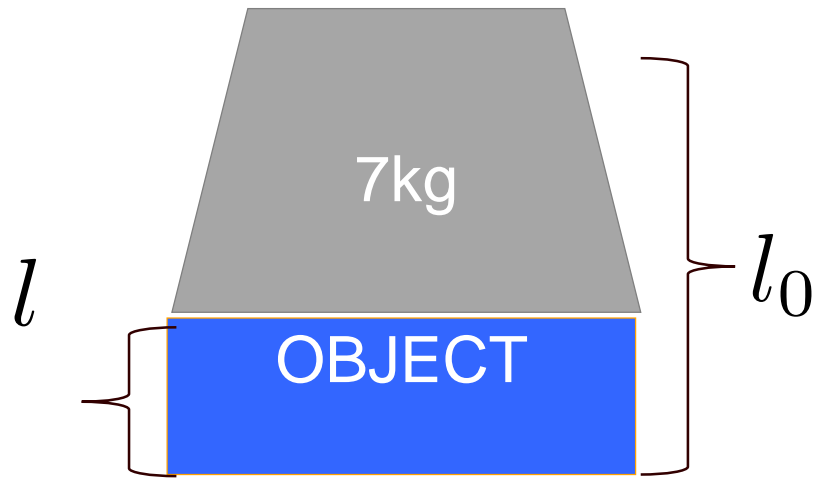
Simple Measurement



Simple Measurement

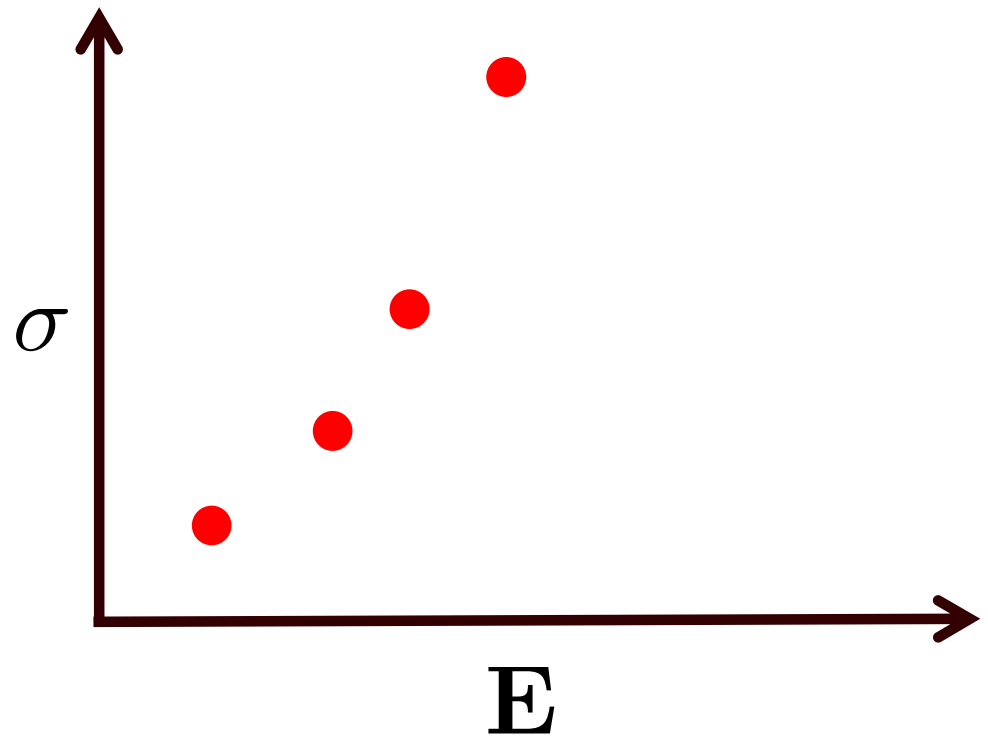
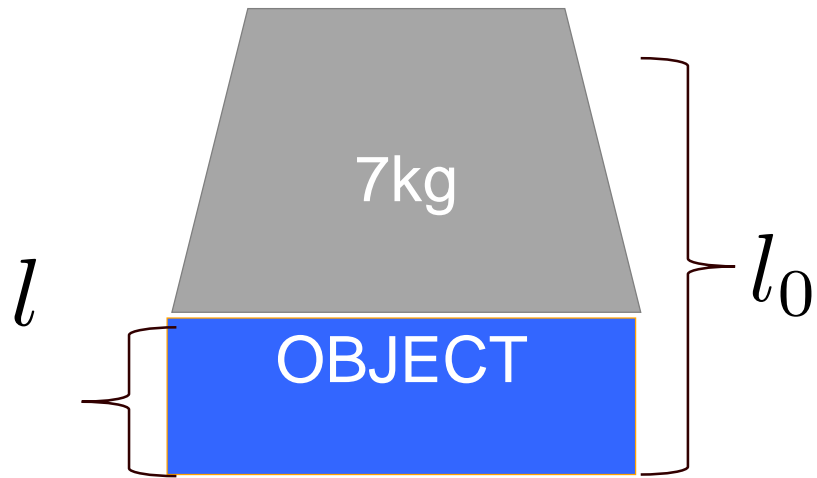


Simple Measurement



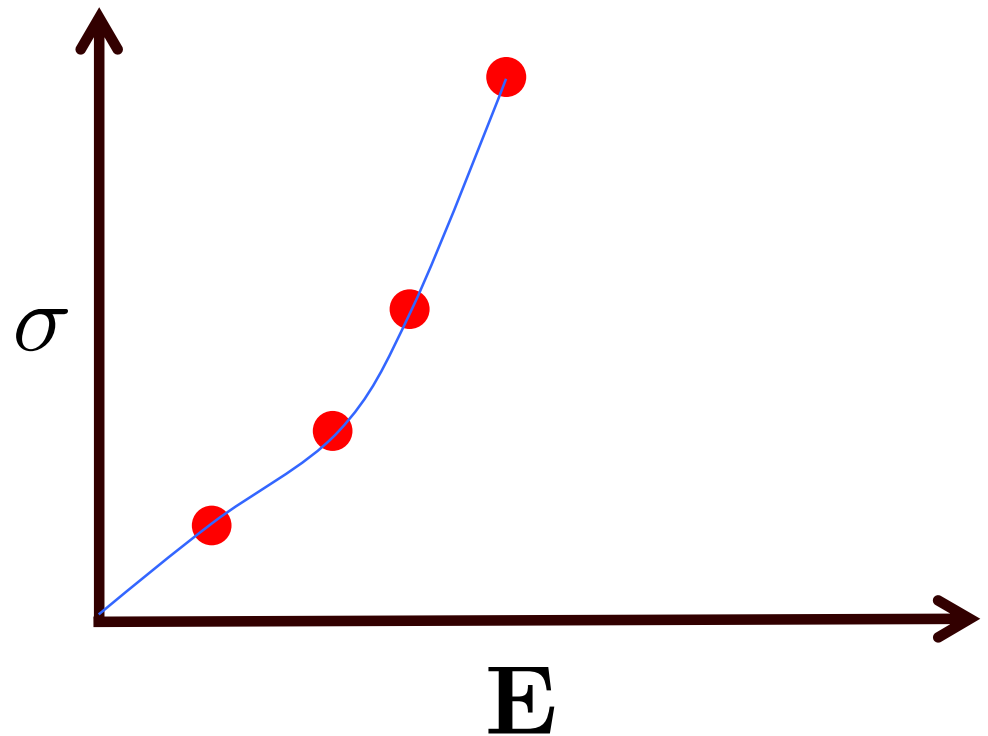
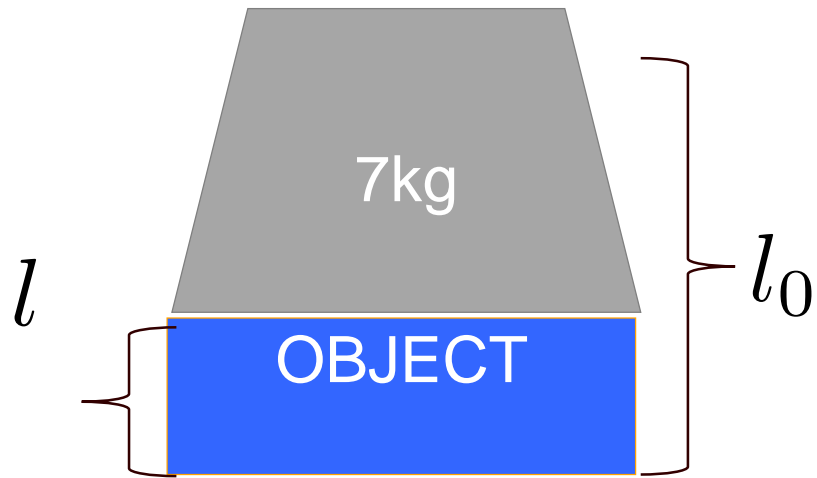
Simple Measurement

