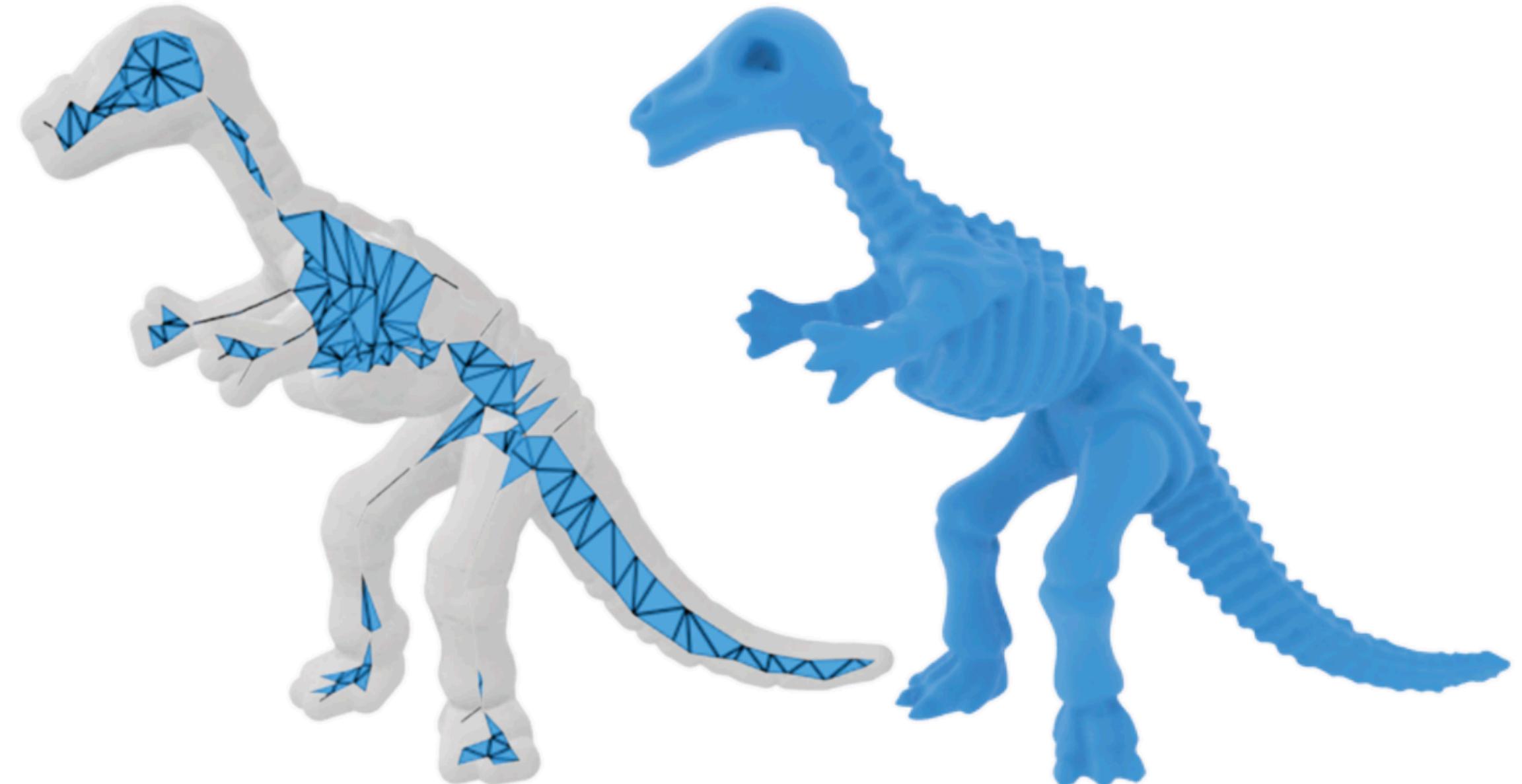


Instructor: Minchen Li



# Lec 12: Reduced Order Models

## 15-763: Physics-Based Animation of Solids and Fluids (S25)

# Recap: Frictional Self-Contact

Idea: Approximating Contact Forces as Conservative Forces

$$\begin{aligned} & \int_{\partial\Omega^0} Q_i(\mathbf{X}, t) T_i(\mathbf{X}, t) ds(\mathbf{X}) \\ &= \int_{\Gamma_D} Q_i(\mathbf{X}, t) T_{D|i}(\mathbf{X}, t) ds(\mathbf{X}) + \int_{\Gamma_N} Q_i(\mathbf{X}, t) T_{N|i}(\mathbf{X}, t) ds(\mathbf{X}) \\ &+ \int_{\Gamma_C} Q_i(\mathbf{X}, t) T_{C|i}(\mathbf{X}, t) ds(\mathbf{X}) + \int_{\Gamma_C} Q_i(\mathbf{X}, t) T_{F|i}(\mathbf{X}, t) ds(\mathbf{X}). \end{aligned}$$

(Here  $\Gamma_C$  can overlap with  $\Gamma_D$  or  $\Gamma_N$ )

# Recap: Normal Self-Contact Barrier Potential

$$\mathbf{T}_C(\mathbf{X}, t) = -\frac{\partial b(\min_{\mathbf{X}_2 \in \Gamma_C - \mathcal{N}(\mathbf{X})} \|\mathbf{x}(\mathbf{X}, t) - \mathbf{x}(\mathbf{X}_2, t)\|, \hat{d})}{\partial \mathbf{x}(\mathbf{X}, t)}$$

where  $\mathcal{N}(\mathbf{X}) = \{\mathbf{X}_N \in \mathbb{R}^d \mid \|\mathbf{X}_N - \mathbf{X}\| < r\}$  is an infinitesimal circle around  $\mathbf{X}$  with the radius  $r$  sufficiently small to avoid unnecessary contact forces between a point and its geodesic neighbors.

**Need  $\hat{d} \rightarrow 0$ ,  $r \rightarrow 0$ , and  $\hat{d}/r \rightarrow 0$ .**

**Barrier Potential:**

$$\int_{\Gamma_C} \frac{1}{2} b(\min_{\mathbf{X}_2 \in \Gamma_C - \mathcal{N}(\mathbf{X})} \|\mathbf{x}(\mathbf{X}, t) - \mathbf{x}(\mathbf{X}_2, t)\|, \hat{d}) ds(\mathbf{X}) \implies \int_{\Gamma_C} \frac{1}{2} \max_{e \in \mathcal{E} - I(\mathbf{X})} b(d^{\text{PE}}(\mathbf{x}(\mathbf{X}, t), e), \hat{d}) ds(\mathbf{X})$$

**But  $\min()$  is non-smooth!**

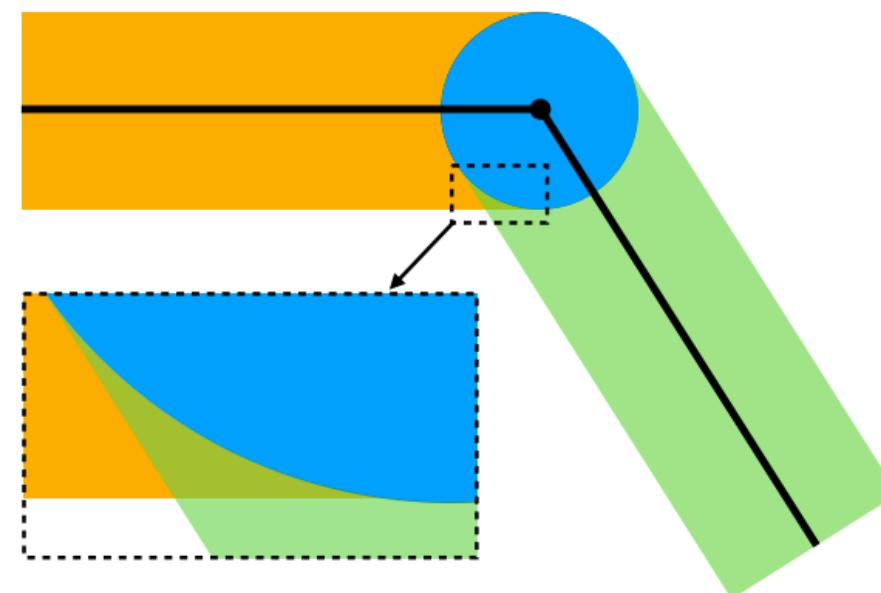
**$b()$  is monotonically decreasing,**

$$\max(a_1, a_2, \dots, a_n) \approx (a_1^p + a_2^p + \dots + a_n^p)^{\frac{1}{p}}$$

**Accurate when  $p \rightarrow \infty$ : Expensive!**

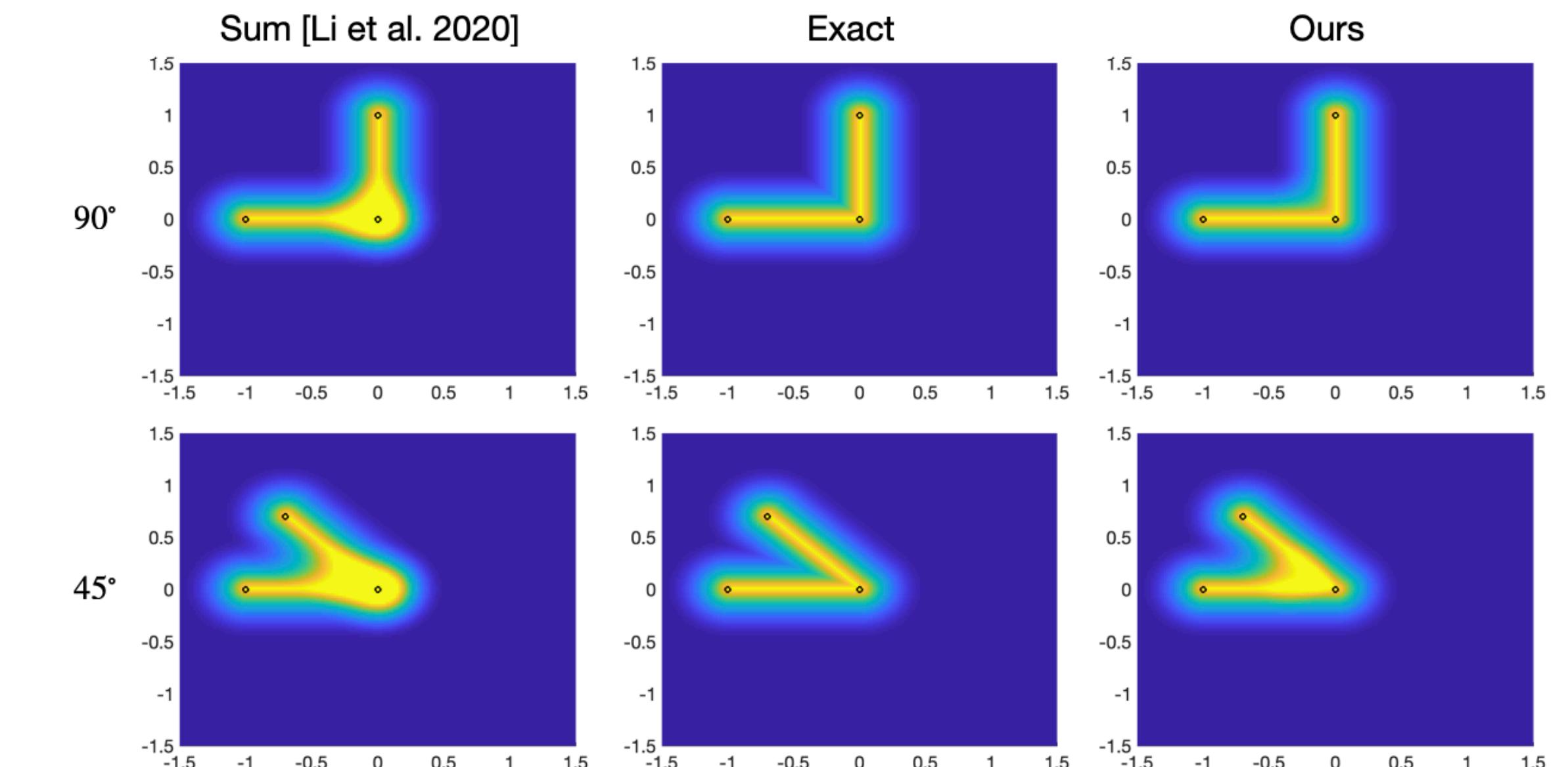
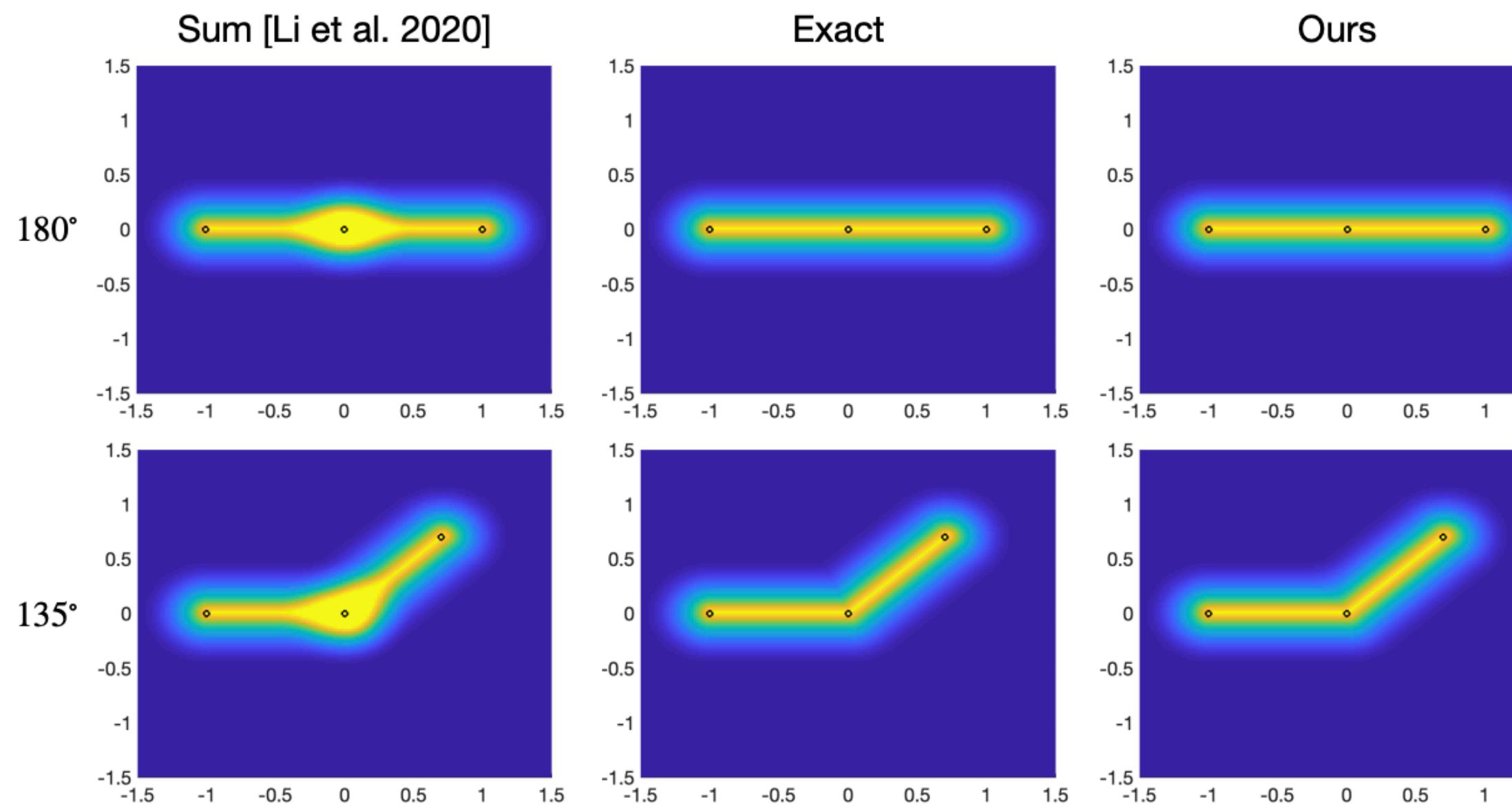
# Recap: Normal Self-Contact