How do we get the stiffness ?



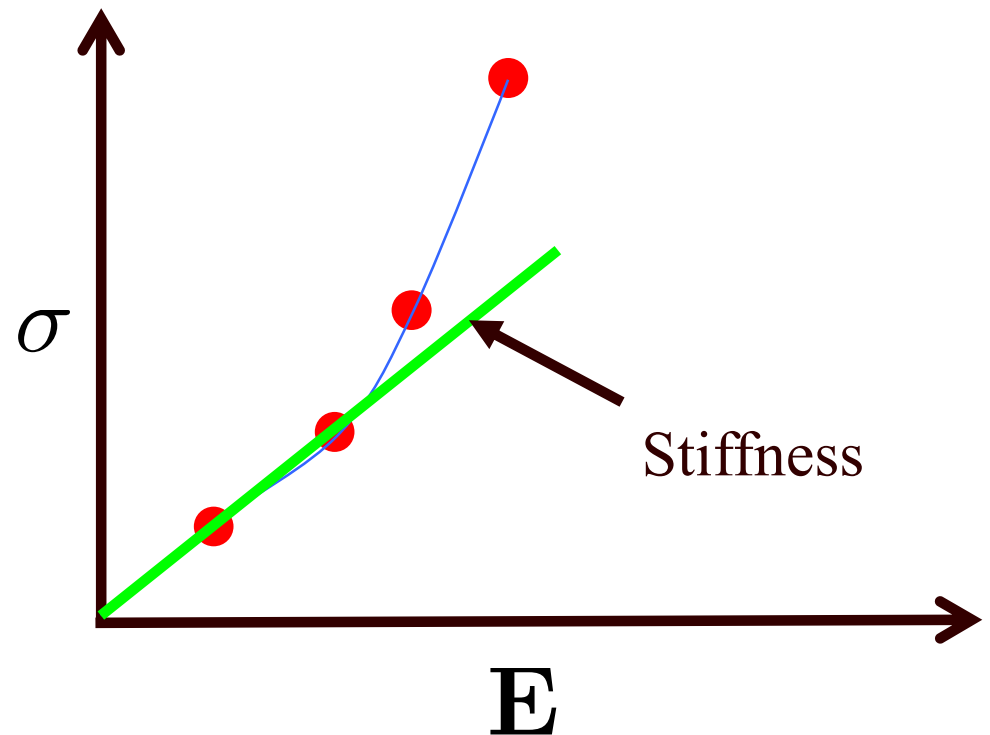
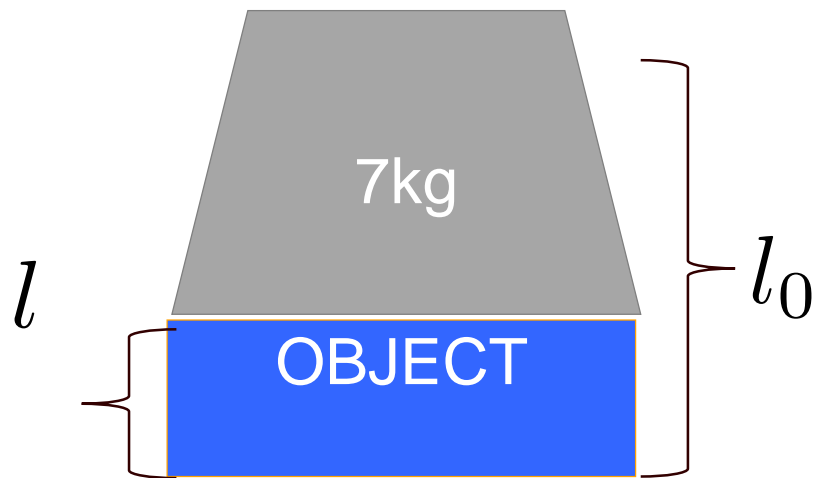
Simple Measurement

How do we get the stiffness ?

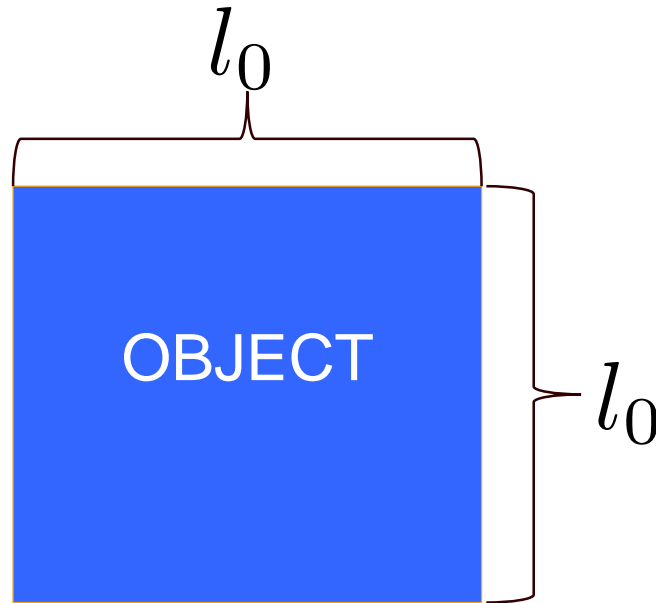


Simple Measurement

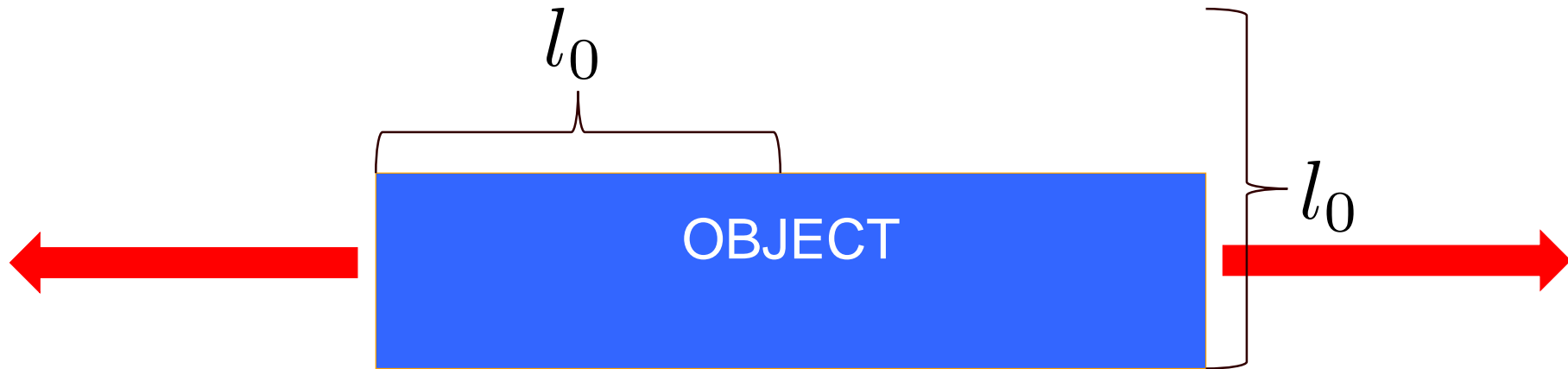
How do we get the stiffness ?



Simple Measurement: Poisson's Ratio

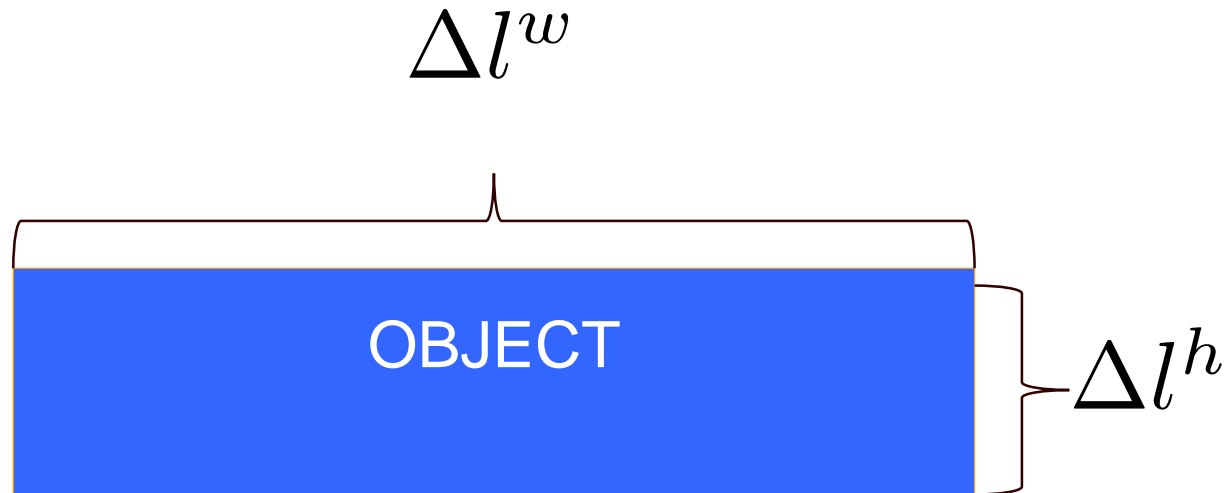


Simple Measurement: Poisson's Ratio



Simple Measurement: Poisson's Ratio

Compute changes in width and height



Simple Measurement: Poisson's Ratio

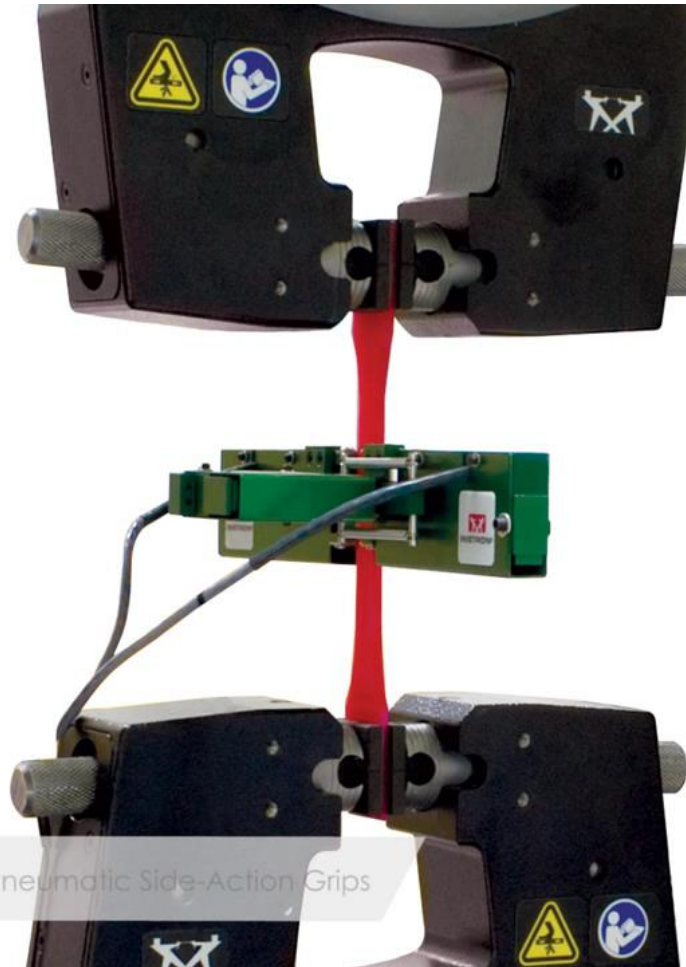
$$\nu = \frac{\Delta l_h}{\Delta l_w}$$

Poisson's Ratio

$$\frac{\Delta l^w}{l_0^w}$$



Measurement Devices

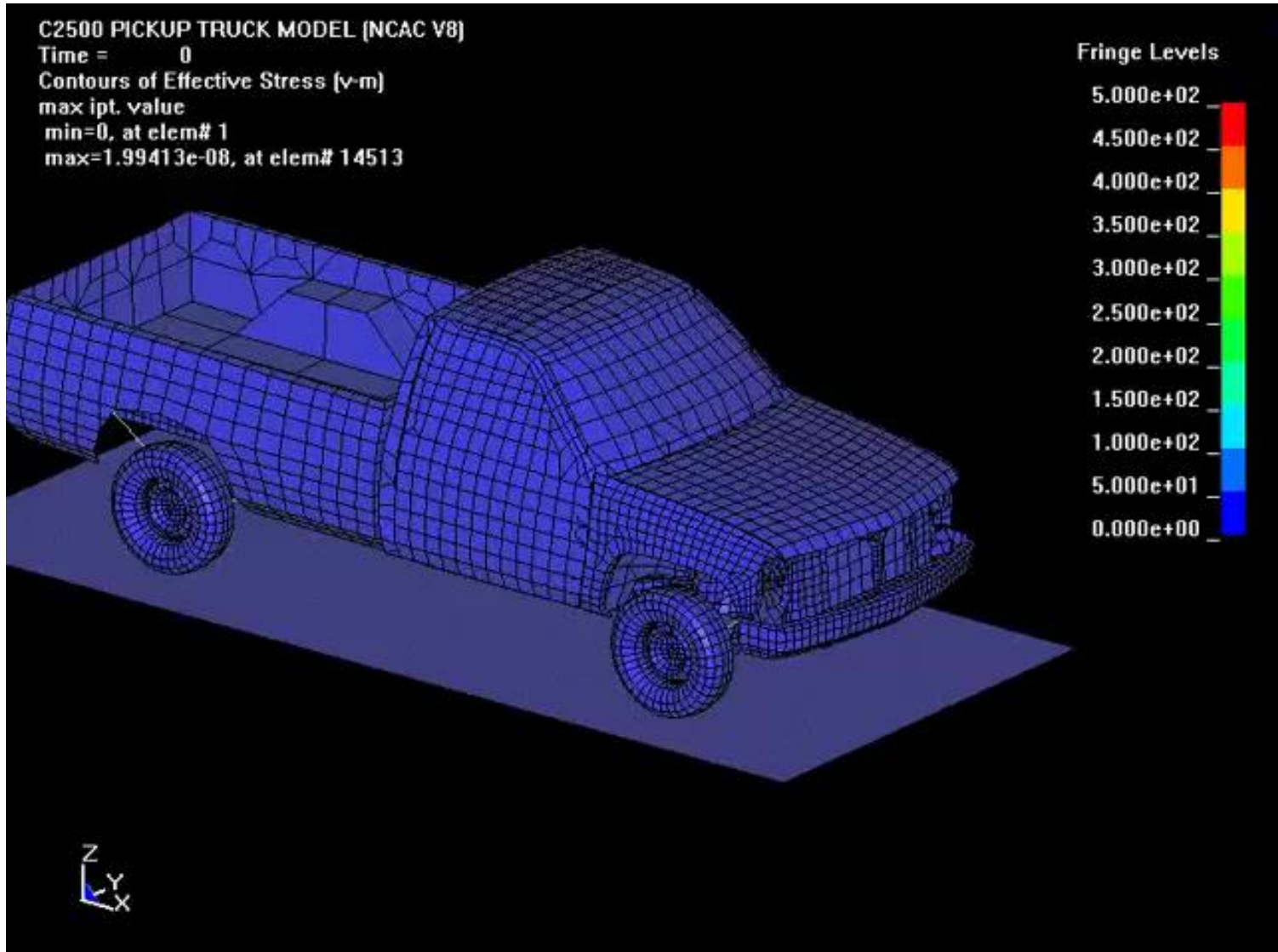


Bi-Axial Extensometer with 5 kN Pneumatic Side-Action Grips

Simulating Elastic Materials with CM+FEM

- ◆ You now have all the mathematical tools you need

Suppose you want to simulate this...



Plastic and Elastic Materials

◆ Elastic Materials

- Objects return to their original shape in the absence of other forces

◆ Plastic Deformations:

- Object does not always return to its original shape

Example: Crushing a Coke Can



Old Reference State



New Reference State

Example: Crushing a van



A Simple Model For Plasticity

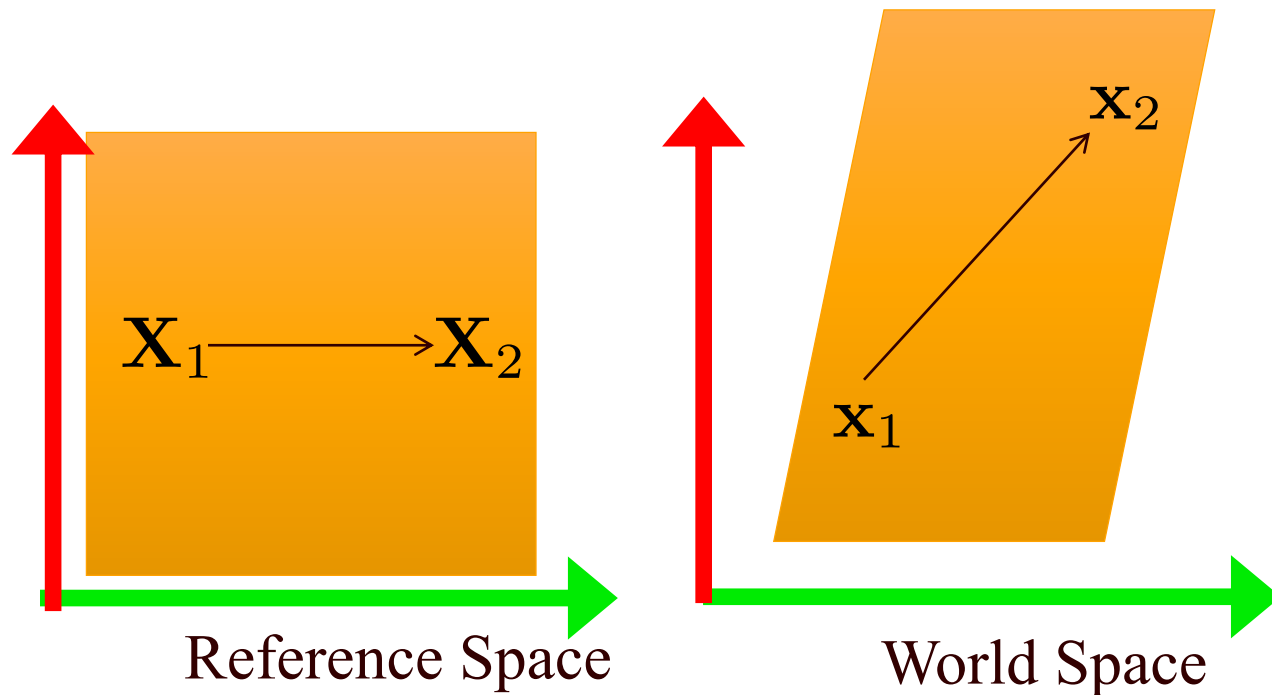
- ◆ Recall our model for strain: $\frac{1}{2} (\mathbf{F}^T \mathbf{F} - \mathbf{I})$
- ◆ Let's consider how to encode a change of reference shape into this metric
 - Changing undeformed mesh is not easy!
- ◆ We want to exchange \mathbf{F} with ${}^w_p \mathbf{F}$, a deformation gradient that takes into account the new shape of our object



New Reference State

Continuum Mechanics: Deformation

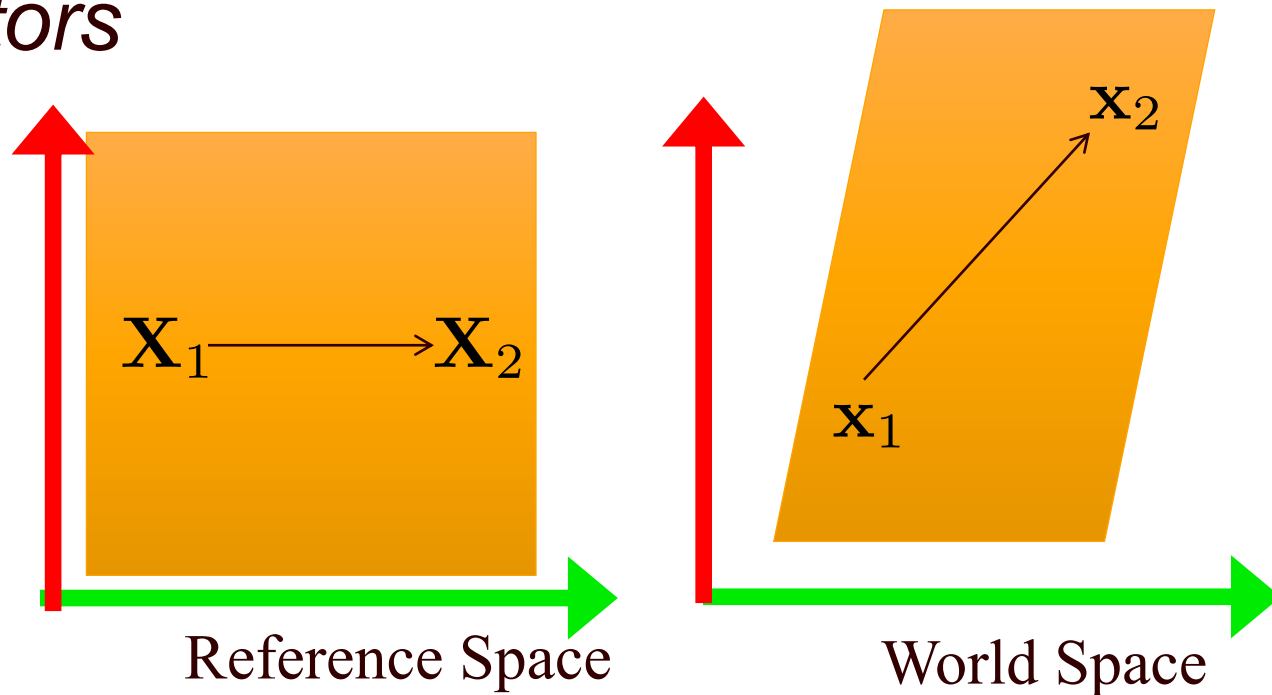
- ◆ *deformation gradient maps undeformed vectors (local) to deformed (world) vectors*