## Smoothly Approximating the Barrier Potential



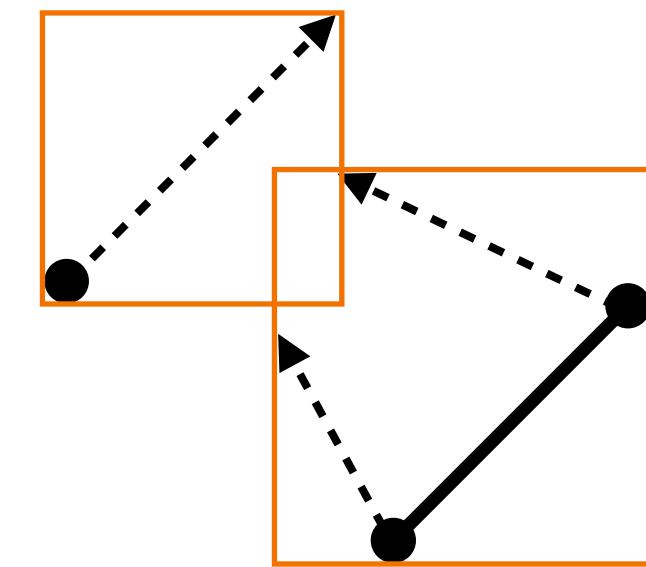
Can subtract the duplicate point-point barrier [Li et al. 2023]:

$$\Psi_c(x) = \sum_{e \in E \setminus x} b(d(x, e), \hat{d}) - \sum_{x_2 \in V_{int} \setminus x} b(d(x, x_2), \hat{d}) \approx \max_{e \in E \setminus x} b(d(x, e), \hat{d})$$

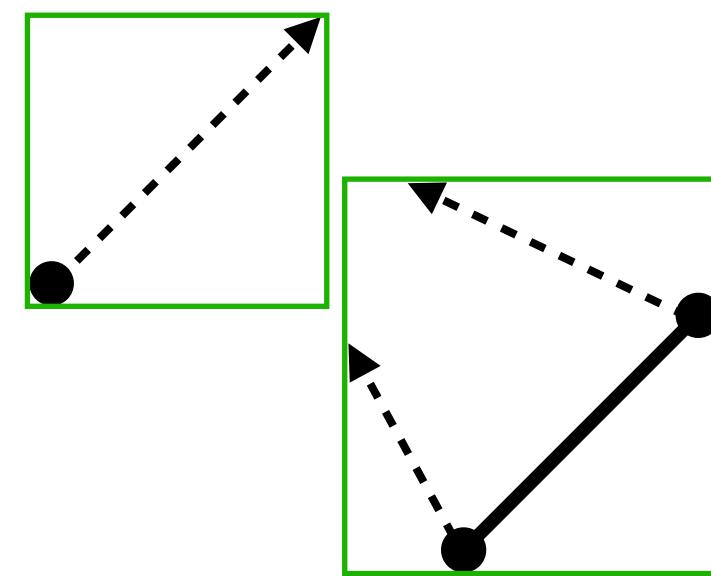


# Recap: Broad Phase CCD

- Step 1: query proximal primitive pairs using spatial data structures:
  - Spatial Hash
  - Bounding Box Hierarchy (BVH)
  - ...
- Step 2: Check bounding box overlap:



**Case 1: needs narrow phase**



**Case 2: can skip**

# Recap: Narrow Phase CCD

## Additive CCD [Li et al. 2021]

Taking a point-edge pair as an example, the key insight of ACCD is that, given the current positions  $\mathbf{p}$ ,  $\mathbf{e}_0$ ,  $\mathbf{e}_1$  and search directions  $\mathbf{d}_p$ ,  $\mathbf{d}_{e0}$ ,  $\mathbf{d}_{e1}$ , its TOI can be calculated as

$$\alpha_{\text{TOI}} = \frac{\|\mathbf{p} - ((1 - \lambda)\mathbf{e}_0 + \lambda\mathbf{e}_1)\|}{\|\mathbf{d}_p - ((1 - \lambda)\mathbf{d}_{e0} + \lambda\mathbf{d}_{e1})\|},$$

assuming  $(1 - \lambda)\mathbf{e}_0 + \lambda\mathbf{e}_1$  is the point on the edge that  $\mathbf{p}$  will first collide with. The issue is that we do not a priori know  $\lambda$ . But we can derive a lower bound of  $\alpha_{\text{TOI}}$  as

$$\begin{aligned} \alpha_{\text{TOI}} &\geq \frac{\min_{\lambda \in [0,1]} \|\mathbf{p} - ((1 - \lambda)\mathbf{e}_0 + \lambda\mathbf{e}_1)\|}{\|\mathbf{d}_p\| + \|(1 - \lambda)\mathbf{d}_{e0} + \lambda\mathbf{d}_{e1}\|} \\ &\geq \frac{d^{\text{PE}}(\mathbf{p}, \mathbf{e}_0, \mathbf{e}_1)}{\|\mathbf{d}_p\| + \max(\|\mathbf{d}_{e0}\|, \|\mathbf{d}_{e1}\|)} = \alpha_l \end{aligned}$$

```

 $\bar{p} \leftarrow \sum_i p_i / 4$ 
for  $i$  in  $\{0, 1, 2, 3\}$  do
   $p_i \leftarrow p_i - \bar{p}$ 

```

**Algorithm:**

```

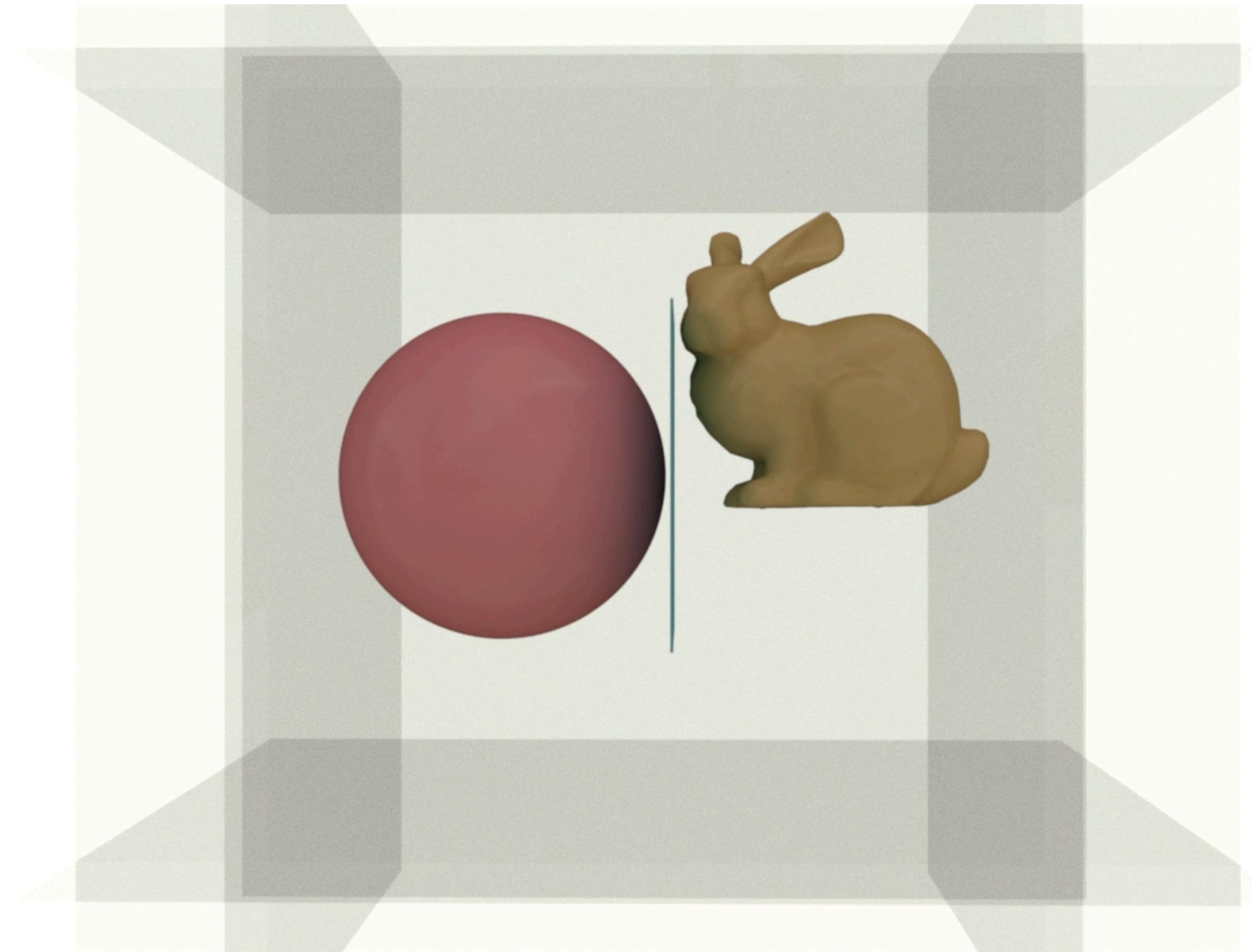
Make a local copy of  $x$ 
 $\alpha \leftarrow 0$ 
While distance not close enough
  Calculate lower bound  $\alpha_l$ 
   $x \leftarrow x + \alpha_l p$ 
   $\alpha \leftarrow \alpha + \alpha_l$ 
Return  $\alpha$ 