$$d\mathbf{x} \approx \mathbf{F}d\mathbf{X}$$

Continuum Mechanics: Deformation

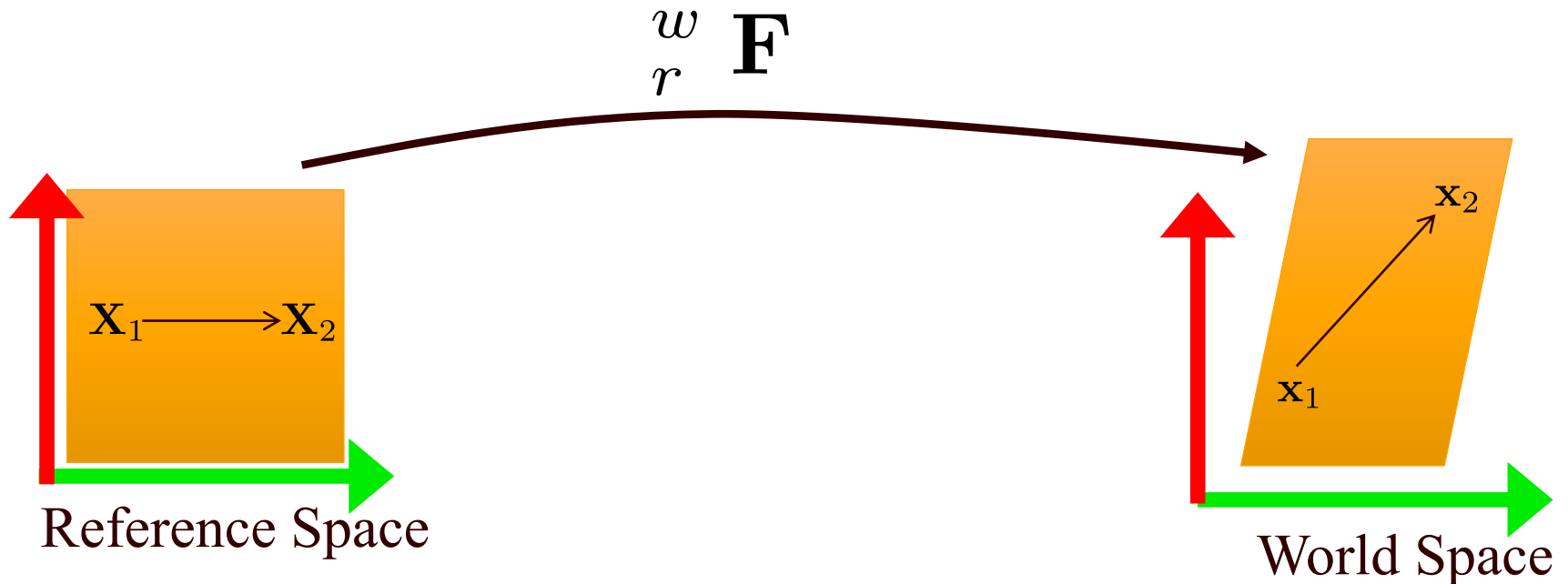
- ◆ *deformation gradient maps undeformed vectors (reference) to deformed (world) vectors*