```

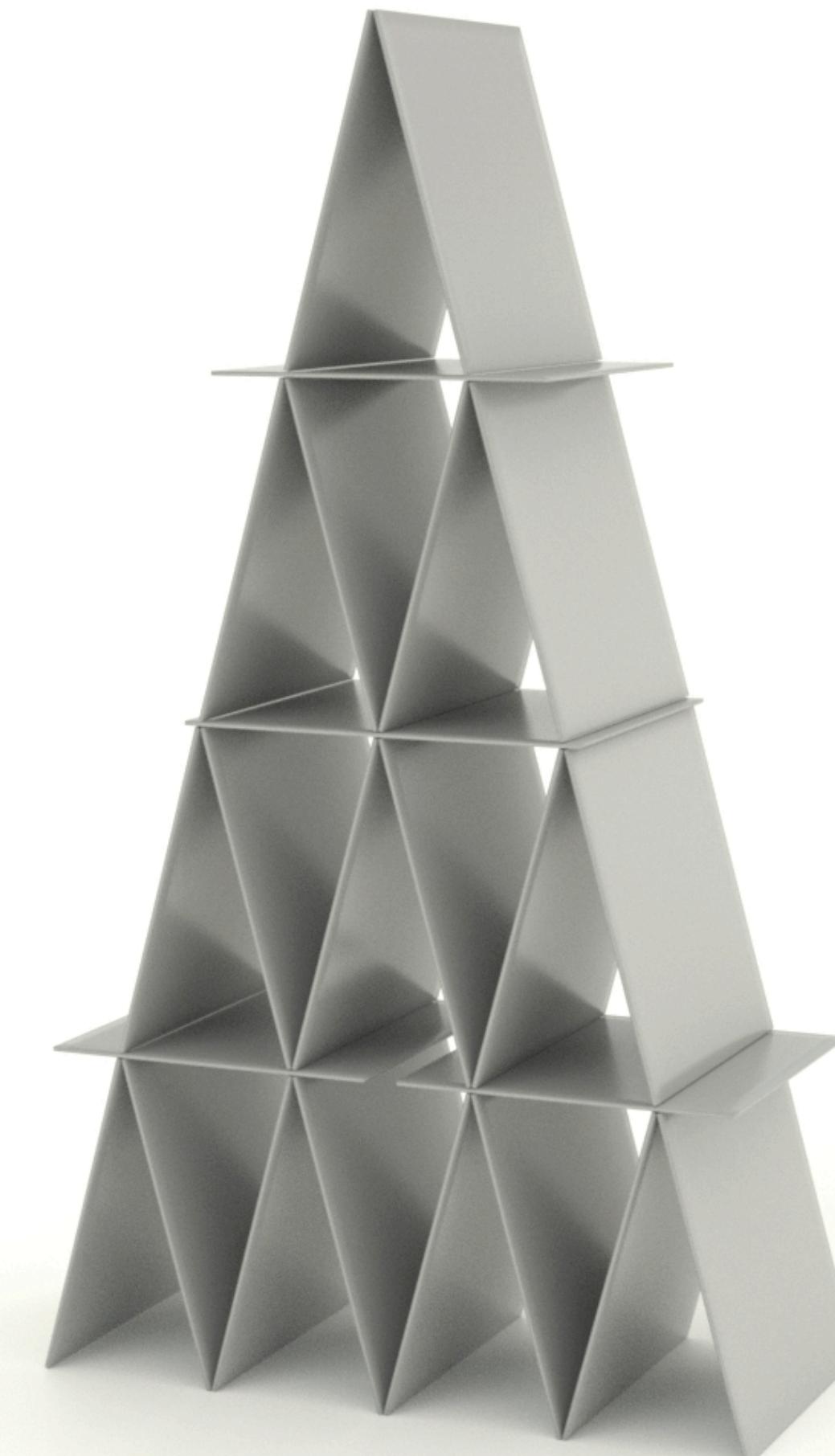
Only need to evaluate distances;  
 More robust than root-finding;  
 Generalize to higher-order primitives.

# Results: Elastic Body Simulation

With Guarantees of Nonpenetration, Non-inversion, and Convergence



$E = \sim 10^5 \text{ Pa}$



$E = \sim 10^9 \text{ Pa}$

# Today:

- Simulating Stiff Elastic Solids
- Modal-Order Reduction
- Implementation & Demo

**Today:**

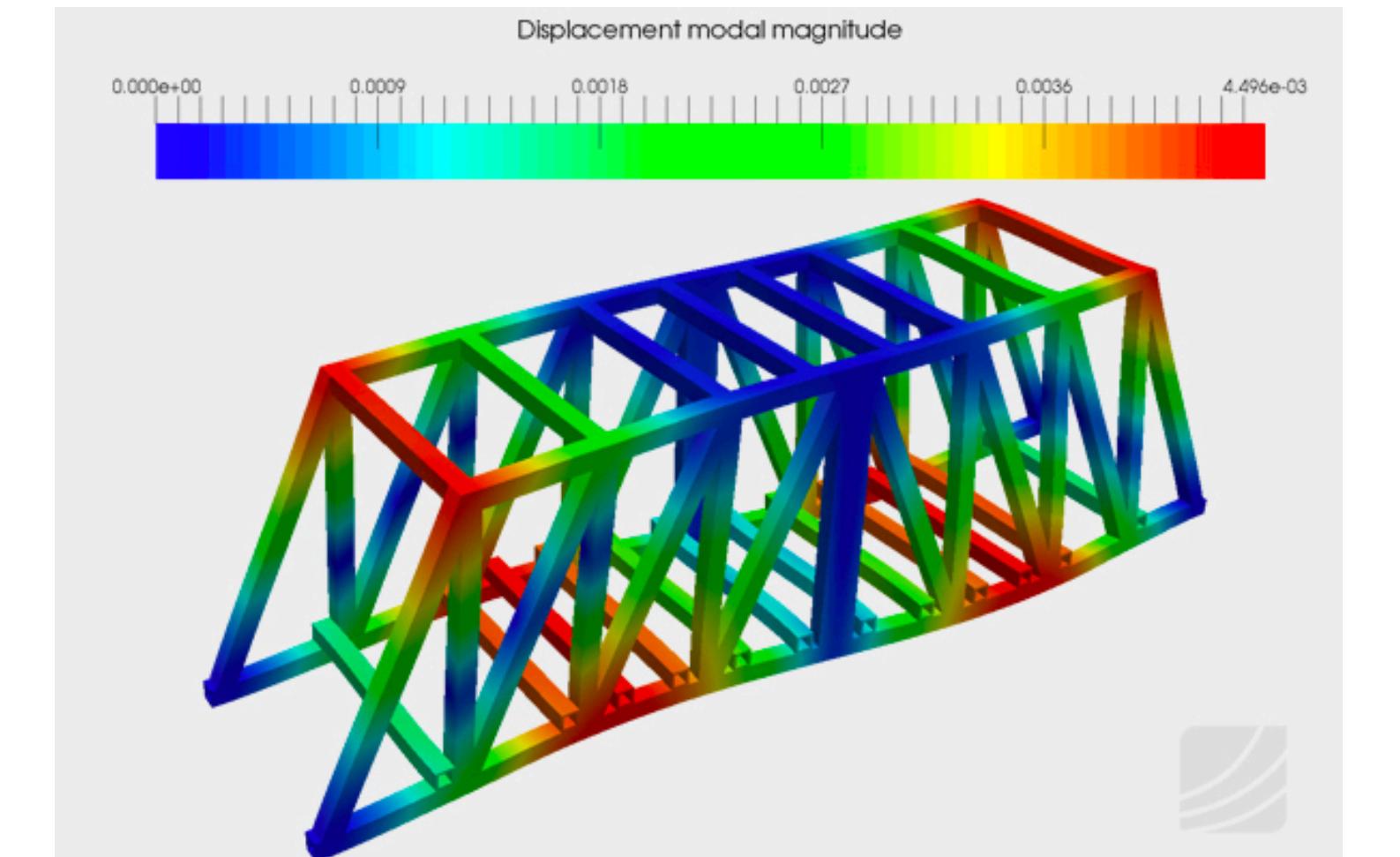
- **Simulating Stiff Elastic Solids**
- Modal-Order Reduction
- Implementation & Demo

# Simulating Super Stiff Materials

## Finite Element Method (FEM)

Material	Young's modulus (GPa)
Aluminium ( $_{13}\text{Al}$ )	68
Bone, human cortical	14
Gold	77.2
Wood, red maple	9.6 – 11.3
High-strength concrete	30

- Applications:



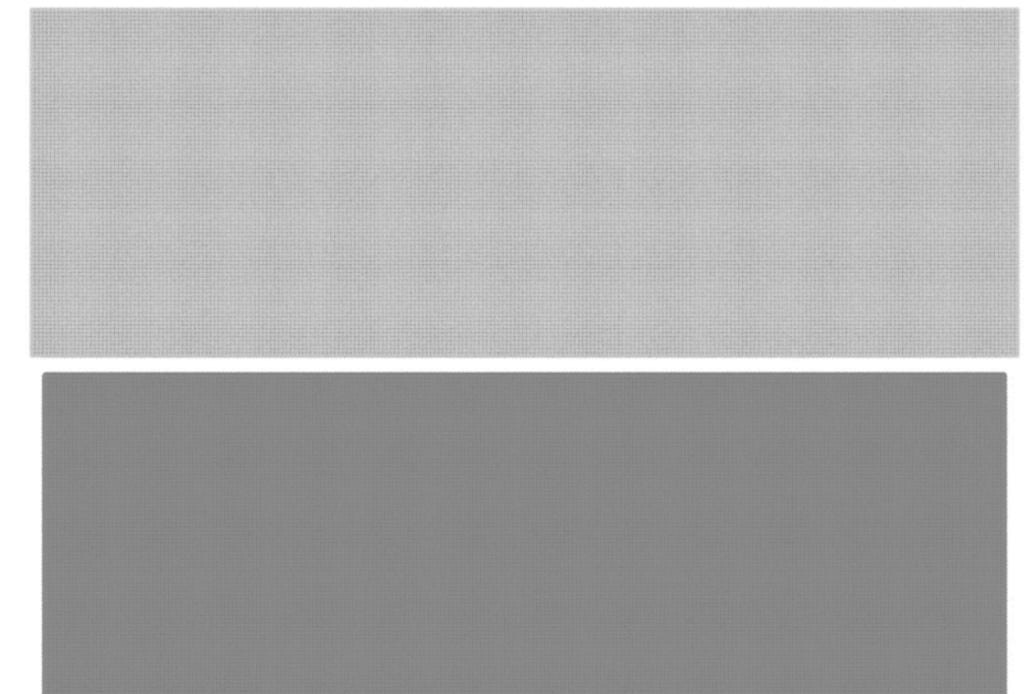
Structural Analysis

- Usually no visible deformation before fracture
- Can compute stress distribution using FEM

$$\min_{\text{structure}} \Psi$$

$$\text{s.t. } -\nabla_x \Psi + f^{\text{ext}} = 0$$

volume < target



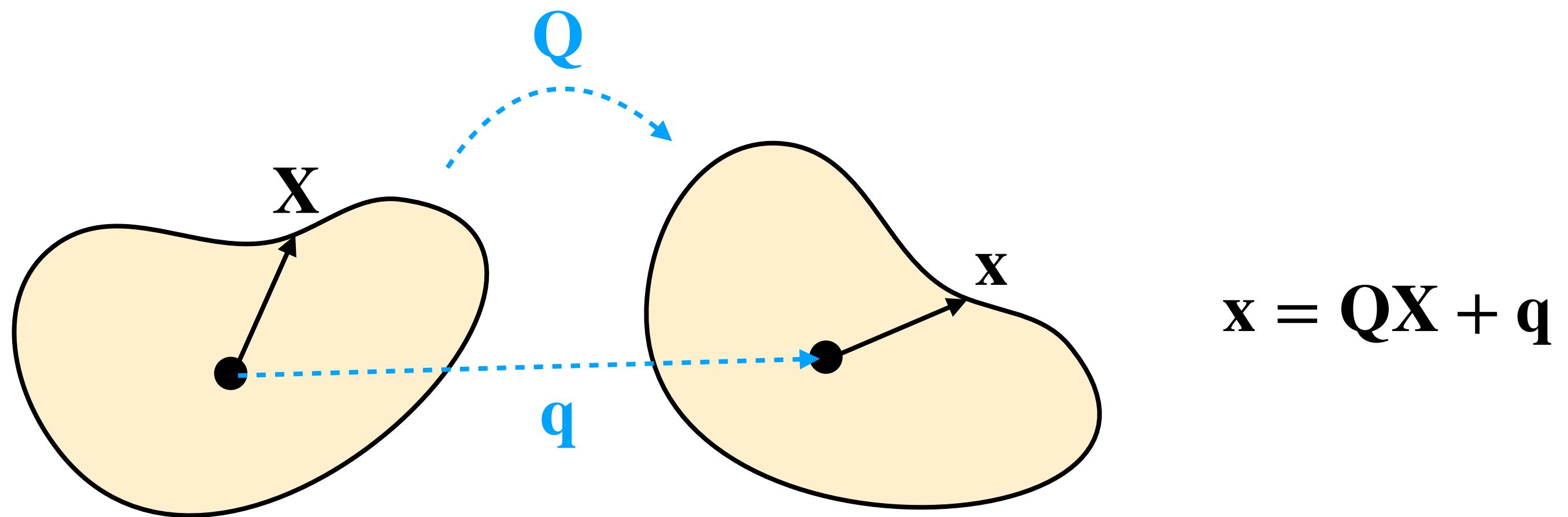
Topology Optimization

# Simulating Super Stiff Materials

## Rigid Body Representation

If only care about the motions,

Can simply track rotation  $Q$  and translation  $q$  per body!



Constant deformation gradient per body,  
No volumetric discretization needed!