$$d\mathbf{x} \approx_r^w \mathbf{F} d\mathbf{X}$$

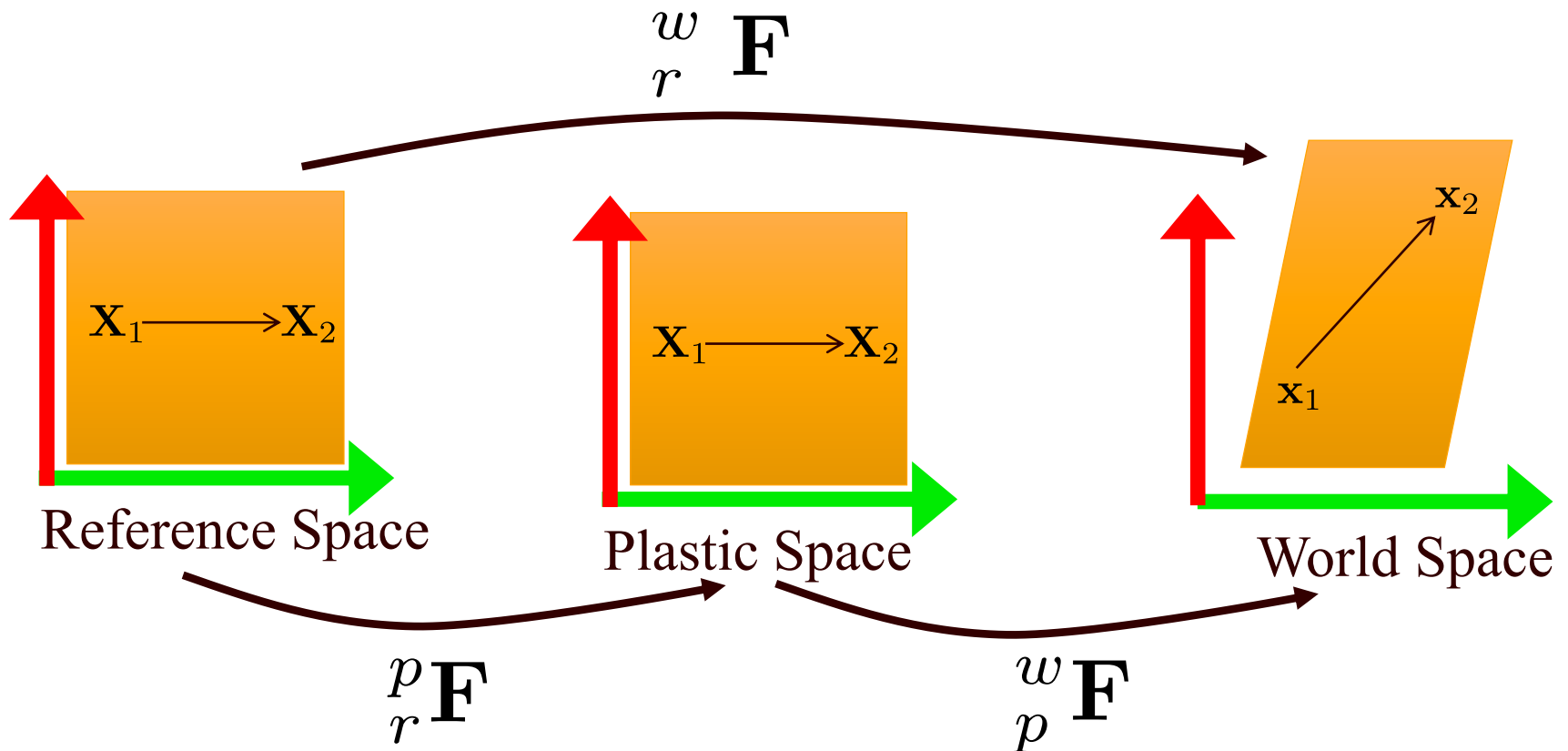
Continuum Mechanics: Deformation

- ◆ \mathbf{F} transforms a vector from Reference space to World Space



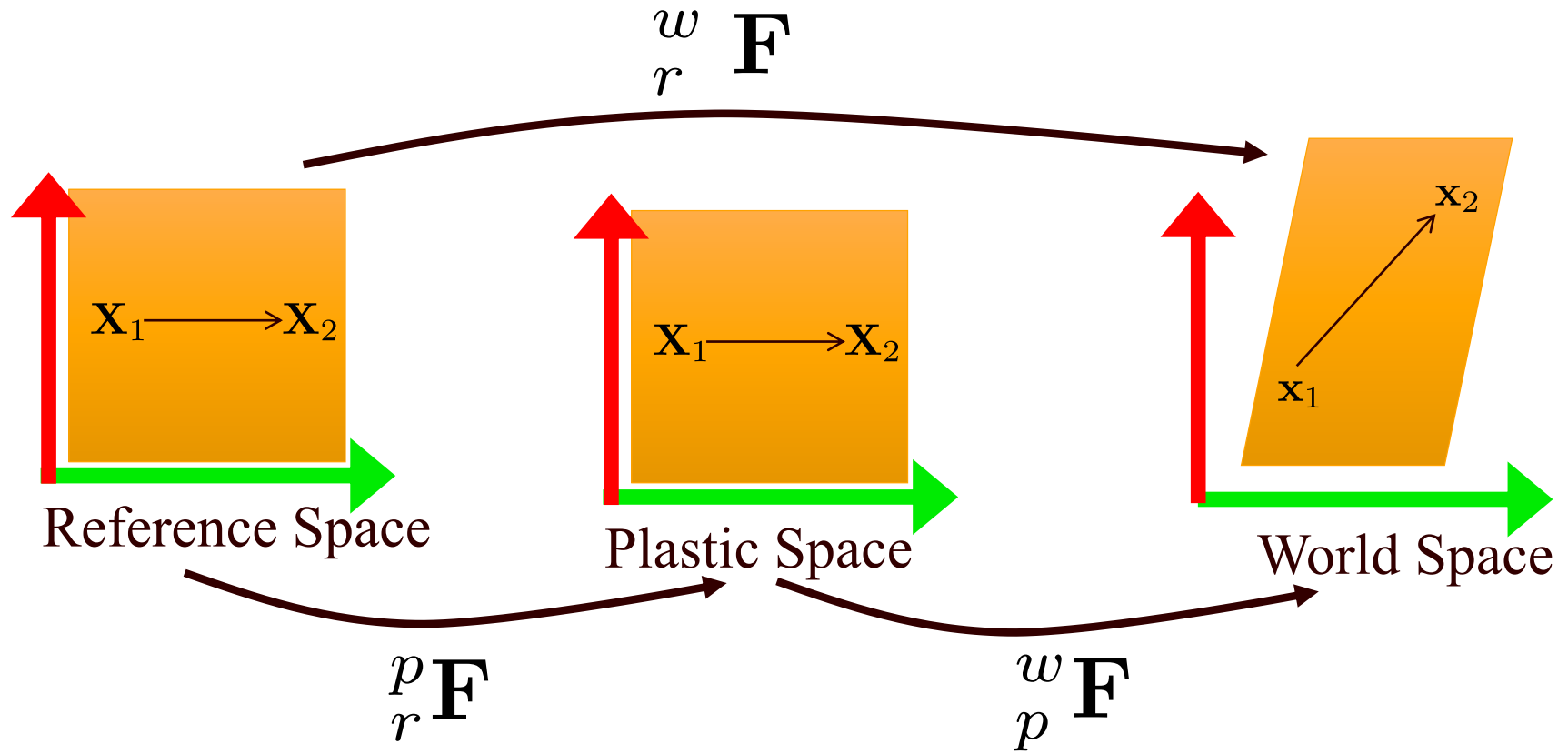
Continuum Mechanics: Deformation

◆ Introduce a new space



Continuum Mechanics: Deformation

- ◆ Our goal is to use ${}^w_p \mathbf{F}$ but we only have access to ${}^w_r \mathbf{F}$

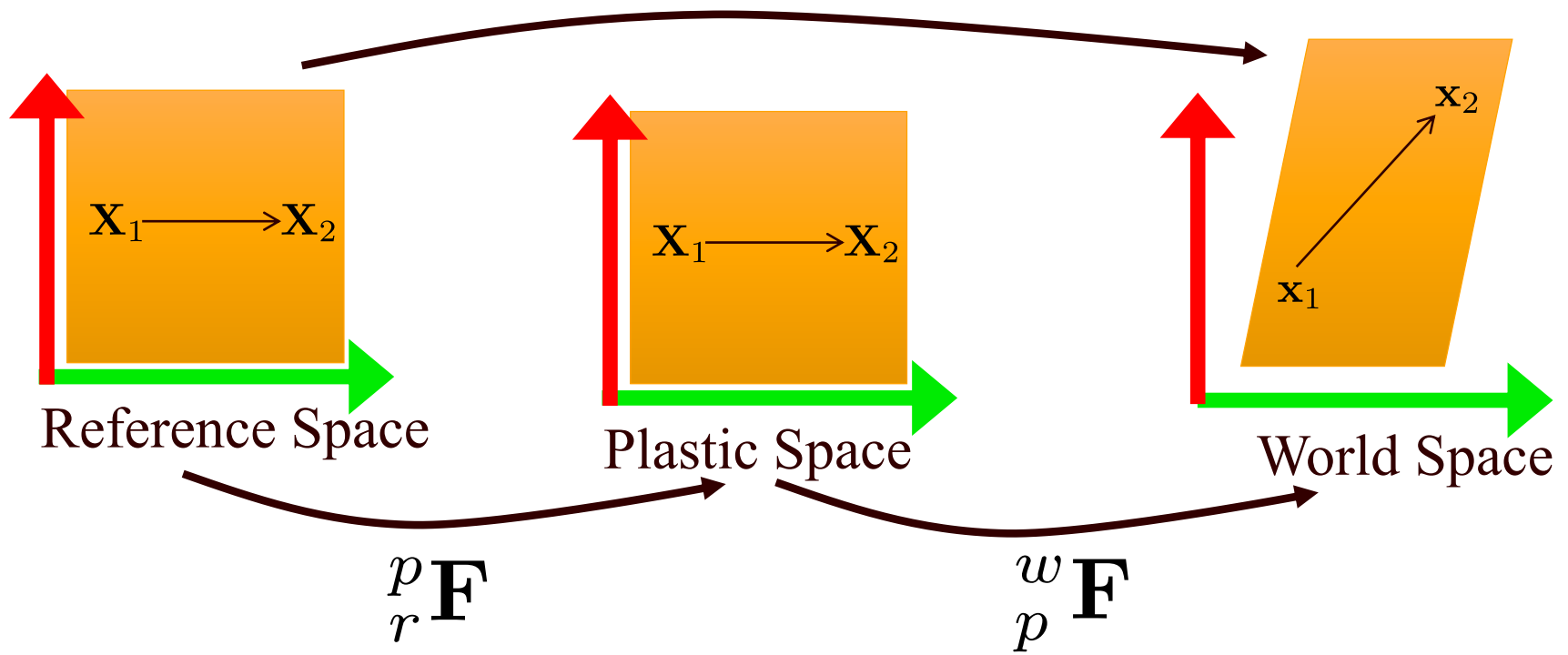


Continuum Mechanics: Deformation

◆ Our goal is to use ${}^w_p \mathbf{F}$ but we only have access to ${}^w_r \mathbf{F}$