# Simulating Super Stiff Materials

## Rigid Body Dynamics: Derivation

**Full order dynamics:**

$$\min_x \frac{1}{2} \|x - \tilde{x}^n\|_M^2 + h^2 \sum P(x)$$

**Reduced order DOF:**

$$\mathbf{x} = \mathbf{Q}\mathbf{X} + \mathbf{q} \in \mathbb{R}^3 \iff$$

$$\begin{aligned} x &= \bar{X}Q + \bar{S}q \in \mathbb{R}^{3l} \\ Q &\in \mathbb{R}^{9m}, \bar{X} \in \mathbb{R}^{3l \times 9m} \\ q &\in \mathbb{R}^{3m}, \bar{S} \in \mathbb{R}^{3l \times 3m} \end{aligned}$$

**Reduced order dynamics (from subspace optimization):**

$$\min_{Q,q} \frac{1}{2} \|\bar{X}Q + \bar{S}q - \tilde{x}^n\|_M^2 + h^2 \sum P(\bar{X}Q + \bar{S}q) \quad \text{s.t.} \quad \mathbf{Q}^T \mathbf{Q} = \mathbf{I} \quad \forall \mathbf{Q} \quad (\text{or } f(Q) = 0)$$

$\implies$

$$\bar{X}^T M(\bar{X}Q + \bar{S}q - \tilde{x}^n) + h^2 \sum \bar{X}^T \nabla P(\bar{X}Q + \bar{S}q) + (\nabla f(Q))^T \lambda = 0$$

$$\bar{S}^T M(\bar{X}Q + \bar{S}q - \tilde{x}^n) + h^2 \sum \bar{S}^T \nabla P(\bar{X}Q + \bar{S}q) = 0$$

$$f(Q) = 0$$

$l$ : number of nodes,  
 $m$ : number of bodies

**Alternative derivations:**

- **Lagrangian Mechanics;**
- **Linear and Angular Momentum Conservations;**
- ...

# Simulating Super Stiff Materials

## Rigid Body Dynamics: Mass Matrix and Inertia Tensor

Reduced order dynamics (from subspace optimization):

$$\bar{X}^T M(\bar{X}Q + \bar{S}q - \tilde{x}^n) + h^2 \sum \bar{X}^T \nabla P(\bar{X}Q + \bar{S}q) + (\nabla f(Q))^T \lambda = 0$$

$$\bar{S}^T M(\bar{X}Q + \bar{S}q - \tilde{x}^n) + h^2 \sum \bar{S}^T \nabla P(\bar{X}Q + \bar{S}q) = 0$$

$$f(Q) = 0$$

- $\bar{X}^T M \bar{X}$  is the mass matrix of  $Q$  related to inertia tensor

Calculating  $\bar{X}^T M \bar{X}$  without volumetric discretization:

1. Convert to continuous form  $\int_{\Omega^0} \rho \mathbf{X} \mathbf{X}^T d\mathbf{X}$
2. Transform to surface integral using Divergence Theorem
3. Discretize the surface integral

# Simulating Super Stiff Materials

## Rigid Body Dynamics: Change of Variable

Reduced order dynamics (from subspace optimization):

$$\min_{Q,q} \frac{1}{2} \|\bar{X}Q + \bar{S}q - \tilde{x}^n\|_M^2 + h^2 \sum P(\bar{X}Q + \bar{S}q) \quad \text{s.t.} \quad Q^T Q = I \quad \forall Q \quad (\text{or } f(Q) = 0)$$

Use rotation vector  $\theta$ :

$$\min_{\theta,q} \frac{1}{2} \|\bar{X}R(\theta) + \bar{S}q - \tilde{x}^n\|_M^2 + h^2 \sum P(\bar{X}R(\theta) + \bar{S}q)$$

Unconstrained!  
6 DOF per body!

Rodrigues' Rotation Formula:

$$\mathcal{R}(\theta) = \text{Id} + \sin(\|\theta\|) \left[ \frac{\theta}{\|\theta\|} \right] + (1 - \cos(\|\theta\|)) \left[ \frac{\theta}{\|\theta\|} \right]^2$$

Highly nonlinear!

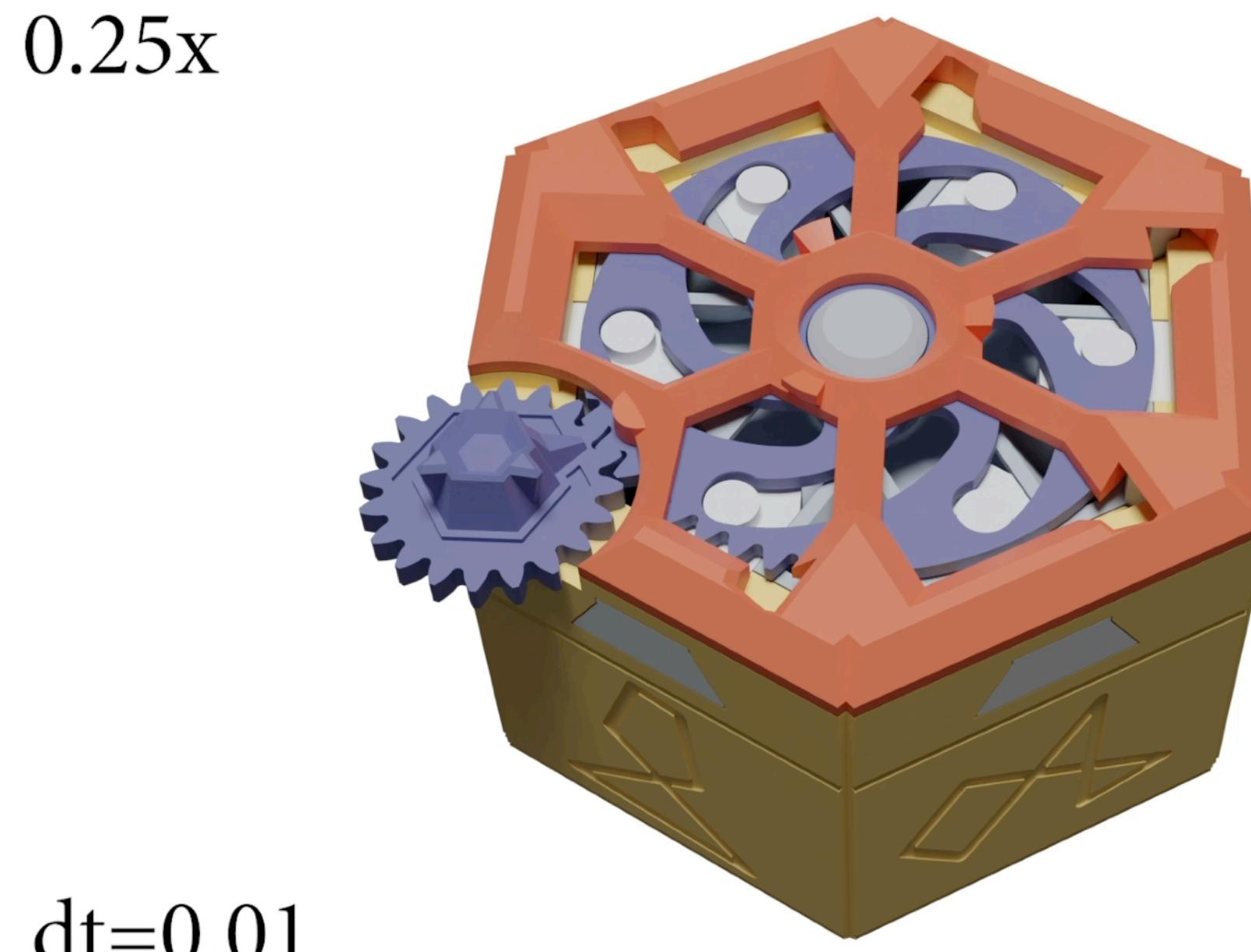
# Simulating Super Stiff Materials

## Rigid Body Dynamics: Frictional Contact via IPC [Li et al. 2020]

Reduced order dynamics (from subspace optimization):

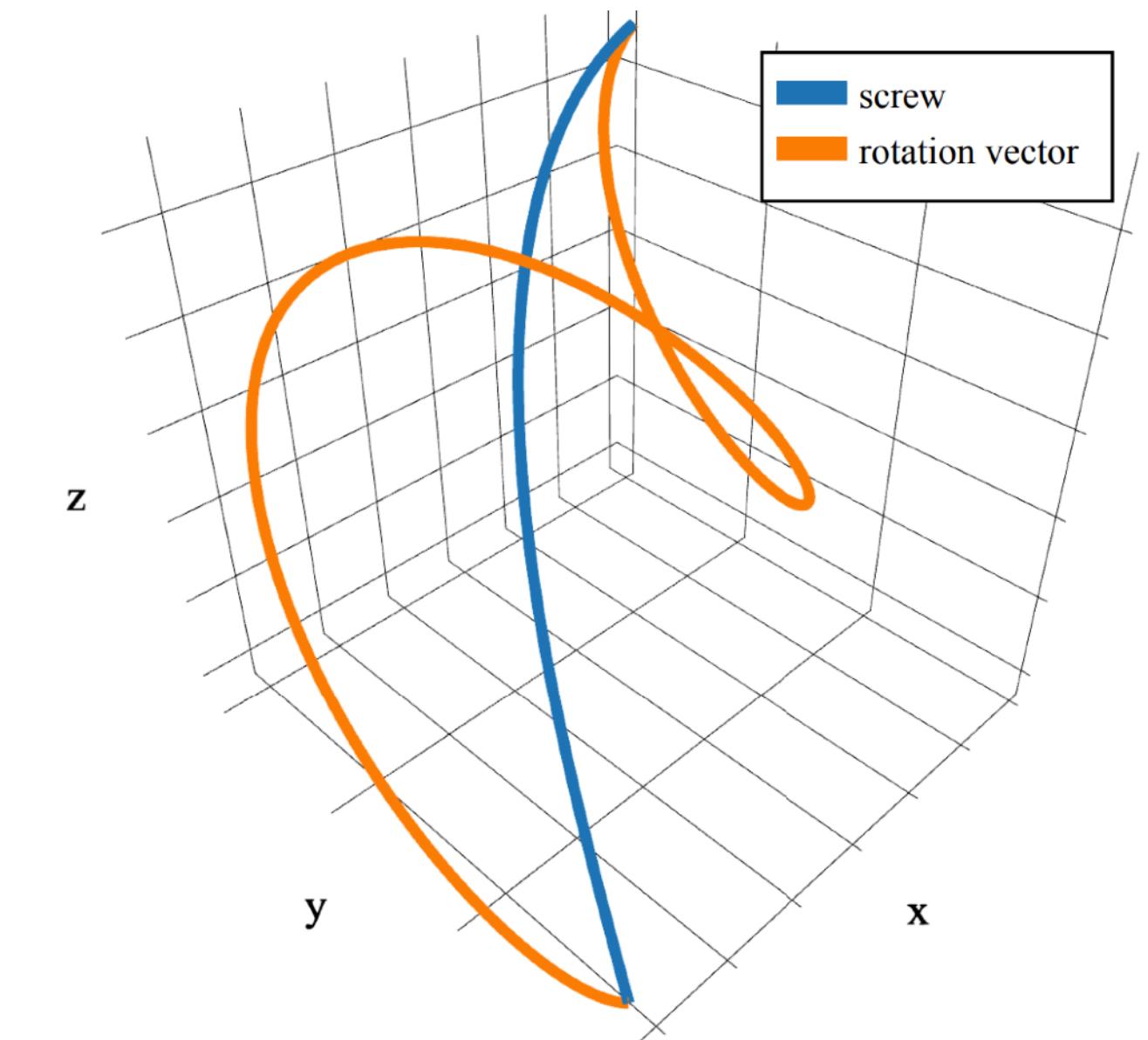
$$\min_{\theta, q} \frac{1}{2} \|\bar{X}R(\theta) + \bar{S}q - \tilde{x}^n\|_M^2 + h^2 \sum P(\bar{X}R(\theta) + \bar{S}q)$$

Just include IPC energies here



But line search is on  $\theta$ , and  $x(\theta)$  is nonlinear

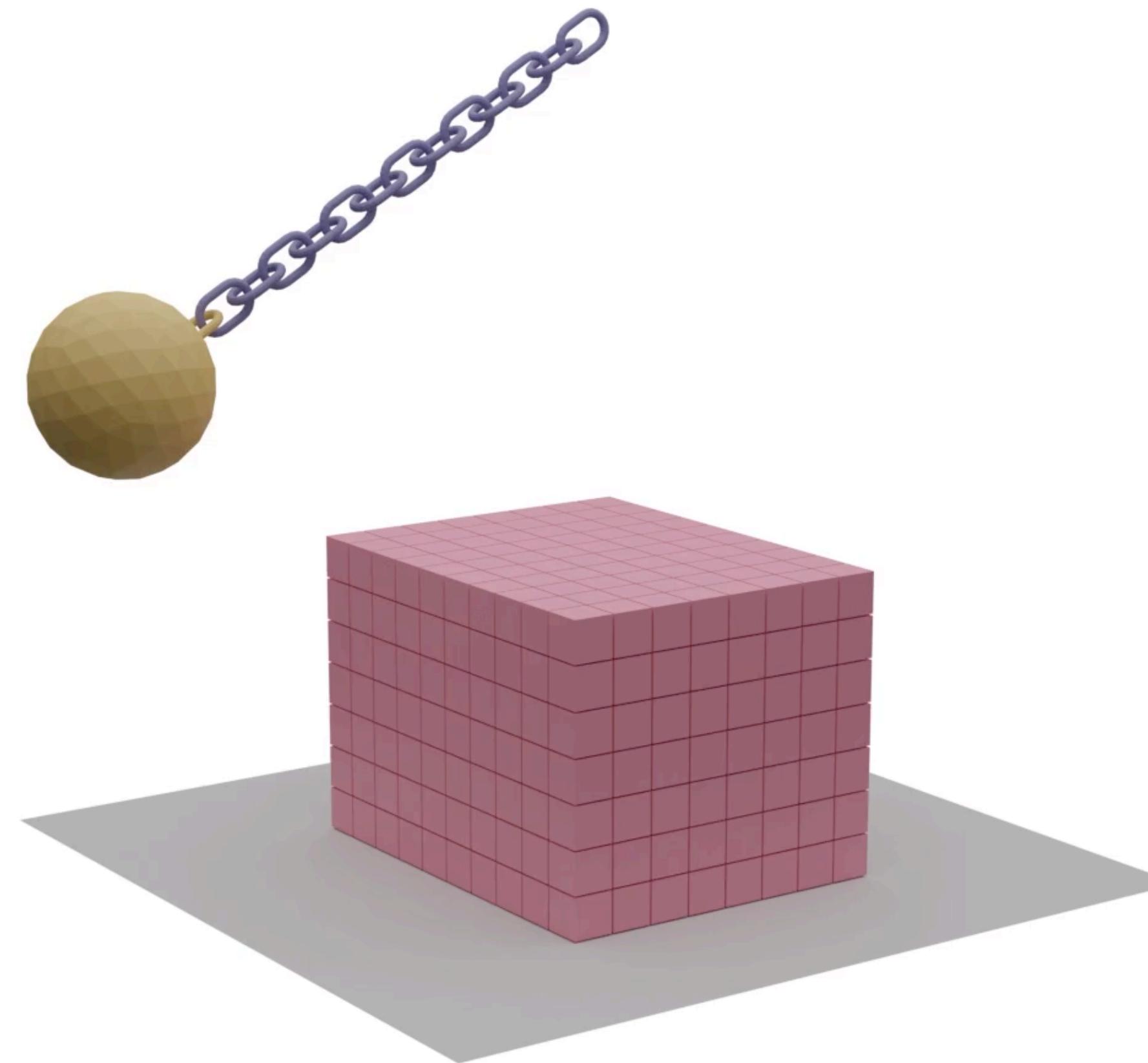
So CCD is on nonlinear trajectories:



Very expensive!

# Simulating Super Stiff Materials

Rigid-IPC [Ferguson et al. 2021] vs IPC [Li et al. 2020]



Example	runtime (s) (IPC)	runtime (s) (Rigid)	speed-up	iterations (IPC)	iterations (Rigid)
Pendulum	339.7	133.1	2.6x	10K	3K
Double pendulum	914.0	1559.9	0.6x	12K	4K
Arch (25 stones)	26.5	55.8	0.5x	2K	2K
Arch (101 stones)	238.3	487.8	0.5x	4K	5K
Wrecking ball	7179.8	5748.1	1.2x	9K	18K

**Rigid-IPC performs well for complex geometries**

# Simulating Super Stiff Materials

## Enforcing Rigidity via Penalty Method

Reduced order dynamics (from subspace optimization):

$$\min_{Q,q} \frac{1}{2} \|\bar{X}Q + \bar{S}q - \tilde{x}^n\|_M^2 + h^2 \sum P(\bar{X}Q + \bar{S}q) \quad \text{s.t.} \quad Q^T Q = I \quad \forall Q \quad (\text{or } f(Q) = 0)$$

**Don't need elasticity**

Reduced order dynamics with penalty method:

$$\min_{Q,q} \frac{1}{2} \|\bar{X}Q + \bar{S}q - \tilde{x}^n\|_M^2 + h^2 \sum P(\bar{X}Q + \bar{S}q)$$

**Use elasticity with large Young's modulus**

– the strain energy  $\Psi$  is effectively a penalty function for

12 DOF per body, still significantly reduced

$x = \bar{X}Q + \bar{S}q$  is linear w.r.t. both  $Q$  and  $q$   $\rightarrow$  linear CCD

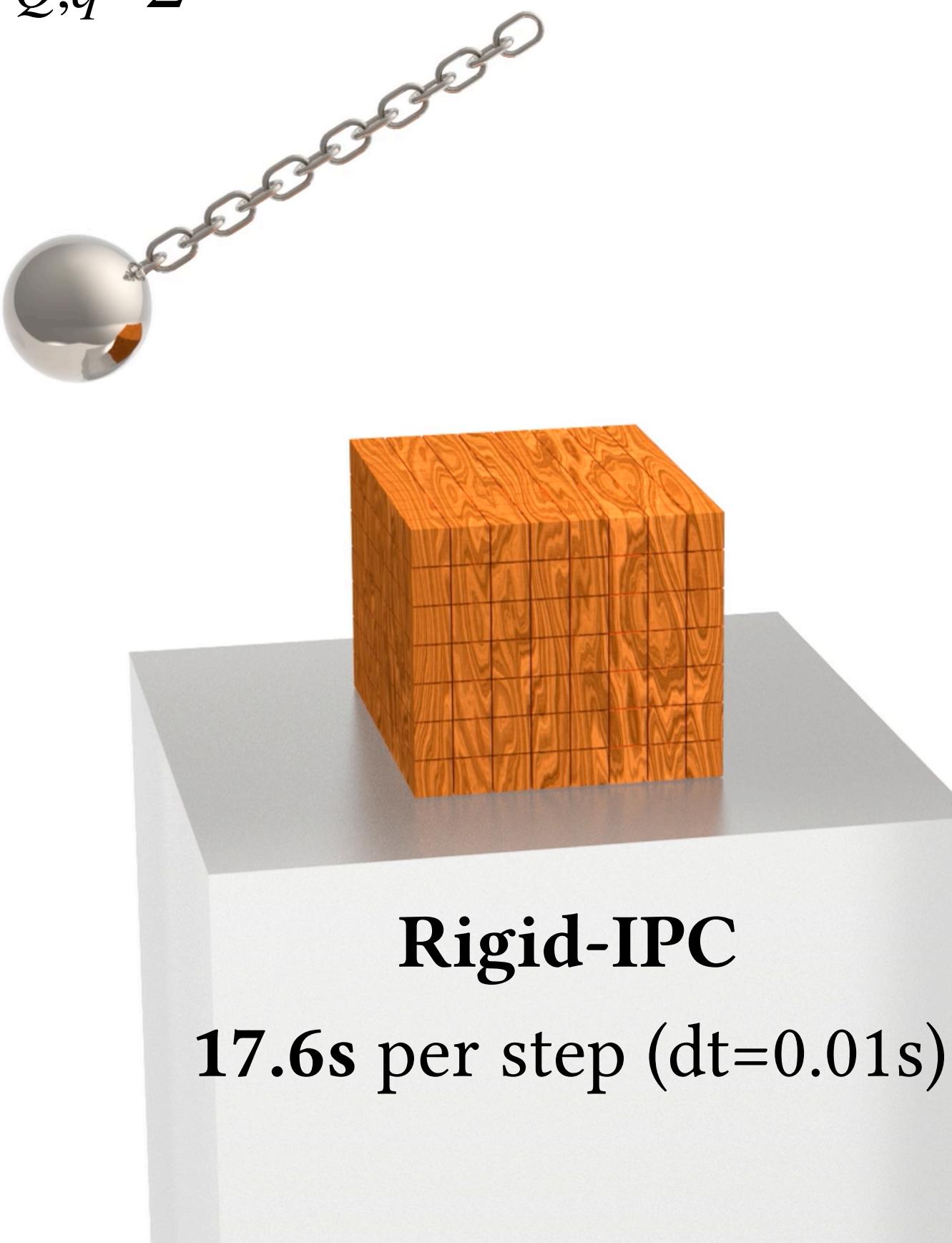
A stiff  $\Psi$  won't make the problem harder with stiff IPC energies

# Simulating Super Stiff Materials

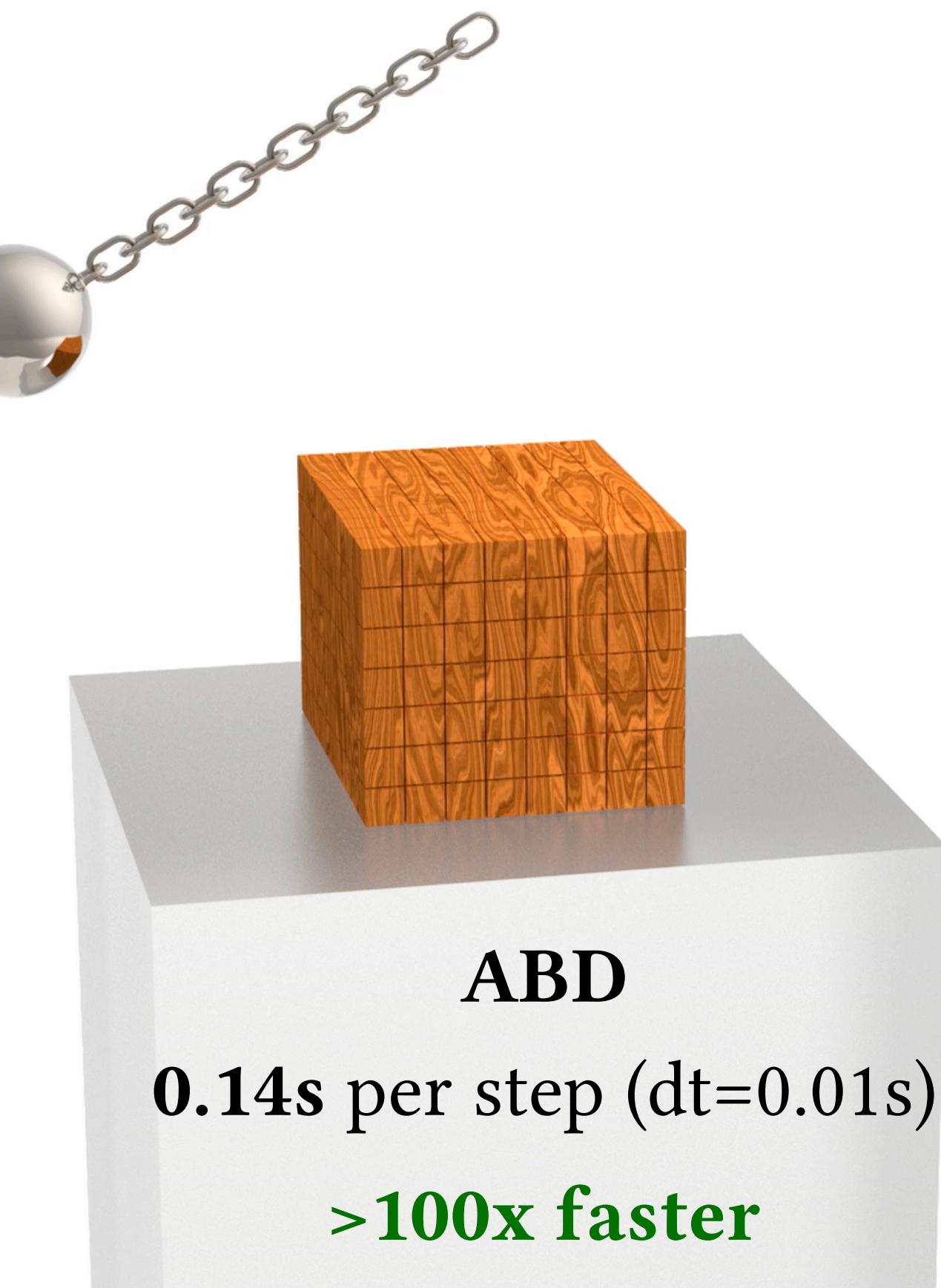
## Affine Body Dynamics (ABD)

$$\min_{Q,q} \frac{1}{2} \|\bar{X}Q + \bar{S}q - \tilde{x}^n\|_M^2 + h^2 \sum P(\bar{X}Q + \bar{S}q)$$