$${}^w_p \mathbf{F} = {}^w_r \mathbf{F} {}^p_r \mathbf{F}^{-1}$$

$${}^w_r \mathbf{F}$$

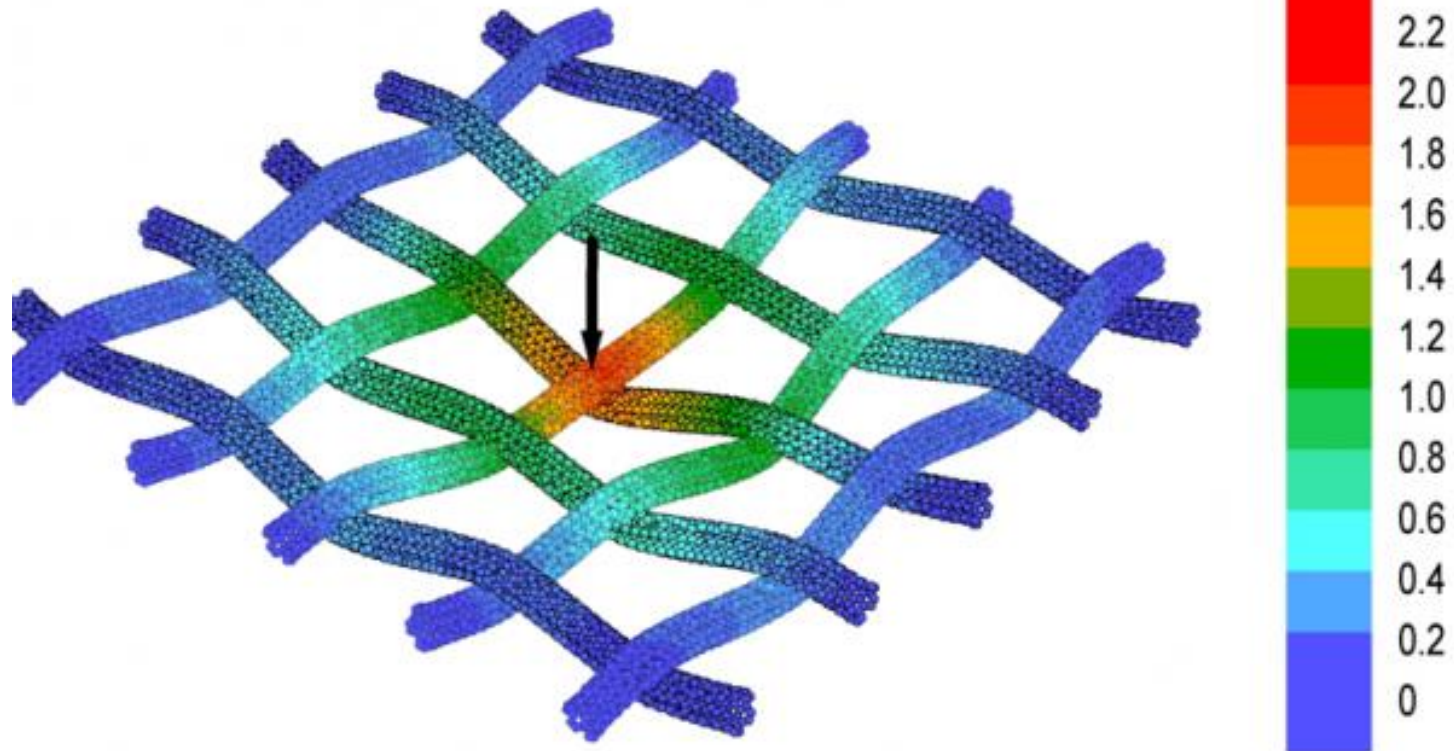


Continuum Mechanics: Deformation

- ◆ Our goal is to use ${}^w_p \mathbf{F}$ but we only have access to ${}^w_r \mathbf{F}$
$${}^w_p \mathbf{F} = {}^w_r \mathbf{F} {}^p_r \mathbf{F}^{-1}$$
- ◆ Keep an estimate of ${}^p_r \mathbf{F}^{-1}$ per element, built incrementally

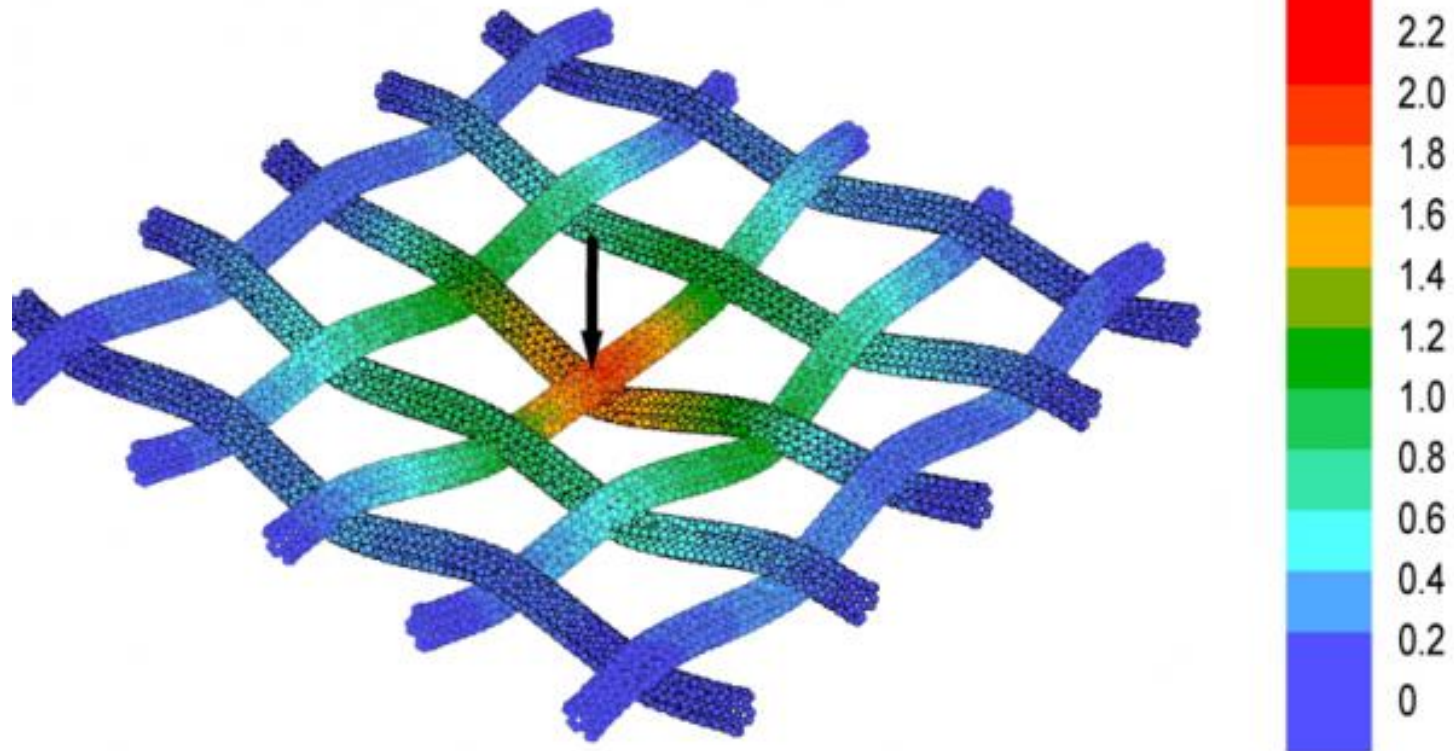
How to Compute the Plastic Deformation Gradient

- We compute the strain/stress for each element during simulation
- When it gets above a certain threshold store \mathbf{F} as $\frac{p}{r} \mathbf{F}$