Use elasticity with large Young's modulus



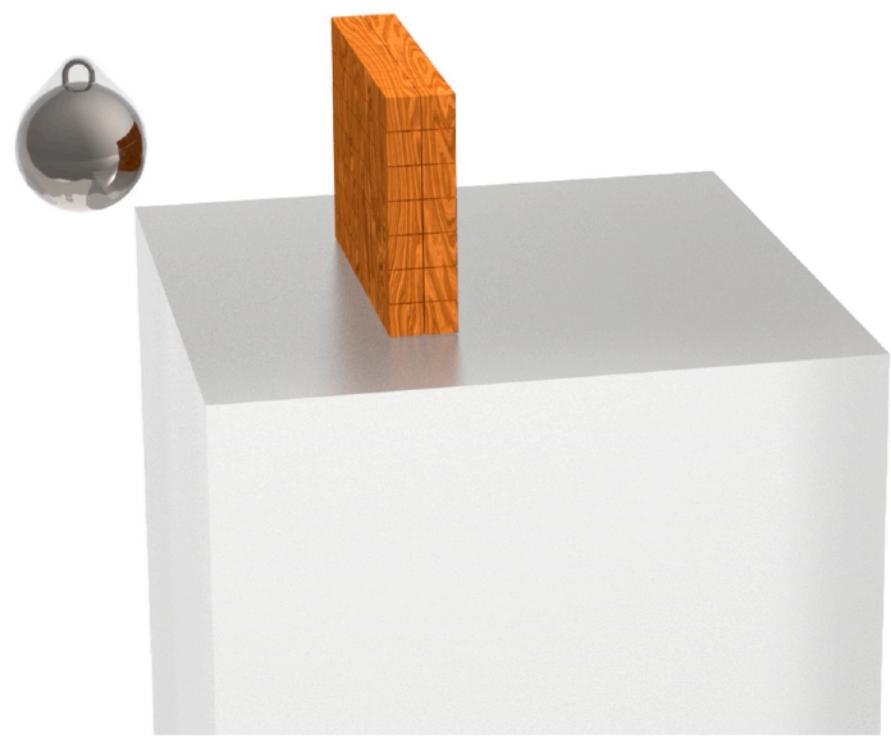
14K triangles  
575 bodies



# Simulating Super Stiff Materials

## Bullet v.s. ABD

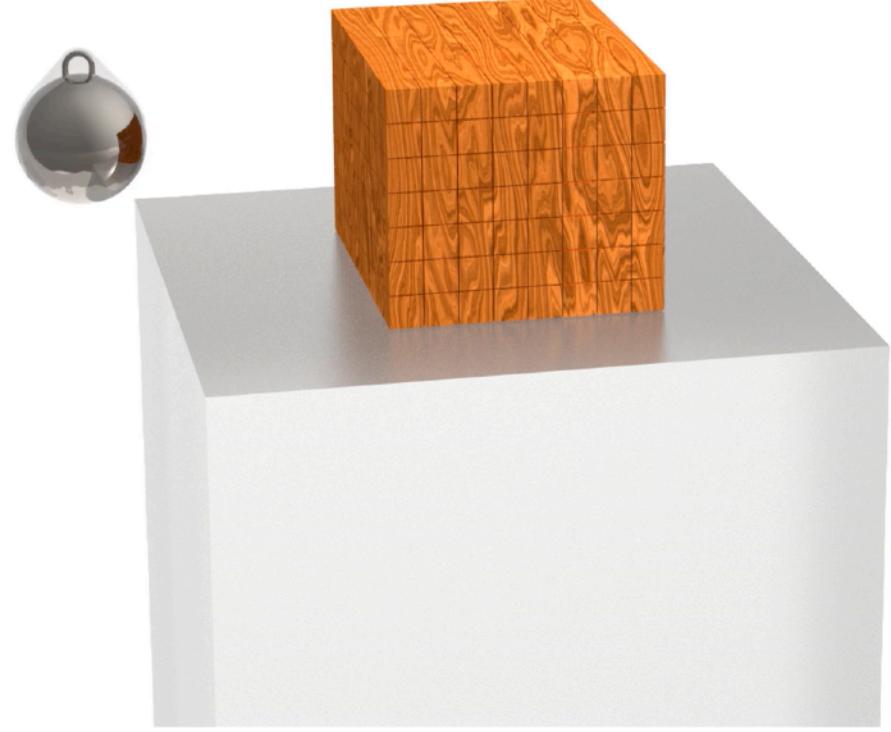
3.5K triangles  
142 bodies



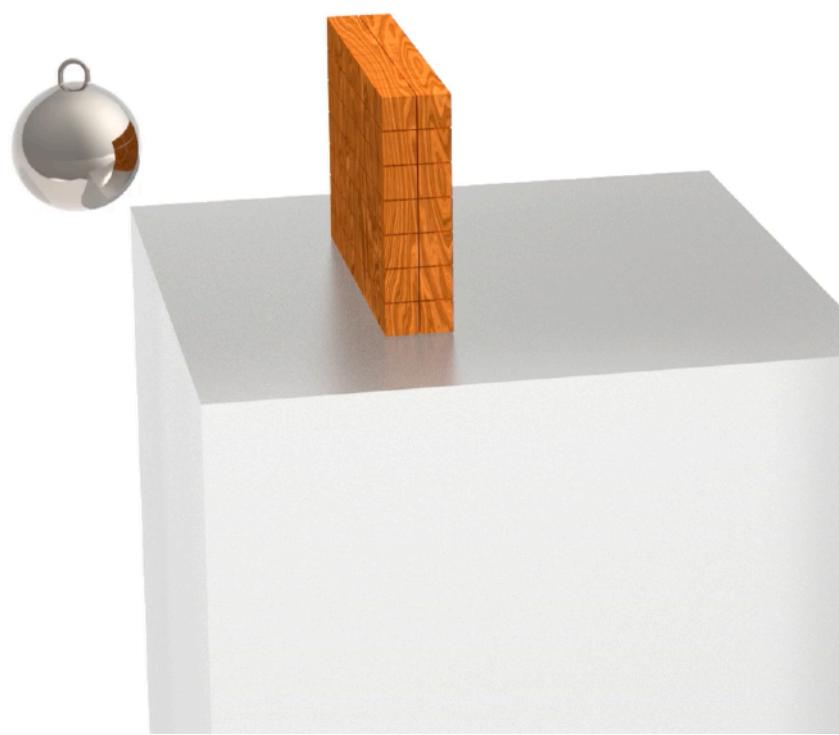
### Bullet

58ms per 1/240s step  
82ms per 1ms step

11K triangles  
562 bodies

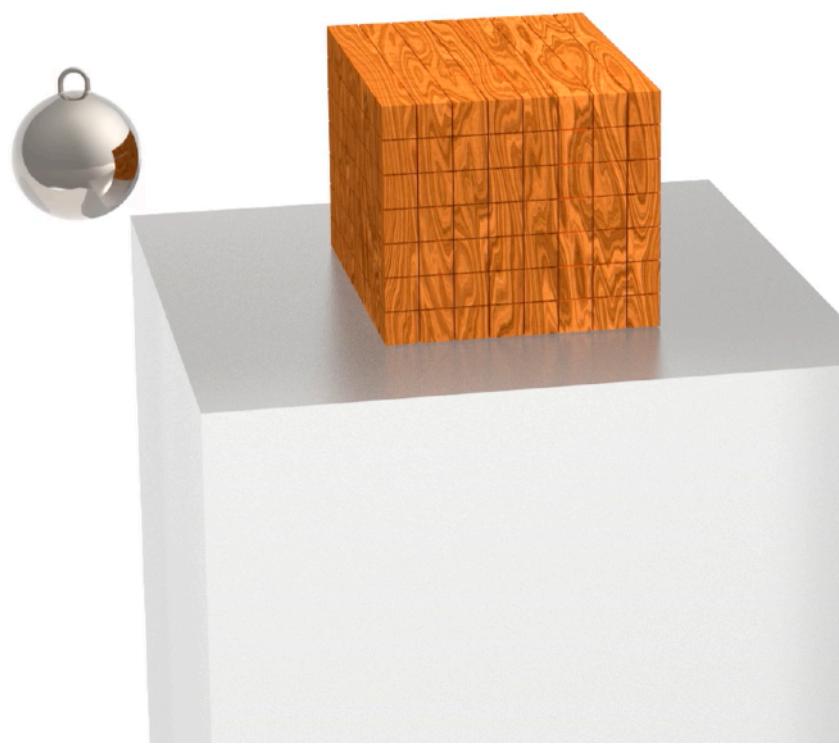


809ms per 1/240s step  
804ms per 1ms step



### ABD

41ms per 1/240s step  
19ms per 1ms step  
**>4x faster**



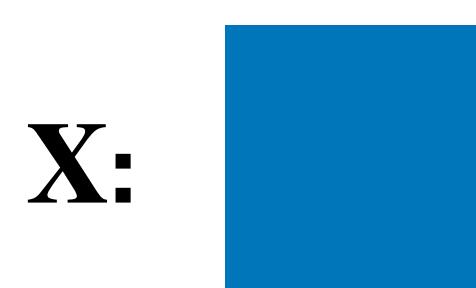
328ms per 1/240s step  
102ms per 1ms step  
**>8x faster**

# ABD in Another Perspective

## Affine Deformation Modes

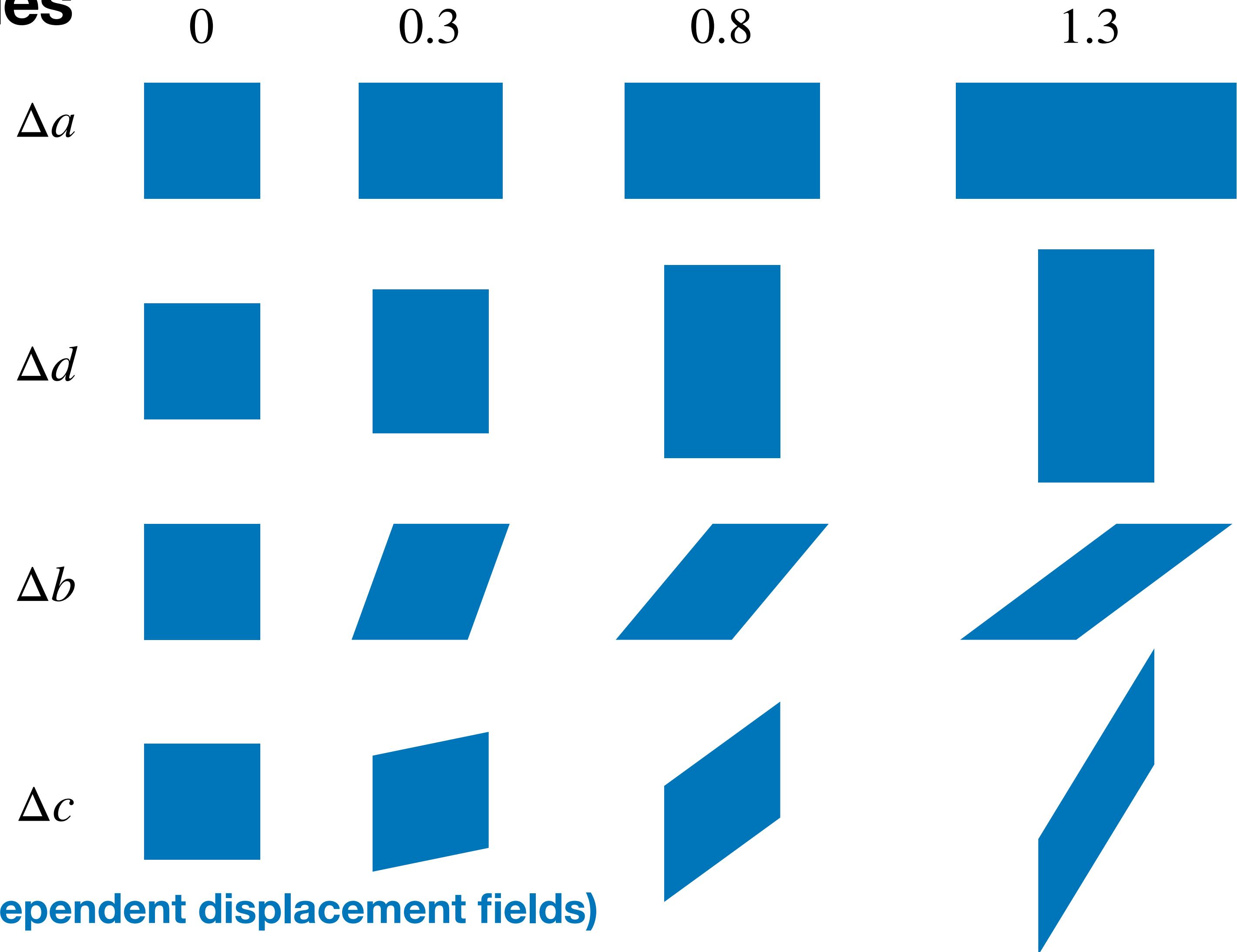
$$\mathbf{x} = \begin{bmatrix} a & b \\ c & d \end{bmatrix} \mathbf{X} + \begin{bmatrix} e \\ f \end{bmatrix}$$

DOF:  $a, b, c, d, e, f$



$$\mathbf{x} = A \begin{bmatrix} a \\ b \\ c \\ d \\ e \\ f \end{bmatrix} = aA_1 + bA_2 + \dots$$

**Deformation modes** (linearly independent displacement fields)



# Today:

- Simulating Stiff Elastic Solids  
*SE(3), Affine Body, Deformation Modes*
- **Modal-Order Reduction**
- Implementation & Demo

# Reduced Simulation of Deformable Solids

## Linear Modal Analysis

$$\mathbf{x} = A \begin{bmatrix} a \\ b \\ c \\ d \\ e \\ f \end{bmatrix} = aA_1 + bA_2 + \dots$$

Deformation modes (linearly independent displacement fields)

How do we generate more meaningful deformation modes?

Assume linear elasticity problem:  $M\ddot{u} + Ku = f$  s.t.  $Sx = 0$  (Dirichlet BC)

Intuition: Meaningful deformation modes are those don't generate large forces

Can solve the generalized Eigenvalue problem to find them:  $\bar{K}y = \lambda \bar{M}y$

(where  $\bar{K}$  and  $\bar{M}$  do not account for BC nodes)

(Take the Eigenvectors with smallest Eigenvalues as modes.)

# Reduced Simulation of Deformable Solids

## Linear Modal Analysis: Time Integration

$$\mathbf{x} = A \begin{bmatrix} a \\ b \\ c \\ d \\ e \\ f \end{bmatrix} = aA_1 + bA_2 + \dots$$

Deformation modes (linearly independent displacement fields)

Can solve  $\bar{K}y = \lambda \bar{M}y$  and take Eigenvectors with the smallest Eigenvalues as more modes.

The Eigenvectors will be orthonormal w.r.t.  $\bar{M}$ , i.e.  $(y^i)^T \bar{M} y^j = \delta_{ij}$ .

Now let  $u = x - X = Uz$ , where  $z \in \mathbb{R}^k$  are the reduced DOF,  $U \in \mathbb{R}^{3n \times k}$  formed by the Eigenvectors

Plugging in  $M\ddot{u} + Ku = f$ , ignoring BCs for now:

$$MU\ddot{z} + KUz = f$$

$$MU\ddot{z} + MU\Lambda z = f$$

$\Lambda \in \mathbb{R}^{k \times k}$  is a diagonal matrix of Eigenvalues

$$U^T MU\ddot{z} + U^T MU\Lambda z = U^T f$$

Left-multiply  $U^T$  on both sides

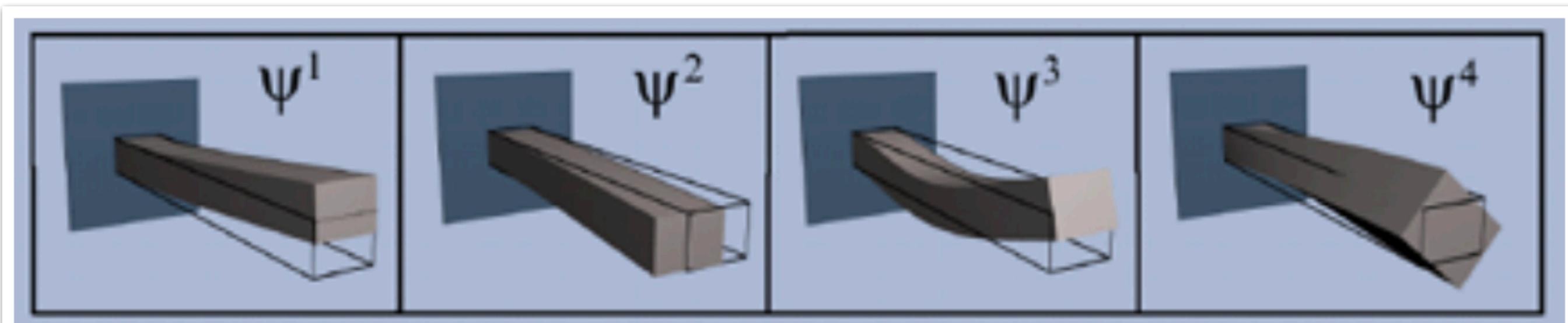
$$\ddot{z} + \Lambda z = U^T f$$

Diagonal system! Super fast!

# Reduced Simulation of Deformable Solids

## Linear Modal Analysis: Effectiveness

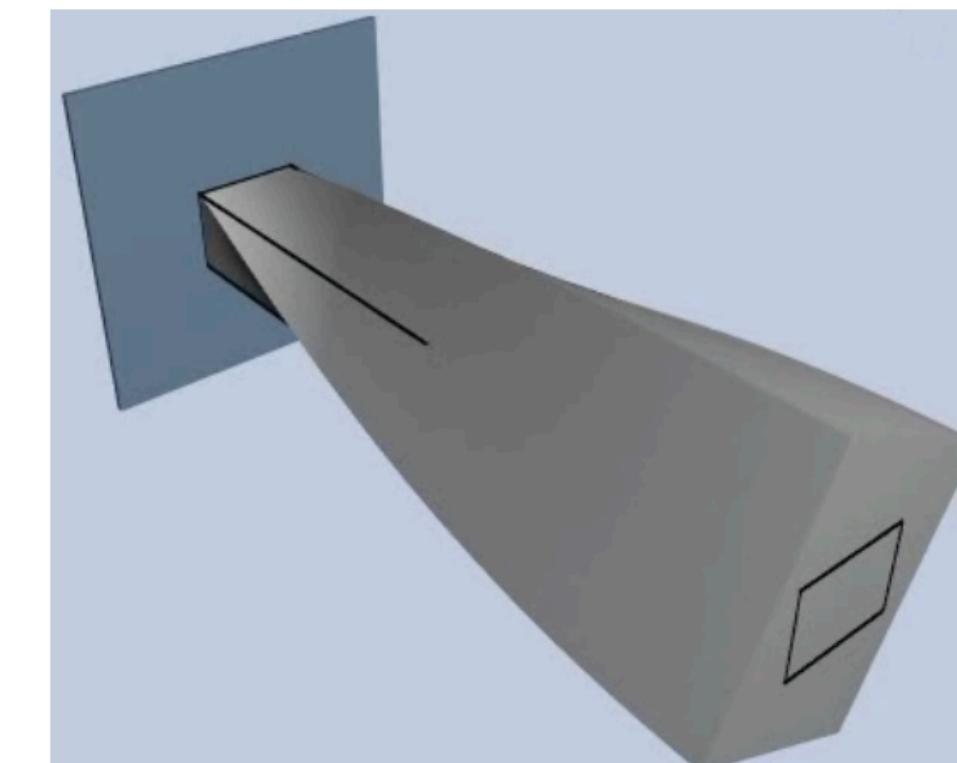
Works well for small deformations:



However:

Figure 2: Linear modes for a cantilever beam.

— Consistent with our knowledge  
of linear elasticity



linear

nonlinear

Figure 3: Model reduction applied to a linear and nonlinear system.

# Reduced Simulation of Deformable Solids

## Nonlinear Elasticity, Linear Modes

$M\ddot{u} + f^{int}(u) = f$  or equivalently, using Incremental Potential:  $\min_x \frac{1}{2} \|x - \tilde{x}^n\|_M^2 + h^2 \sum P(x)$

Plugging in  $u = U_z$ :  $\min_z \frac{1}{2} \|X + U_z - \tilde{x}^n\|_M^2 + h^2 \sum P(X + U_z)$  (Can compute  $U$  using  $\nabla^2 P(X)$ )

**Gradient:**  $U^T M(X + U_z - \tilde{x}^n) + h^2 \sum U^T \nabla P(X + U_z)$

**Hessian:**  $U^T M U + h^2 \sum U^T \nabla^2 P(X + U_z) U$

**Issue 1: Hessian can be dense!**

**Solution:** use locally supported modes, e.g. Cage-based deformation, Medial Axis Mesh [Lan et al. 2021], etc.

**Issue 2: Calculating  $\nabla P$  and  $\nabla^2 P$  are still slow (requiring full space computations)**

**Solution:** use numerical integration to approximate Gradient and Hessian, minimizing the number of quadratures [An et al. 2008]

# Reduced Simulation of Deformable Solids

## Nonlinear Elasticity, Linear Modes

$$\min_z \frac{1}{2} \|X + Uz - \tilde{x}^n\|_M^2 + h^2 \sum P(X + Uz)$$

**Issue 3: modes computed at rest shape (using  $\nabla^2 P(X)$ ) can result in artificial stiffening at large deformation**

**Solution 1: use simulated poses/deformed configurations as data, and perform PCA to construct  $U$**

**Solution 2: use nonlinear modes  $u = f(z)$  where  $f$  is a nonlinear function**

**e.g. in rigid body dynamics,  $u = f(\theta)$  is nonlinear**

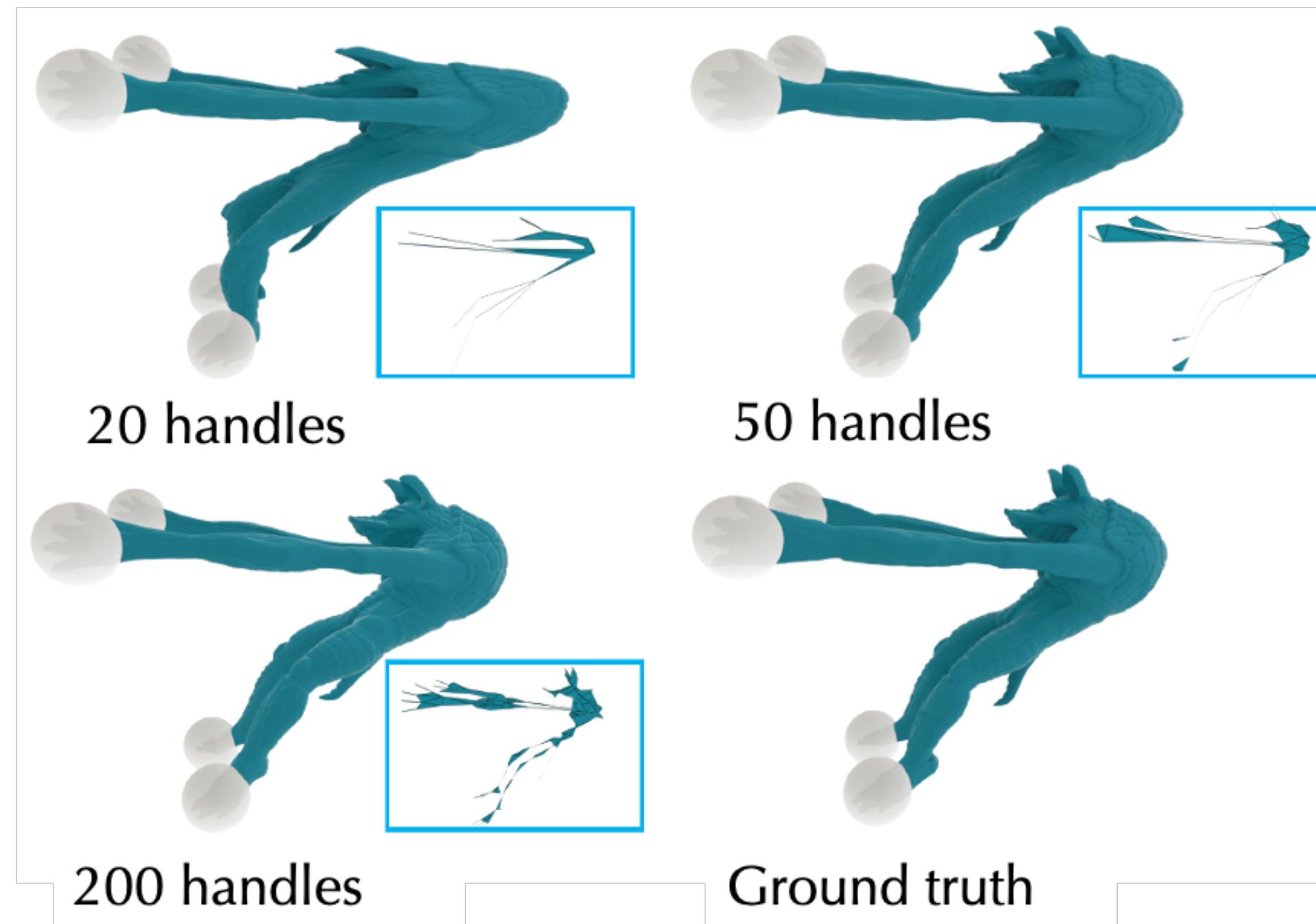
**Use modal derivatives to construct a quadratic function  $u = f(z)$  [\*]**

**Use neural networks to learn  $u = f(z)$**

**Remarks: Affine modes are linear modes, and are spatially linear; PCA and Eigen modes are linear modes, but can be spatially nonlinear.**