How to Compute the Plastic Deformation Gradient

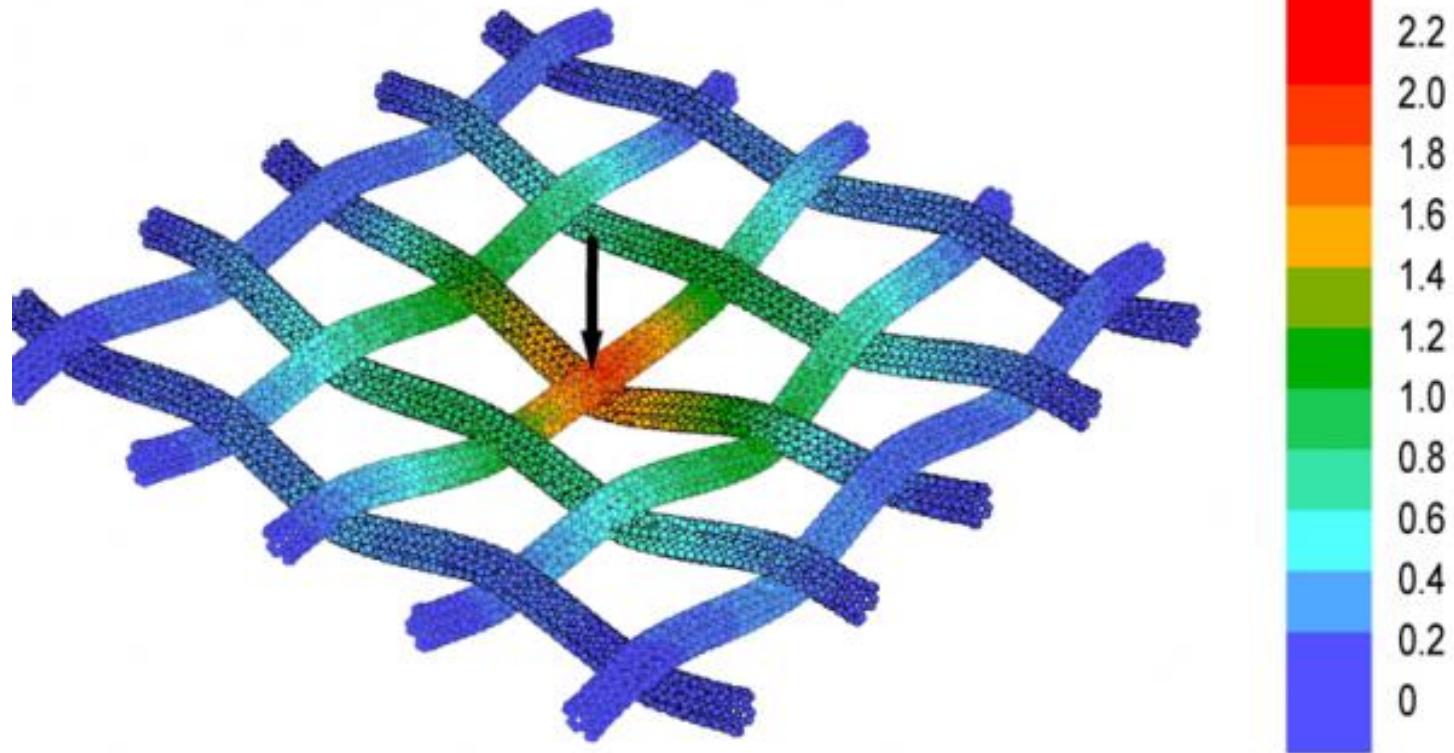
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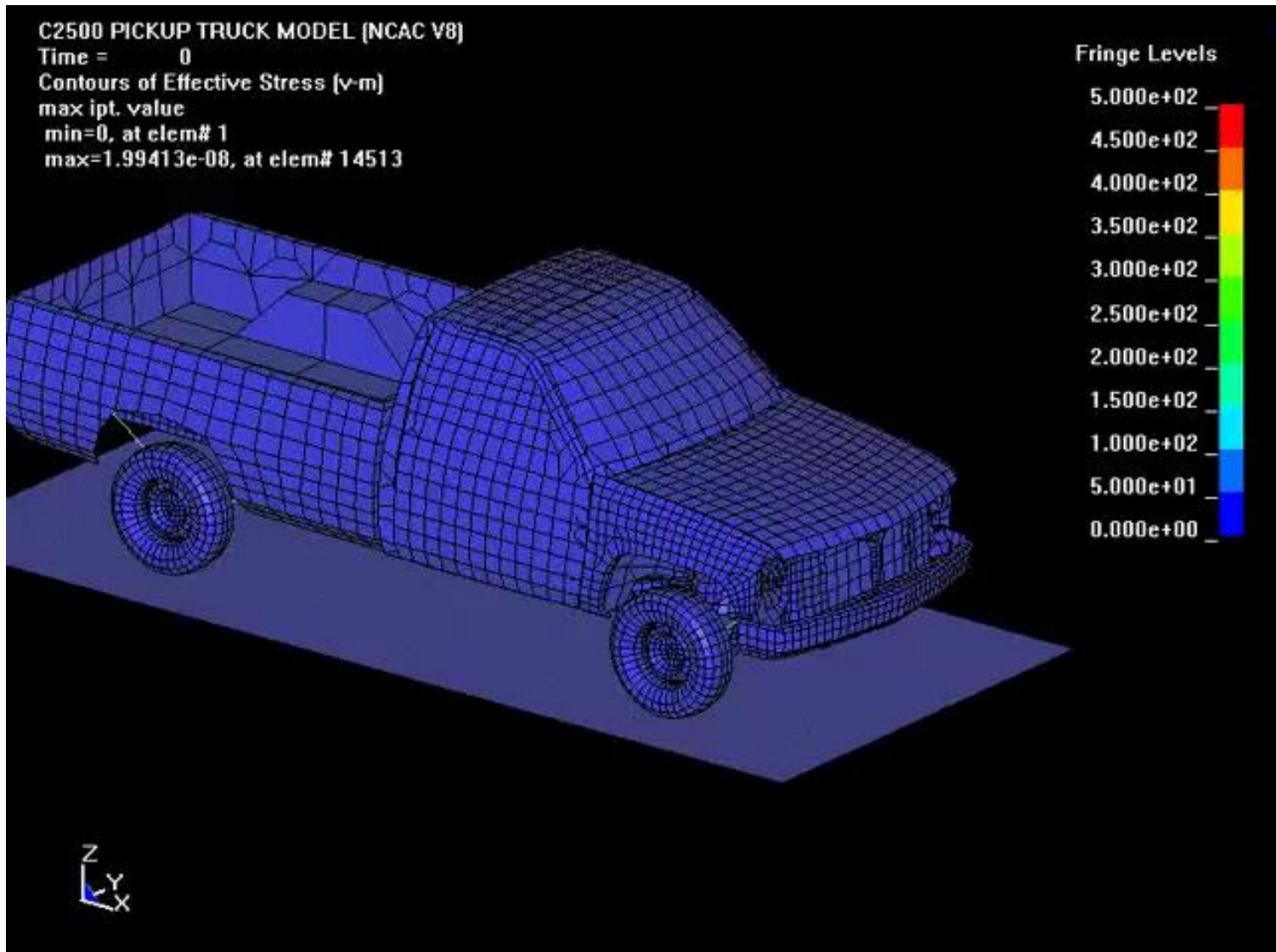
How to Compute the Plastic Deformation Gradient

- Each subsequent simulation step uses

$${}^w_p \mathbf{F} = {}^w_r \mathbf{F}_r^p \mathbf{F}^{-1}$$



So now you too can simulate this...



Questions?