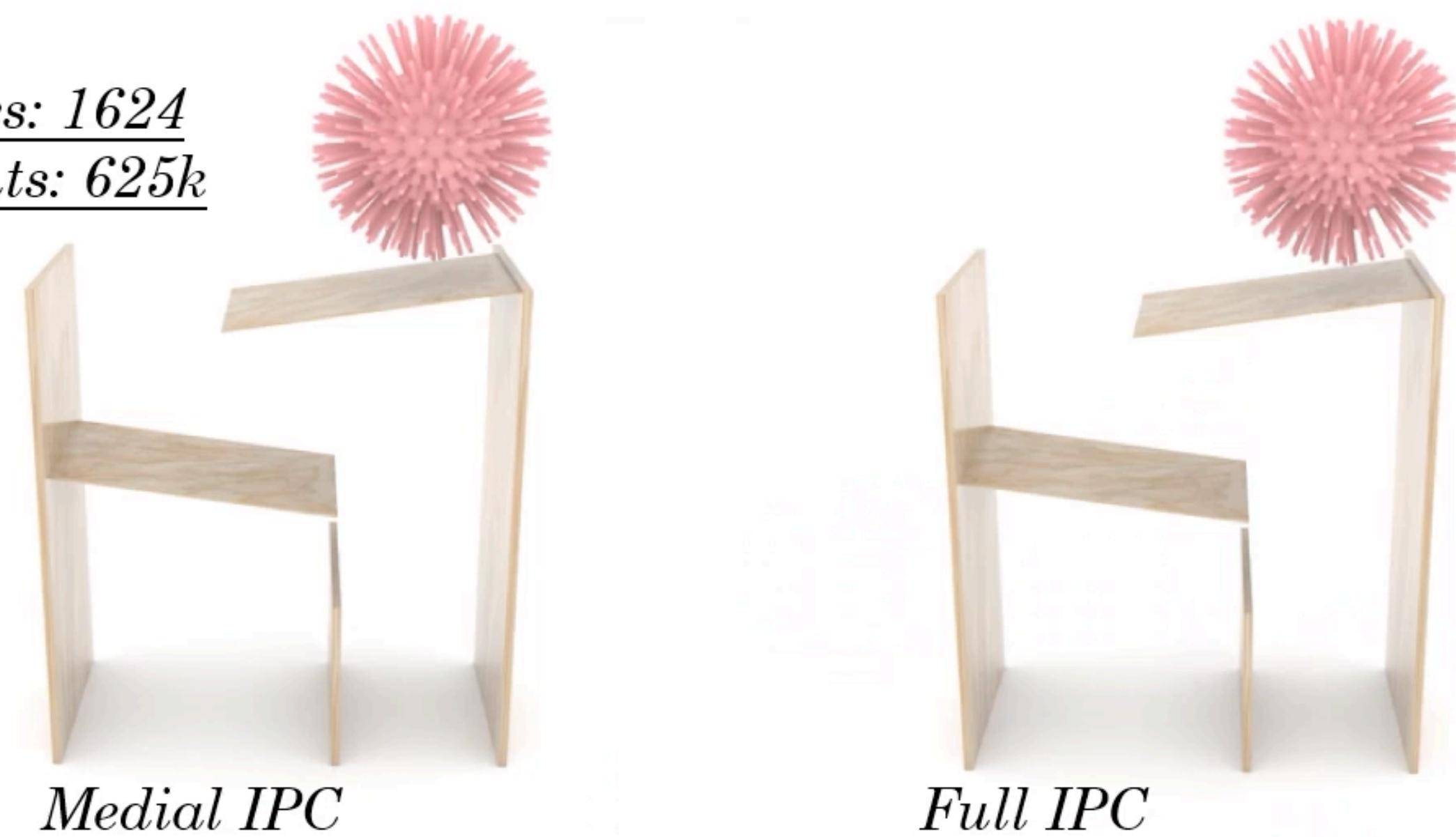
# Reduced Simulation of Deformable Solids

## Results from Medial IPC [Lan et al. 2021]



**Puffer Ball x 1**  
*36× speedup*

# of Handles: 1624  
# of Elements: 625k



# Today:

- **Simulating Stiff Elastic Solids**  
*SE(3), Affine Body, Deformation Modes*
- **Modal-Order Reduction**  
*Linear Modal Analysis for Linear and Nonlinear Elasticity*
- **Implementation & Demo**

# Implementation

## Compute Basis (Polynomial)

```

if order == 1: # linear basis, or affine basis
    basis = np.zeros((len(x) * 2, 6)) # 1, x, y for both x- and y-displacements
    for i in range(len(x)):
        for d in range(2):
            basis[i * 2 + d][d * 3] = 1
            basis[i * 2 + d][d * 3 + 1] = x[i][0]
            basis[i * 2 + d][d * 3 + 2] = x[i][1]

elif order == 2: # quadratic polynomial basis
    basis = np.zeros((len(x) * 2, 12)) # 1, x, y, x^2, xy, y^2 for both x- and y-displacements
    for i in range(len(x)):
        for d in range(2):
            basis[i * 2 + d][d * 6] = 1
            basis[i * 2 + d][d * 6 + 1] = x[i][0]
            basis[i * 2 + d][d * 6 + 2] = x[i][1]
            basis[i * 2 + d][d * 6 + 3] = x[i][0] * x[i][0]
            basis[i * 2 + d][d * 6 + 4] = x[i][0] * x[i][1]
            basis[i * 2 + d][d * 6 + 5] = x[i][1] * x[i][1]

elif order == 3: # cubic polynomial basis
    basis = np.zeros((len(x) * 2, 20)) # 1, x, y, x^2, xy, y^2, x^3, x^2y, xy^2, y^3 for both x- and y-displacements
    for i in range(len(x)):
        for d in range(2):
            basis[i * 2 + d][d * 10] = 1
            basis[i * 2 + d][d * 10 + 1] = x[i][0]
            basis[i * 2 + d][d * 10 + 2] = x[i][1]
            basis[i * 2 + d][d * 10 + 3] = x[i][0] * x[i][0]
            basis[i * 2 + d][d * 10 + 4] = x[i][0] * x[i][1]
            basis[i * 2 + d][d * 10 + 5] = x[i][1] * x[i][1]
            basis[i * 2 + d][d * 10 + 6] = x[i][0] * x[i][0] * x[i][0]
            basis[i * 2 + d][d * 10 + 7] = x[i][0] * x[i][0] * x[i][1]
            basis[i * 2 + d][d * 10 + 8] = x[i][0] * x[i][1] * x[i][1]
            basis[i * 2 + d][d * 10 + 9] = x[i][1] * x[i][1] * x[i][1]

```

utils.py

$$\mathbf{x} = \begin{bmatrix} a & b \\ c & d \end{bmatrix} \mathbf{X} + \begin{bmatrix} e \\ f \end{bmatrix}$$

DOF:  $a, b, c, d, e, f$

$\mathbf{X}:$  



$$\mathbf{x} = A \begin{bmatrix} a \\ b \\ c \\ d \\ e \\ f \end{bmatrix} = aA_1 + bA_2 + \dots$$

**Deformation**

**modes (linearly independent displacement fields)**

For each 2 rows of  $A$ , find the corresponding node and its material space coordinates  $(x, y)$ , then use the following function to calculate  $A$ 's entry in each column:

$$col_a(x, y) = (x, 0)$$

$$col_d(x, y) = (0, y)$$

$$col_b(x, y) = (y, 0)$$

$$col_e(x, y) = (1, 0)$$

$$col_c(x, y) = (0, x)$$

$$col_f(x, y) = (0, 1)$$

# Implementation

## Compute Basis (Modal)

- For simplicity, we directly use the Eigenvectors of  $\nabla^2\Psi(X)$  (no PSD projection) with the smallest Eigenvalues as basis.

```
if order <= 0 or order >= len(x) * 2:
    print("invalid number of target basis for modal reduction")
    exit()
IJV = NeoHookeanEnergy.hess(x, e, vol, IB, mu_lame, lam, project_PSD=False)
H = sparse.coo_matrix((IJV[2], (IJV[0], IJV[1])), shape=(len(x) * 2, len(x) * 2)).tocsr()
eigenvalues, eigenvectors = eigsh(H, k=order, which='SM') # get 'order' eigenvectors with smallest eigenvalues
return eigenvectors
```

utils.py

# Implementation

## Simulation in the Subspace

- Pick and compute the basis:

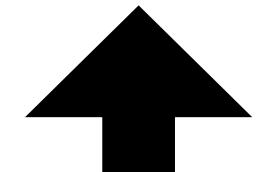
simulator.py

```
# compute reduced basis using 0: no reduction; 1: polynomial functions; 2: modal reduction
reduced_basis = utils.compute_reduced_basis(x, e, vol, IB, mu_lame, lam, method=1, order=2)
```

- Solve the time stepping optimization in the subspace:

```
def search_dir(x, e, x_tilde, m, vol, IB, mu_lame, lam, y_ground, contact_area, is_DBC, reduced_basis, h):
    projected_hess = IP_hess(x, e, x_tilde, m, vol, IB, mu_lame, lam, y_ground, contact_area, h)
    reshaped_grad = IP_grad(x, e, x_tilde, m, vol, IB, mu_lame, lam, y_ground, contact_area, h).reshape(len(x) * 2, 1)
    # eliminate DOF by modifying gradient and Hessian for DBC:
    for i, j in zip(*projected_hess.nonzero()):
        if is_DBC[int(i / 2)] | is_DBC[int(j / 2)]:
            projected_hess[i, j] = (i == j)
    for i in range(0, len(x)):
        if is_DBC[i]:
            reshaped_grad[i * 2] = reshaped_grad[i * 2 + 1] = 0.0
    reduced_hess = reduced_basis.T.dot(projected_hess.dot(reduced_basis)) # applying chain rule
    reduced_grad = reduced_basis.T.dot(reshaped_grad) # applying chain rule
    return (reduced_basis.dot(spsolve(reduced_hess, -reduced_grad))).reshape(len(x), 2) # transform to full space after the solve
```

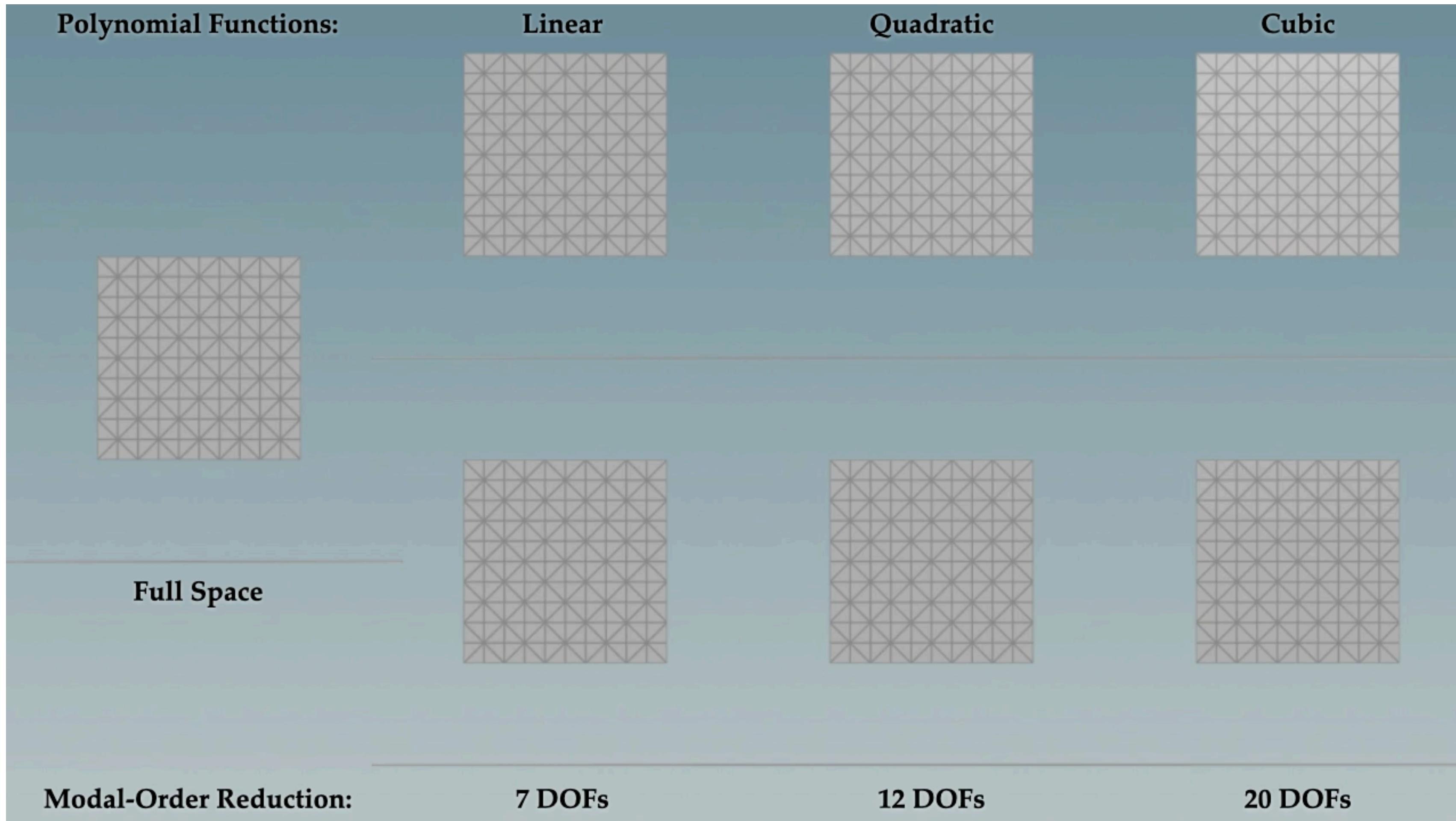
Instead of  $p = H^{-1}(-g)$ ,  
Compute  $p = A(A^T H A)^{-1}(-A^T g)$



time\_integrator.py

# Demo!

Code: [github.com/phys-sim-book/solid-sim-tutorial](https://github.com/phys-sim-book/solid-sim-tutorial)



# Today:

- **Simulating Stiff Elastic Solids**  
*SE(3), Affine Body, Deformation Modes*
- **Modal-Order Reduction**  
*Linear Modal Analysis for Linear and Nonlinear Elasticity*
- **Implementation & Demo**  
*Compute Basis, Simulation in the Subspace*

# Next Lecture: Codimensional Solids



# Image Sources

- <https://padeepz.net/ce6602-syllabus-structural-analysis-2-regulation-2013-anna-university/>
- [https://en.wikipedia.org/wiki/Young%27s modulus](https://en.wikipedia.org/wiki/Young%27s_modulus)
- <http://viterbi-web.usc.edu/~jbarbic/femdefo/barbic-courseNotes-modelReduction.pdf>